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# NASA Contractor Report 3552



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## An Electronic Control for an Electrohydraulic Active Control Landing Gear for the F-4 Aircraft

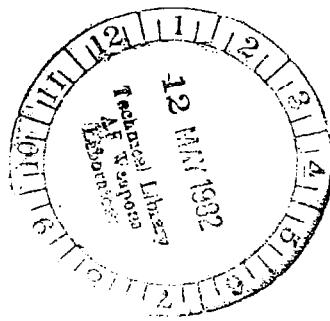
Irving Ross and Ralph Edson

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## NASA Contractor Report 3552

# An Electronic Control for an Electrohydraulic Active Control Landing Gear for the F-4 Aircraft

**Irving Ross and Ralph Edson**  
*Hydraulic Research Textron, Inc.*  
*Valencia, California*

Prepared for  
Langley Research Center  
under Contract NAS1-16420



National Aeronautics  
and Space Administration

**Scientific and Technical  
Information Branch**

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## 1.0

### SUMMARY

The electronic controller described in this report is a modification of the controller which was designed under NASA Contract NAS1-14459 and fully documented in Reference 1.

As in the original design, the controller continuously compares the kinetic energy of the aircraft with the work potential of the gear until the work potential exceeds the kinetic energy. The wing/gear interface force present at this condition becomes the command force to a servo loop which maintains the wing/gear interface force at this level by providing a signal to an electrohydraulic servovalve to port flow into or out of the landing gear.

Analytical results indicate that the controller provides significant reductions in forces sustained by the aircraft during vertical drops and radical reductions in forces during rollout over repaired bomb craters.

## 2.0

### INTRODUCTION

Hydraulic Research Textron (HRT) was retained under NASA Contract NAS1-16420 to design a controller for an active control landing gear (ACLG) to be used on the F-4 aircraft. The design was a modification of the controller originally designed for a 2948 kg. (6,500 lb) aircraft and described in detail in Reference 1.

The design was to be based on a digital computer simulation using a linear model of the aircraft and landing gear. However, it became apparent early in the program that the linear model was not adequate by itself to predict performance under certain phases of landing where nonlinear relationships prevail. Therefore extensive use was made of the HRT nonlinear model as well as the linear model to achieve the final design. The parameters of the aircraft/landing gear system were supplied by NASA.

The problem was similar to that encountered in the original design. However, the aircraft is heavier and the stroke of the strut is greater so that new scaling requirements are imposed. These are discussed in Appendix A.

## 3.0 DYNAMIC ANALYSIS OF F-4 ACTIVE CONTROL LANDING GEAR

### 3.1 PREFACE

This section presents the dynamic analyses that were performed for the development of an electrohydraulic active control system for the F-4 landing gear. The main objective of these analyses was to develop a loop compensation network for the active control landing gear concept applied to the F-4 aircraft and to evaluate the performance of the active control gear with respect to the passive (conventional) F-4 landing gear. Section 3.2 contains a list of symbols and section 3.3 describes the analytical tools used in these studies, which are the linear and nonlinear vertical drop dynamic simulation models of the landing gear, without aircraft equations of motion included. Section 3.4 presents the correlation between the linear and nonlinear simulations. Section 3.5 presents the development of the loop compensation network. Section 3.6 presents analytical results for specific landing impact cases and cases of rollout over "repaired bomb craters", using the nonlinear vertical drop model, for both the passive gear and the active control gear.

### 3.2 SYMBOLS

$A_o$  area of orifice in shock strut orifice plate, see Figure 3-1.

$A_p$  landing gear metering pin area, see Figure 3-2.

$A_1$  shock strut hydraulic area (piston area),  $0.01024 \text{ m}^2$   
( $15.87 \text{ in}^2$ )

$A_2$  shock strut pneumatic area (cylinder area),  $0.01494 \text{ m}^2$   
( $23.16 \text{ in}^2$ )

$A_3$  annular area in shock strut between piston and cylinder walls,  $0.00761 \text{ m}^2$  ( $1.179 \text{ in}^2$ )

ATIRE constant in tire deflection force equation, 1.20

$C_d$  discharge coefficient for active control servovalve orifice, 0.62

$C_{do}$  discharge coefficient for shock strut orifice, 0.60

$C_o$  orifice coefficient for shock strut orifice

$$= C_{do} A_o \sqrt{\frac{2g_c}{\rho}}, \text{ m}^4 \text{ sec}^{-1} \cdot N^{1/2} (\text{in}^3/\text{sec}/\text{psi}^{1/2})$$

CP Linearized orifice coefficient for active control servovalve

$$= - \frac{\partial Q_{SV}}{\partial P_1} 3.16 \times 10^{-11} m^5 \cdot N^{-1} \cdot sec^{-1} (0.01334 \text{ in}^3/\text{sec/psi})$$

CP<sub>O</sub> linearized orifice coefficient for shock strut orifice

$$= \frac{\partial Q_O}{\partial P_1} = C_O / (2\sqrt{P_1 - P_2}), 1.901 \times 10^{-9} m^5 \cdot N^{-1} \cdot sec^{-1} (0.8 \text{ in}^3/\text{sec/psi})$$

CQ Linearized orifice coefficient for active control servovalve:

$$= \frac{\partial Q_{SV}}{\partial X_{SV}} = C_{SV} \sqrt{(P_S + P_R)/2}, 8.61 m^2/\text{sec} (13,340 \text{ in}^3/\text{sec/in})$$

C<sub>SV</sub> orifice coefficient for active control servovalve:

$$= C_d W_{SV} \sqrt{2g_C/\rho}, 0.00268 m^3 \cdot sec^{-1} \cdot N^{-1/2} (344.4 \text{ in}^3/(in lbf^{1/2}))$$

f coulomb friction between shock strut piston and cylinder, 222.N (50.lbf)

F<sub>a</sub> vertical force exerted on shock strut by the runway surface, N (lbf)

F<sub>li</sub> impact phase limit force, N (lbf)

F<sub>lim</sub> limit force, N (lbf)

F<sub>s</sub> shock strut force, N (lbf)

F<sub>wg</sub> wing-gear interface force, N (lbf)

g acceleration due to gravity, 9.81m/sec<sup>2</sup>(386.in/sec<sup>2</sup>)

g<sub>c</sub> gravitational acceleration constant

$$1 \text{ kg} \cdot \text{m} \cdot \text{N}^{-1} \cdot \text{sec}^{-2} (12 \text{ slug} \cdot \text{in} \cdot \text{lbf}^{-1} \cdot \text{sec}^{-2})$$

i<sub>1</sub> input signal to electronic compensation networks, A

i<sub>2</sub> output signal from electronic compensation networks, or input signal to active control servovalve, ( $\pm 0.040$  A maximum)

K<sub>a</sub> amplifier gain in active control loop, 0.000020 A/V

$K_f$	position feedback gain in strut position control loop, 563 V/m (14.29V/in)
$K_{FDGE}$	fraction of total strut stroke assumed available when computing impact phase force, 1.0
$K_{SV}$	position gain of servovalve in active control loop, 0.0635 m/A (2.50 in/A)
$K_{TIRE}$	constant in tire deflection force equation 1727.1 kN/m (9862 lbf/in)
$K_x$	gain in strut position control loop, 1.0 m/m (1.0 in/in)
$L$	total lift force, N (lbf)
$M$	mass of airplane per gear, 8345 kg (18398 lbm)
$M_c$	mass of upper portion of landing gear (cylinder plus orifice plate attachment, kg (slugs)
$M_L$	mass of lower portion of landing gear (piston plus tire), 204. kg (13.99 slugs = 1.166 lbf . sec <sup>2</sup> /in = 450. lbm)
$M_U$	upper mass, 8143. kg (558. slugs = 46.5 lbf. sec <sup>2</sup> /in = 17948. lbm)
$PE_t$	potential energy stored in tire due to compression, N . m (ft . lbf)
$P_S$	hydraulic supply pressure, $2.07 \times 10^7$ N/m <sup>2</sup> (3000 . psi)
$P_R$	hydraulic return pressure, 0.0 N/m <sup>2</sup> (0.0 psi)
$P_1$	hydraulic pressure in shock strut piston, N/m <sup>2</sup> (psi)
$P_2$	pneumatic pressure in shock strut cylinder, N/m <sup>2</sup> (psi)
$P_3$	pressure in volume between walls of shock strut piston and cylinder, N/m <sup>2</sup> (psi)
$Q_o$	flow rate through shock strut orifice from piston to cylinder, m <sup>3</sup> /sec (in <sup>3</sup> /sec)

$Q_{sv}$	flow rate from active control servovalve to shock strut piston, linear model, $\text{m}^3/\text{sec}$ ( $\text{in}^3/\text{sec}$ )
$Q_{sv1}$	flow rate through active control servovalve from supply pressure to the shock strut piston, $\text{m}^3/\text{sec}$ ( $\text{in}^3/\text{sec}$ )
$Q_{sv2}$	flowrate through active control servovalve from shock strut piston to return pressure, $\text{m}^3/\text{sec}$ ( $\text{in}^3/\text{sec}$ )
$R_s$	the slope of the limit force with respect to time during transition phase, 444800. $\text{N/sec}$ (100000. $\text{lbf/sec}$ )
$s$	LaPlace operator, $\text{sec}^{-1}$
$t$	time, sec
$v$	velocity, $\text{m/sec}$ ( $\text{in/sec}$ )
$V_s$	sink rate, $\text{m/sec}$ ( $\text{in/sec}$ )
$V_1$	hydraulic volume in shock strut piston and lines up to the active control servovalve, $0.00497 \text{ m}^3$ ( $303.\text{in}^3$ )
$V_2$	pneumatic volume, $0.00742 \text{ m}^3$ ( $453.\text{ in}^3$ ) for fully extended strut
$V_3$	volume between shock strut piston and cylinder, $0.0 \text{ m}^3$ ( $0.0 \text{ in}^3$ ) for fully extended strut
$W_{sv}$	window width of orifices on third stage spool of active control servovalve, $0.0884 \text{ m}$ (3.48 in)
$X_a$	displacement of lower mass of shock strut or axle, $\text{m}$ (in)
$X_c$	commanded position of shock strut, $0.216 \text{ m}$ (8.50 in)
$X_g$	ground level displacement, $\text{m}$ (in)
$X_s$	shock strut stroke, $\text{m}(in)$ $X_s = 0$ fully extended, $X_s = 0.403 \text{ m}$ (15.88 in) fully compressed
$x_{wg}$	displacement of wing gear interface, $\text{m}$ (in)
$\beta$	bulk modulus of hydraulic fluid, $6.89 \times 10^8 \text{ N/m}^2$ ( $1 \times 10^5 \text{ psi}$ )
$\gamma$	ratio of specific heat of gas at constant pressure to that at constant volume, 1.06

$\rho$  mass density of hydraulic fluid,  $838 \text{ kg/m}^3$  ( $0.000941 \text{ slugs/in}^3 = 0.0303 \text{ lbm/in}^3$ )  
 $\tau_f$  time constant in strut position feedback loop, 0.10 sec  
 $\tau_1$  time constant in compensation, 0.001621 sec  
 $\tau_2$  time constant in compensation, 0.0001621 sec  
 $\tau_3$  time constant in compensation,  $6.464 \times 10^{-4}$  sec  
 $\tau_4$  time constant in compensation,  $6.464 \times 10^{-5}$  sec  
 $\omega_c$  corner frequency in active control servovalve transfer function,  $1263 \text{ sec}^{-1}$   
 $\omega_{sv}$  natural frequency in active control servovalve transfer function,  $655.5 \text{ sec}^{-1}$   
 $\omega_1$  natural frequency of notch network,  $565 \text{ sec}^{-1}$   
 $\zeta_{sv}$  damping coefficient in active control servovalve transfer function, 0.436  
 $\zeta_1$  damping coefficient in denominator of notch network, 5.1  
 $\zeta_2$  damping coefficient in numerator of notch network, 0.1

#### Subscripts:

$i$  initial conditions before impact  
 $im$  impact phase  
 $L$  lower mass  
 $max$  maximum value  
 $min$  minimum value  
 $r$  rollout phase  
 $s$  shock strut relative motion of lower mass (piston) with respect to the upper mass (cylinder)  
 $sv$  servovalve  
 $tr$  transition phase  
 $U$  upper mass

**Miscellaneous:**

$d(\ )$  indicates the differential of a variable

$\Delta(\ )$  indicates difference or change in a variable

$(.), (..), (...)$  dots indicate differentiation with respect to time

### 3.3 DYNAMIC SIMULATION MATH MODELS

The main analytical tools used in these studies are the linear (s-domain) and nonlinear (time domain) vertical drop dynamic simulation models of the landing gear. These models simulate motion in the vertical axis only. Aircraft equations of motion are not included, and aircraft mass (per gear) is simulated as a lumped mass resting on top of the landing gear.

#### 3.3.1 Linear Model

The linear model simulated the dynamics of the active control landing gear system in the frequency domain for small perturbations about the condition where the airplane mass (per gear) is resting on top of the gear with the gear always in contact with the ground and with the lower cylinder hydraulic pressure at a value halfway between the hydraulic supply and return pressures. The input disturbance variable is command limit force. Airplane lift and ground level are assumed constant. The linear model is a valuable tool since it allows rapid evaluation of system modifications or the effect of variation in system parameters in the areas of system stability and frequency response. A detailed description of the linear model, including equations, is presented in Reference 1, and will not be repeated here. The values of the constants used in the simulations for this study are given in Section 3.2 of this report.

#### 3.3.2. Nonlinear Model

The nonlinear model is developed from the time-dependent algebraic and differential equations of the system. The response of the system to input disturbances is obtained by integrating the differential equations with respect to time. Controller laws (including switching logic) and all other identifiable nonlinear attributes of the system of significance are simulated. Thus, the nonlinear model represents a more accurate simulation of the actual physical case than the linear model. This however, comes at the expense of considerably longer computational times. The nonlinear model accepts input variations in airplane lift, ground level, and command limit force (for vertical drop impact transients, however, the controller automatically sets the command limit force subsequent to initiation of active control). A detailed description of the nonlinear model, including equations, is presented in Reference 1. The nonlinear model used herein is identical to that described in Reference 1 except the values of the system constants are different and the linear spring tire force assumption was modified to a nonlinear spring, according to the relationship:

$$F_a = \begin{cases} KTIRE(XA - XG)^{ATIRE} & \text{for } XA > XG \\ 0 & \text{for } XA < XG \end{cases}$$

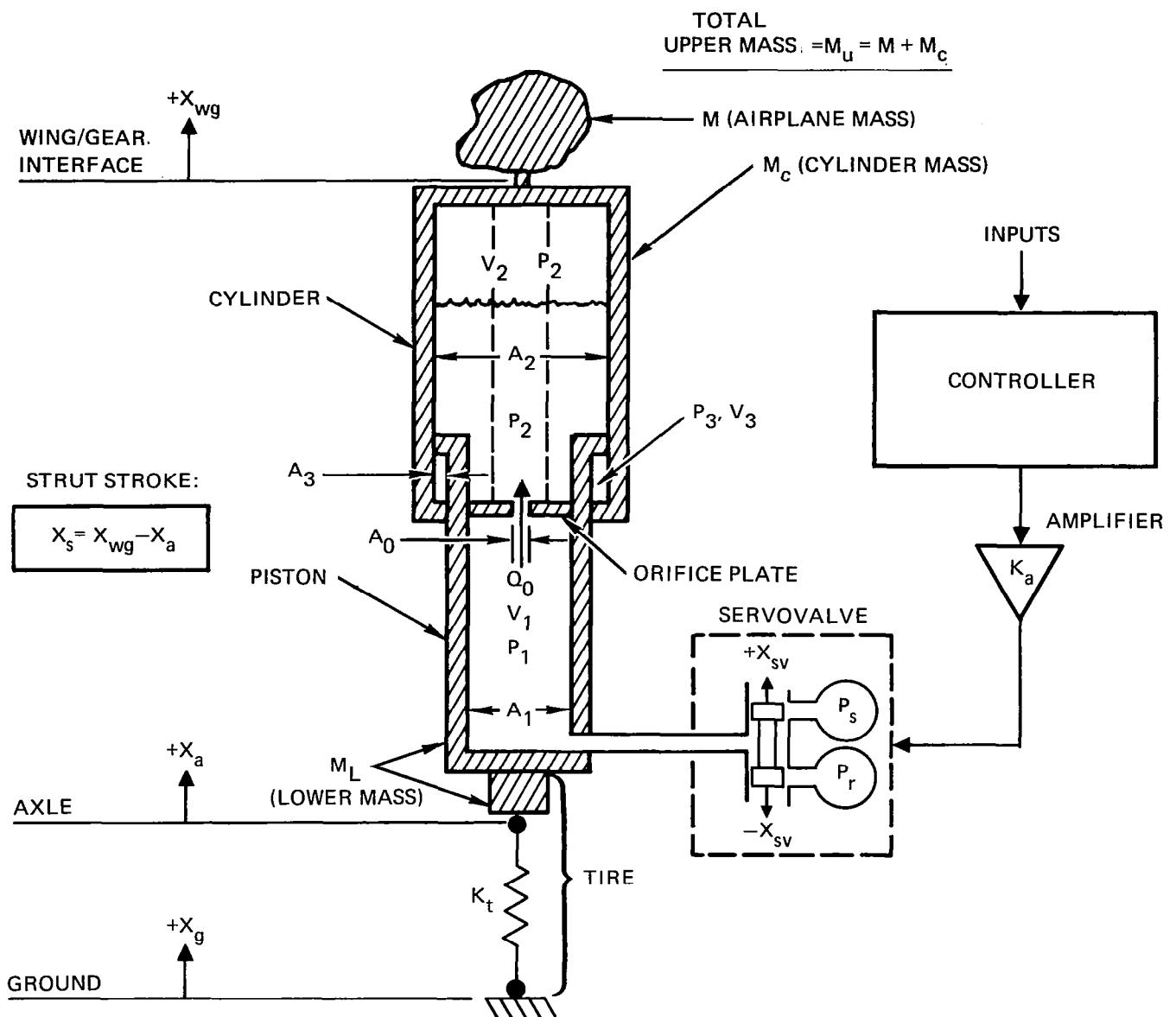
where KTIRE and ATIRE are constants. This equation replaces the expression for  $F_a$  in Equation 4 of Reference 1.

The values of the constants used in the linear and nonlinear simulations for this study are given in Section 3.2.

The important variables are shown in Figure 3-1.

### 3.4 CORRELATION OF LINEAR AND NONLINEAR MODELS

Since both the linear and nonlinear models were utilized in the development of the loop compensation, the first task was to correlate the linear model with the more precise nonlinear model to ensure that it would give at least reasonably credible results. Figures 3-3, 3-4, and 3-5 show frequency response results obtained from the linear and nonlinear models, without compensation. The loop is opened at the point of wing/gear force feedback, and the strut position feedback loop is not included. The input is command limit force and the output is the wing/gear force response. The nonlinear runs were made with zero lift and for command amplitudes of  $\pm 890$  N ( $\pm 200$  pounds), and the amplitude and phase angle at each frequency were computed from a Fourier analysis of the resultant input and output waveforms. The linear model results were obtained using a linearized orifice coefficient for the shock strut orifice ( $CP_o$ ) of  $0.8 \text{ in}^3/\text{sec/psi}$ . This value seemed to give the best overall correlation between the linear and nonlinear models. Note that the agreement is reasonably good out to a frequency of about 150 Hertz. At higher frequencies, the nonlinear model shows considerably more phase lead and less amplitude response than the linear model. Figures 3-6 and 3-7 show open loop Nyquist diagrams for these same results, for the linear and nonlinear models, respectively. Again, reasonably good correlation is indicated.



**FIGURE 3-1 ILLUSTRATION OF VARIABLES USED IN NONLINEAR SIMULATION OF SIMPLIFIED VERTICAL DROP CASE**

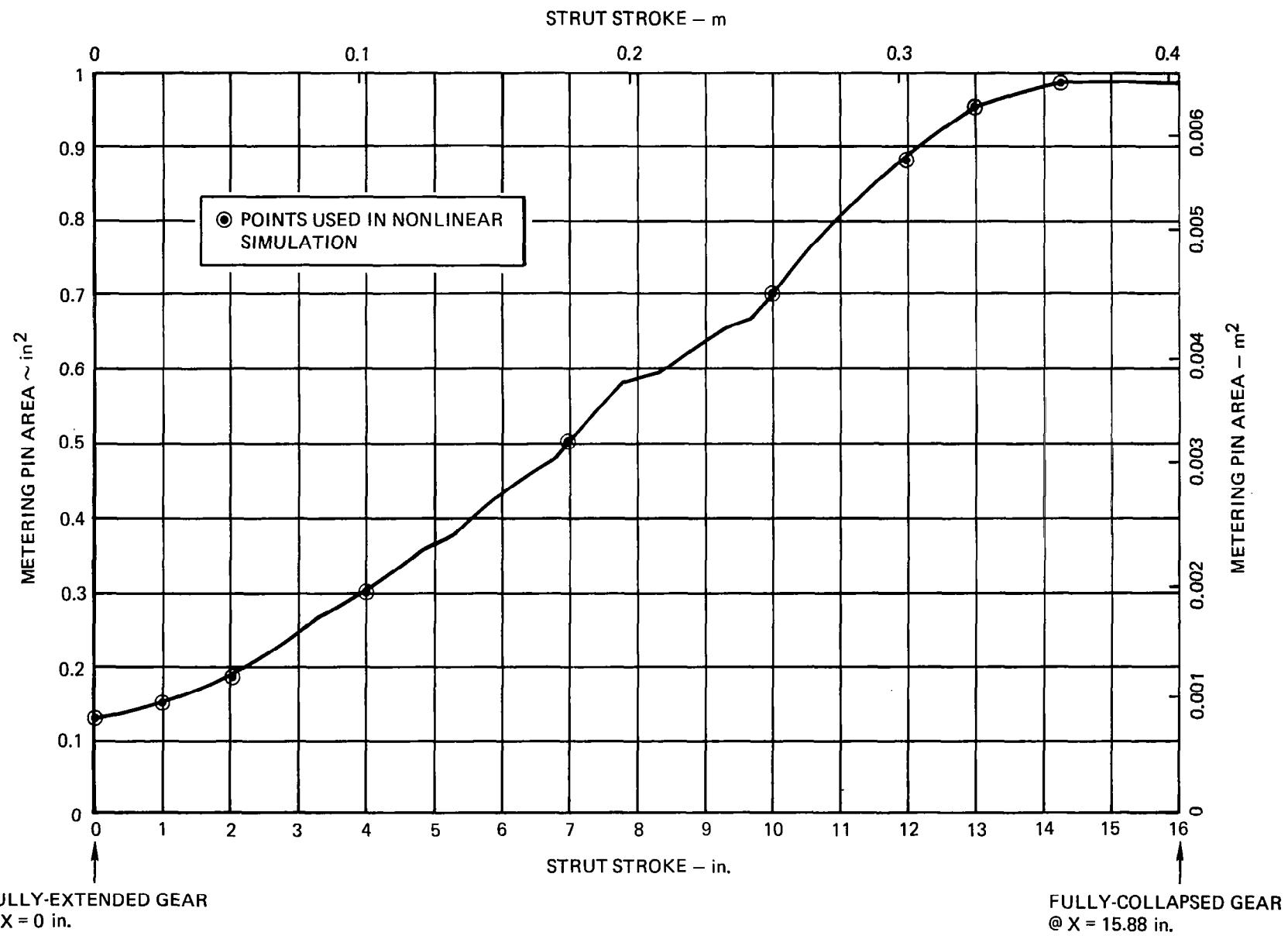


FIGURE 3-2. F-4 LANDING GEAR METERING PIN AREA VERSUS STRUT STROKE

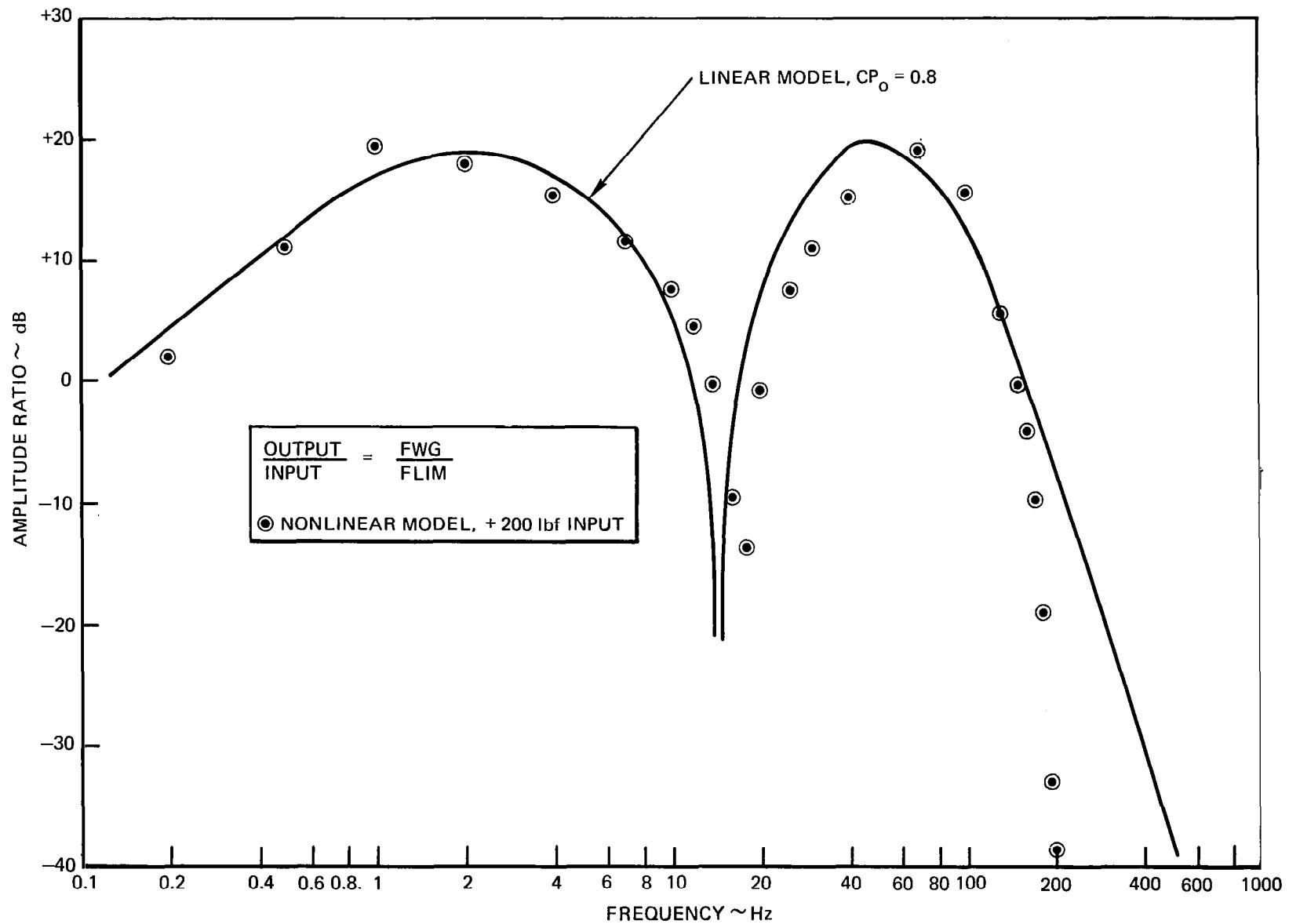


FIGURE 3-3 OPEN-LOOP, NO COMPENSATION  
FREQUENCY RESPONSE

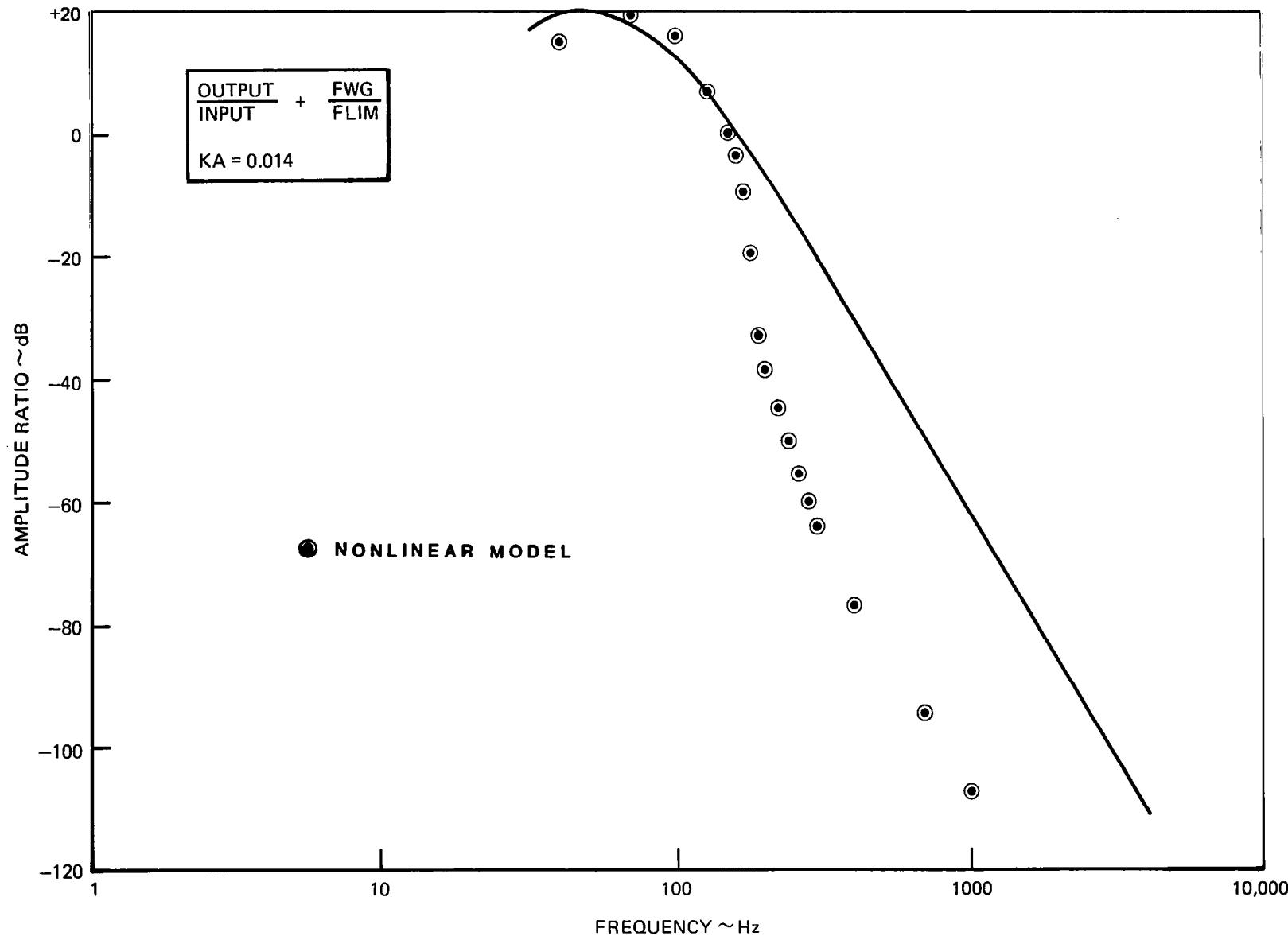


FIGURE 3-4. F-4 GEAR, OPEN-LOOP, NO COMPENSATION FREQUENCY RESPONSE

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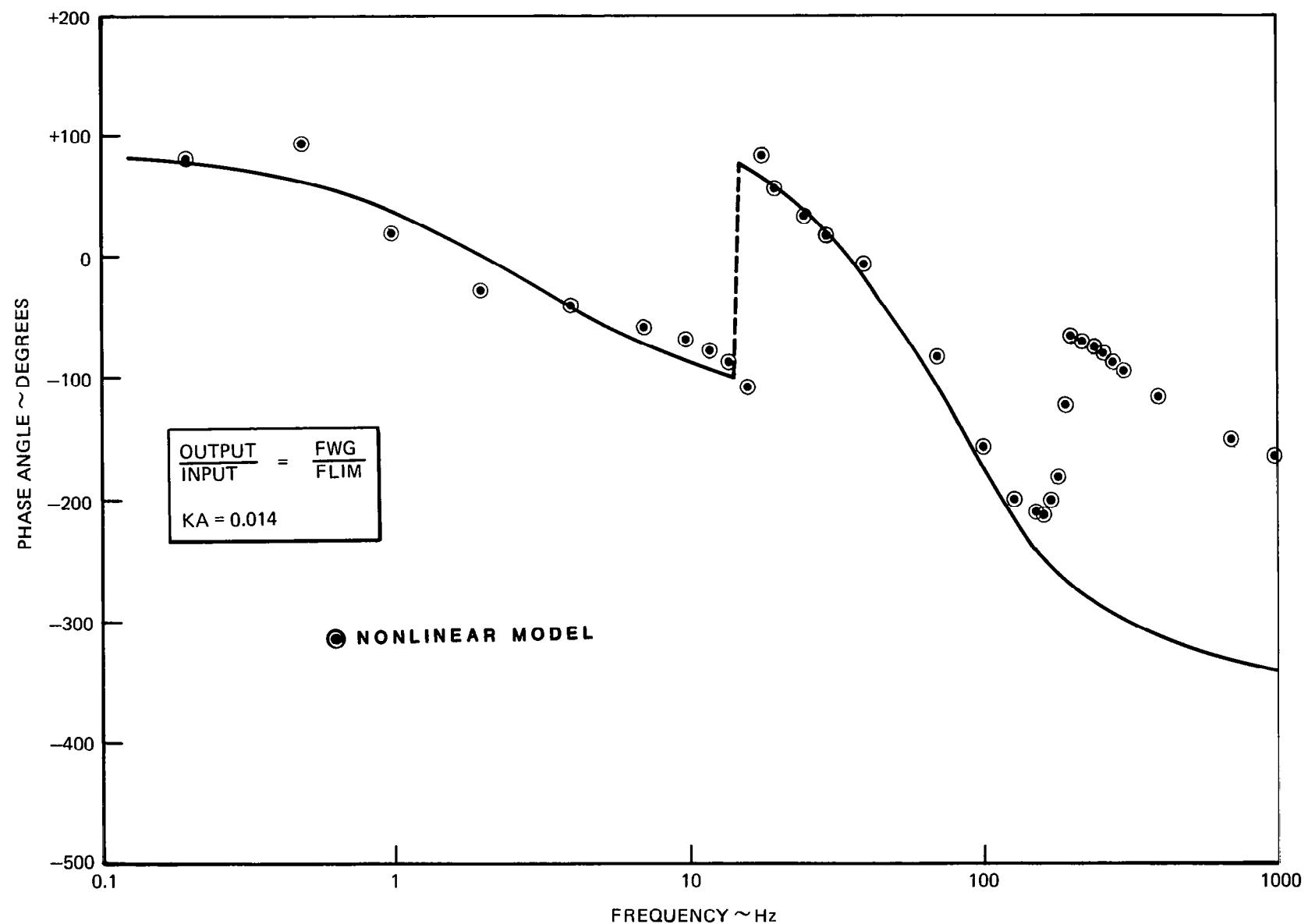


FIGURE 3-5. F-4 GEAR, OPEN-LOOP, NO COMPENSATION FREQUENCY RESPONSE

### 3.5 LOOP COMPENSATION

The open-loop Nyquist diagrams of the uncompensated system presented in the previous section (Figures 3-6 and 3-7) indicate that the system is unstable around 100 Hertz. Thus, compensation is deemed necessary. The compensation that was developed for this system is implemented in the forward path of the control loop, and has the following transfer function.

$$T(s) = \left[ \frac{\frac{s^2}{565.} + \frac{2(.100)}{565.}}{\frac{s^2}{565.} + \frac{2(5.10)}{565.}} \right] \left[ \frac{\frac{s}{617.} + 1}{\frac{s}{6170.} + 1} \right] \left[ \frac{\frac{s}{1547.} + 1}{\frac{s}{15470.} + 1} \right] \quad (3-1)$$

It consists of a notch filter at 90 Hertz and two first-order 20 dB lead/lag networks. The frequency response of the compensation is shown in Figures 3-8 and 3-9 and the Nyquist plot including compensation is shown in Figure 3-10.

To understand the effect of each part of the compensation network on system dynamics, open-loop Nyquist diagrams obtained from the linear model are presented with successive portions of the compensation network incorporated. Figure 3-11 shows the uncompensated Nyquist diagram (this is the same as the results in Figure 3-6 except that the amplifier gain has been adjusted). Figure 3-12 shows the effect of including the compensation notch only. The system is now stable, but rather low damped at a frequency around 60 Hertz. The first lead/lag network was included to add phase lead in this frequency range. The Nyquist diagram with the notch and this lead/lag incorporated is shown in Figure 3-13. The second lead/lag was included to add phase lead in the 190 Hertz range. The open-loop Nyquist diagram with the entire compensation network included is presented in Figure 3-14.

The effect of each part of the compensation network on system dynamics was also evaluated using the nonlinear model on a typical vertical drop case. The conditions of the case are as follows:

1. The sink rate prior to impact is 1.83 m/sec (72 ins/sec)
2. The lift is equal to airplane weight at initial impact, then linearly reduced to 10 percent of the airplane weight over the next 1 second, then held constant at 10 percent thereafter.
3. The ground level is held constant.

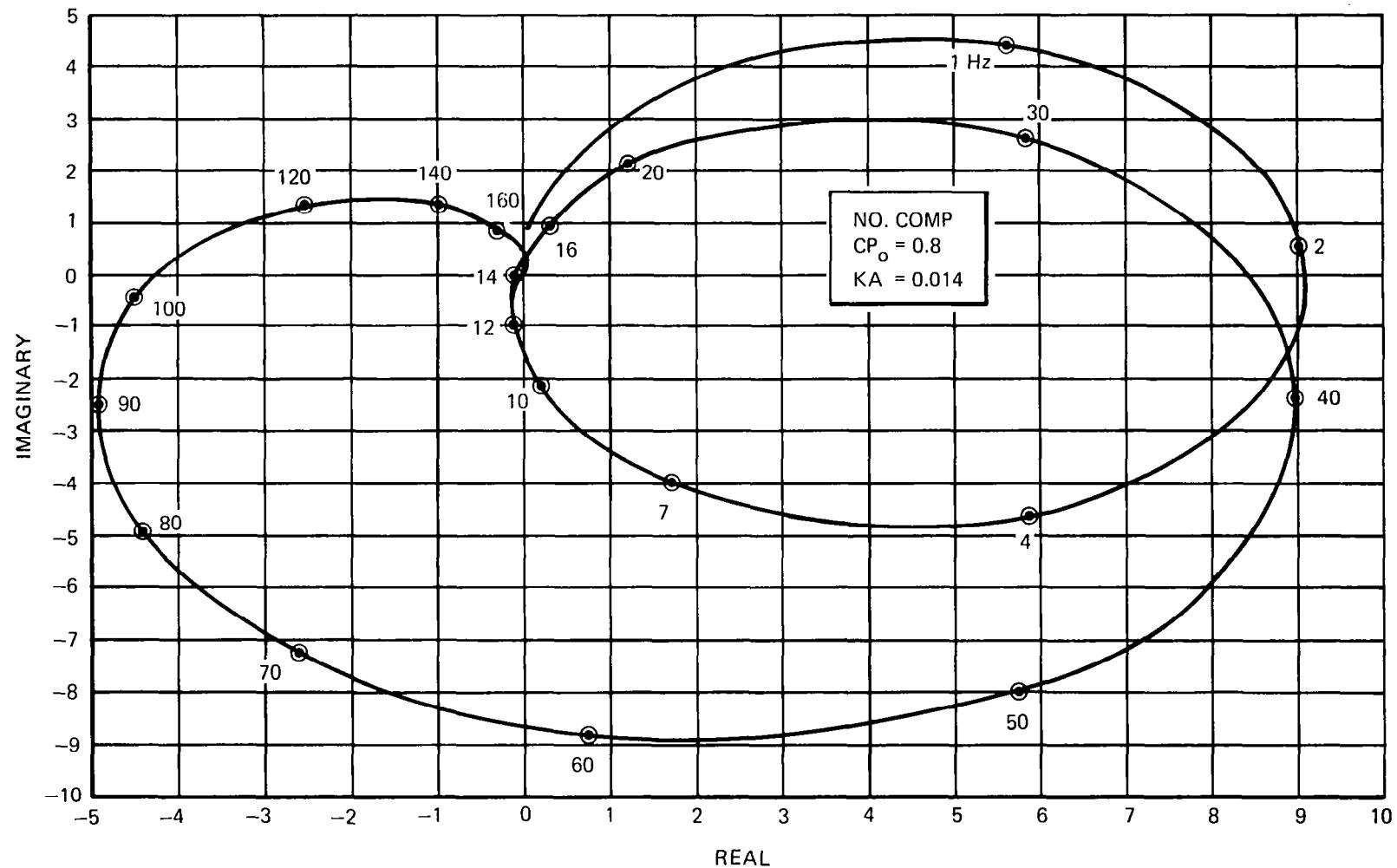


FIGURE 3-6. OPEN-LOOP LINEAR MODEL WITHOUT COMPENSATION

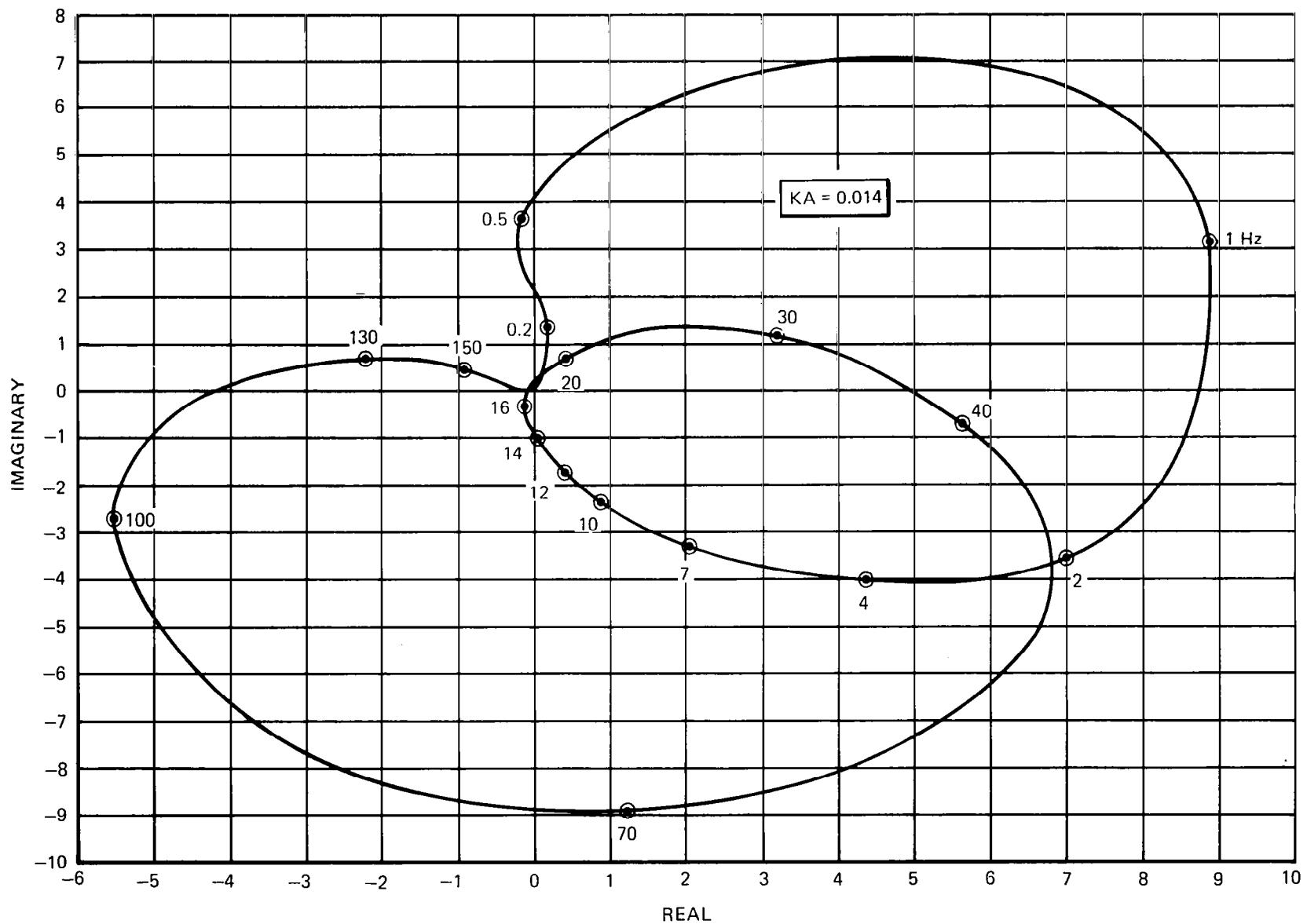


FIGURE 3-7. OPEN-LOOP NONLINEAR MODEL, 889.6N ( $\pm 200$  LBF) INPUT

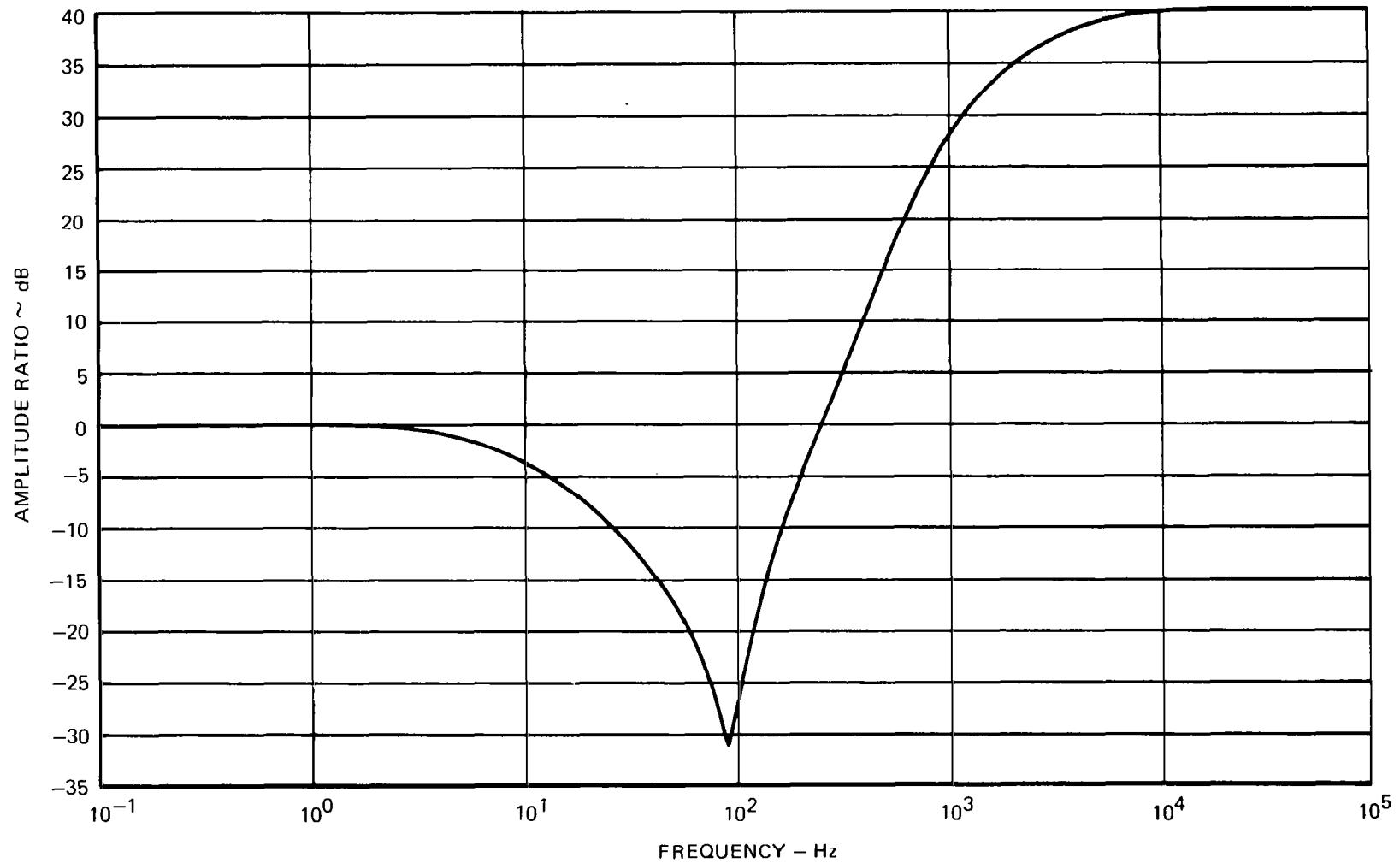


FIGURE 3-8 COMPENSATION

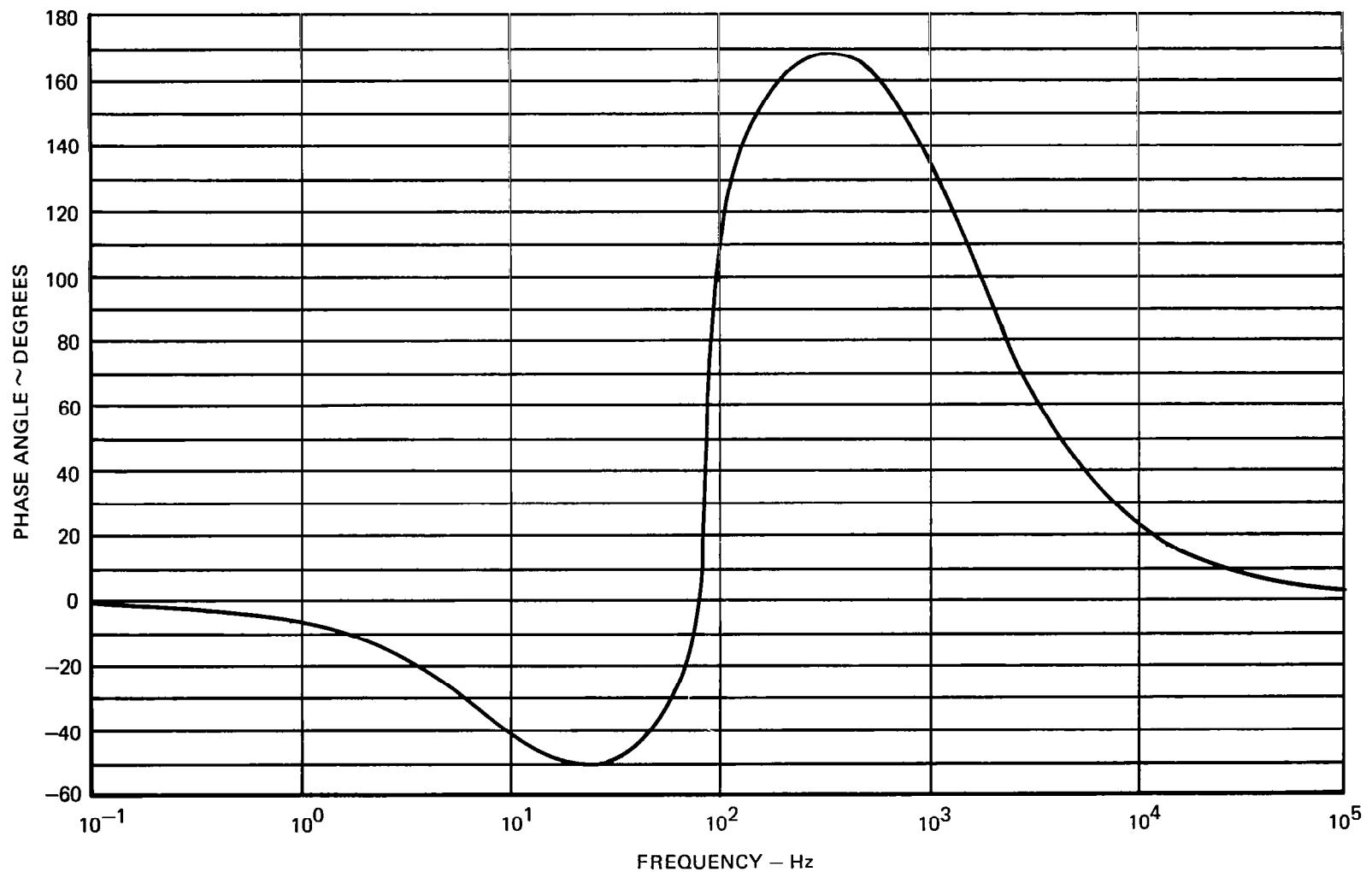


FIGURE 3-9 COMPENSATION

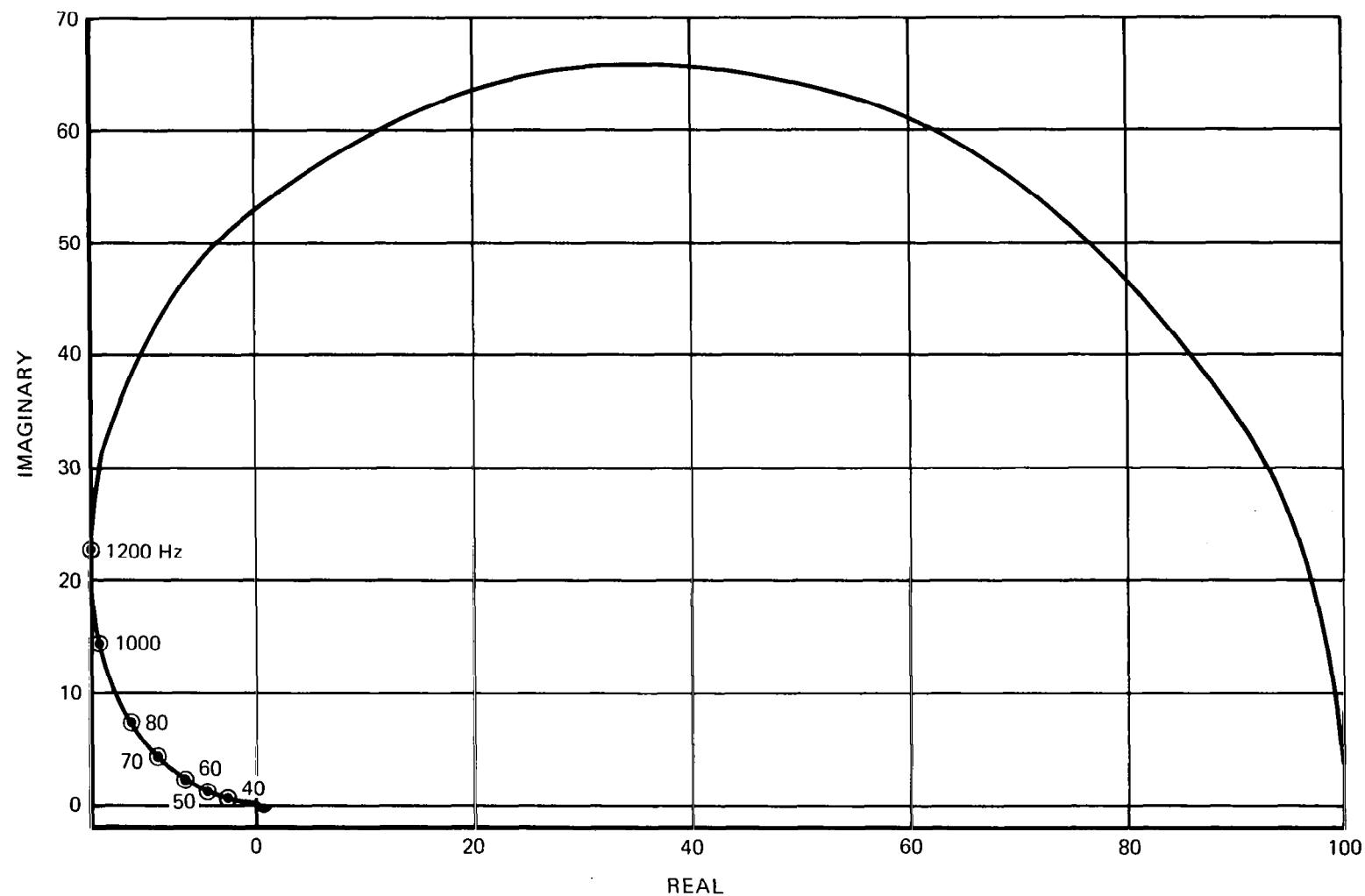


FIGURE 3-10 NYQUIST PLOT—COMPENSATION

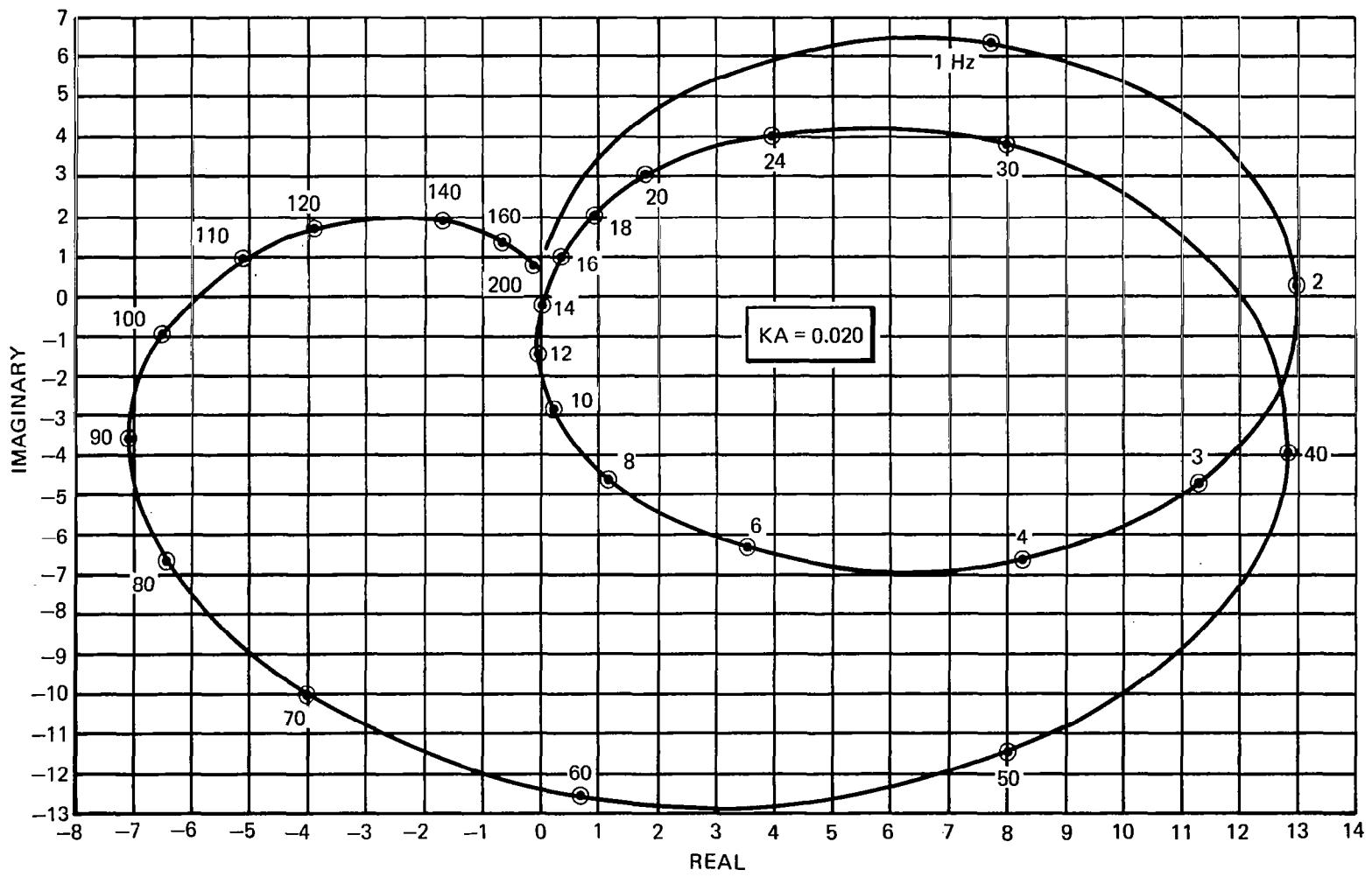


FIGURE 3-11 NO COMPENSATION

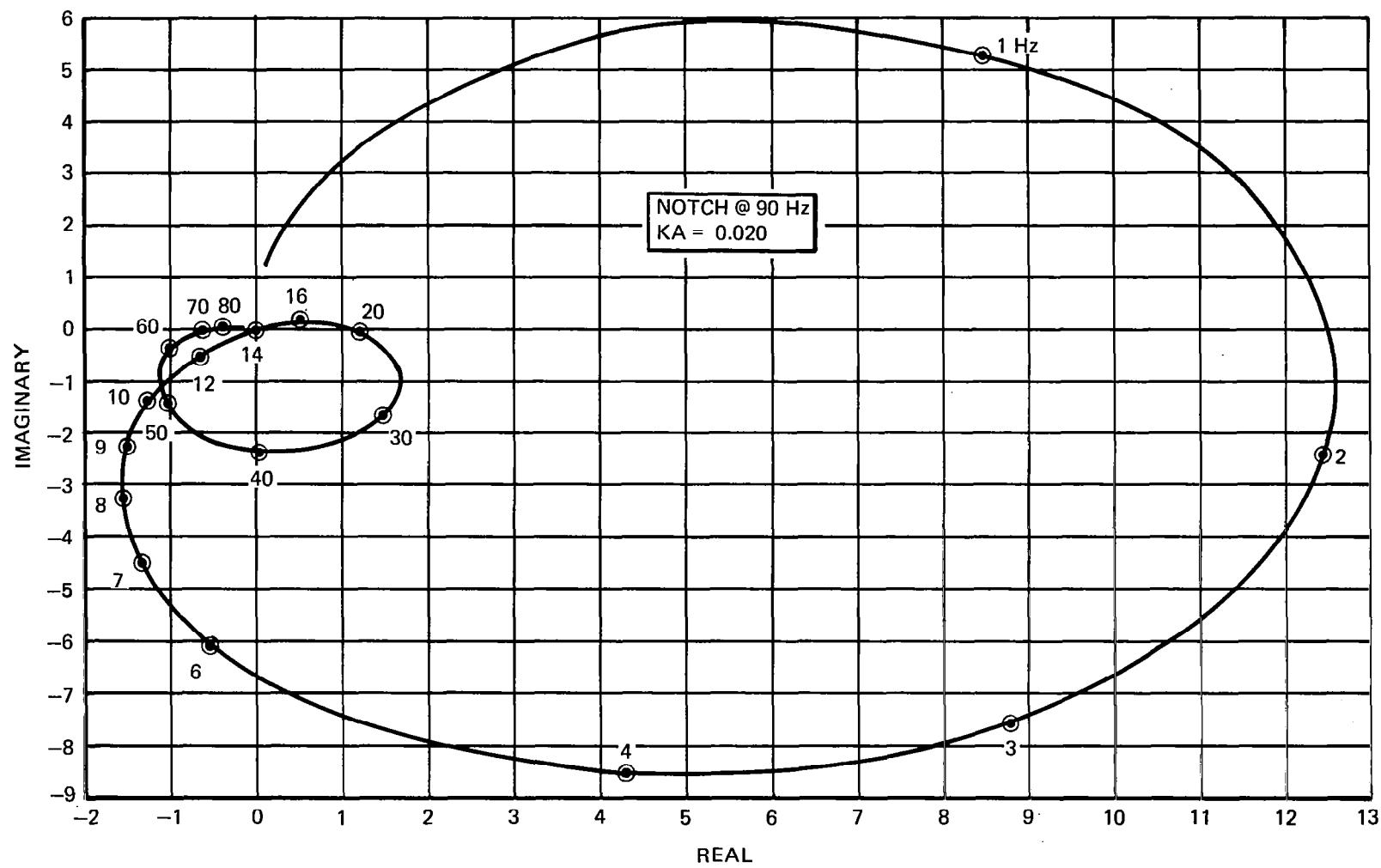


FIGURE 3-12. 90 Hz NOTCH

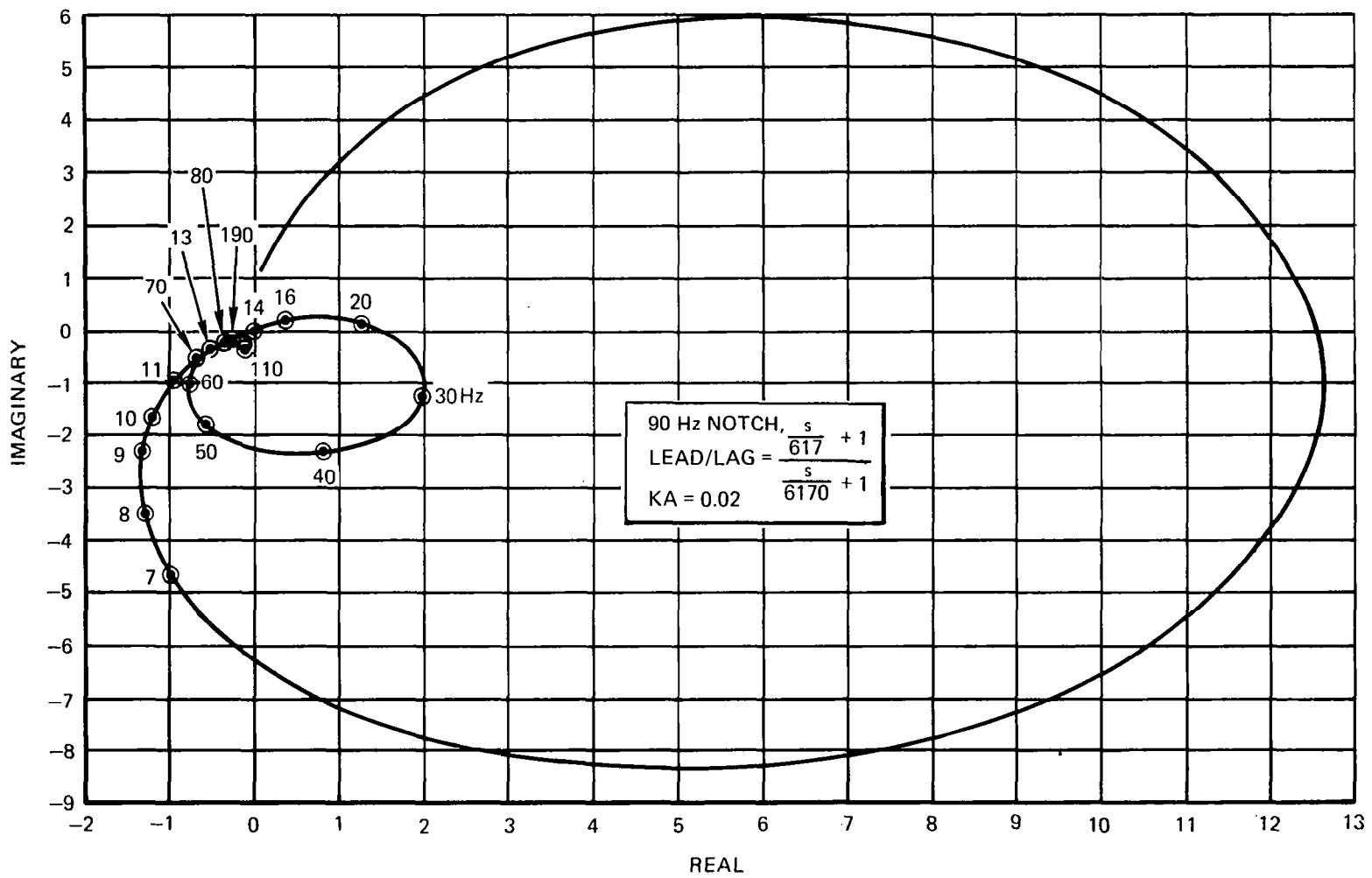


FIGURE 3-13. 90 Hz NOTCH AND LEAD/LAG

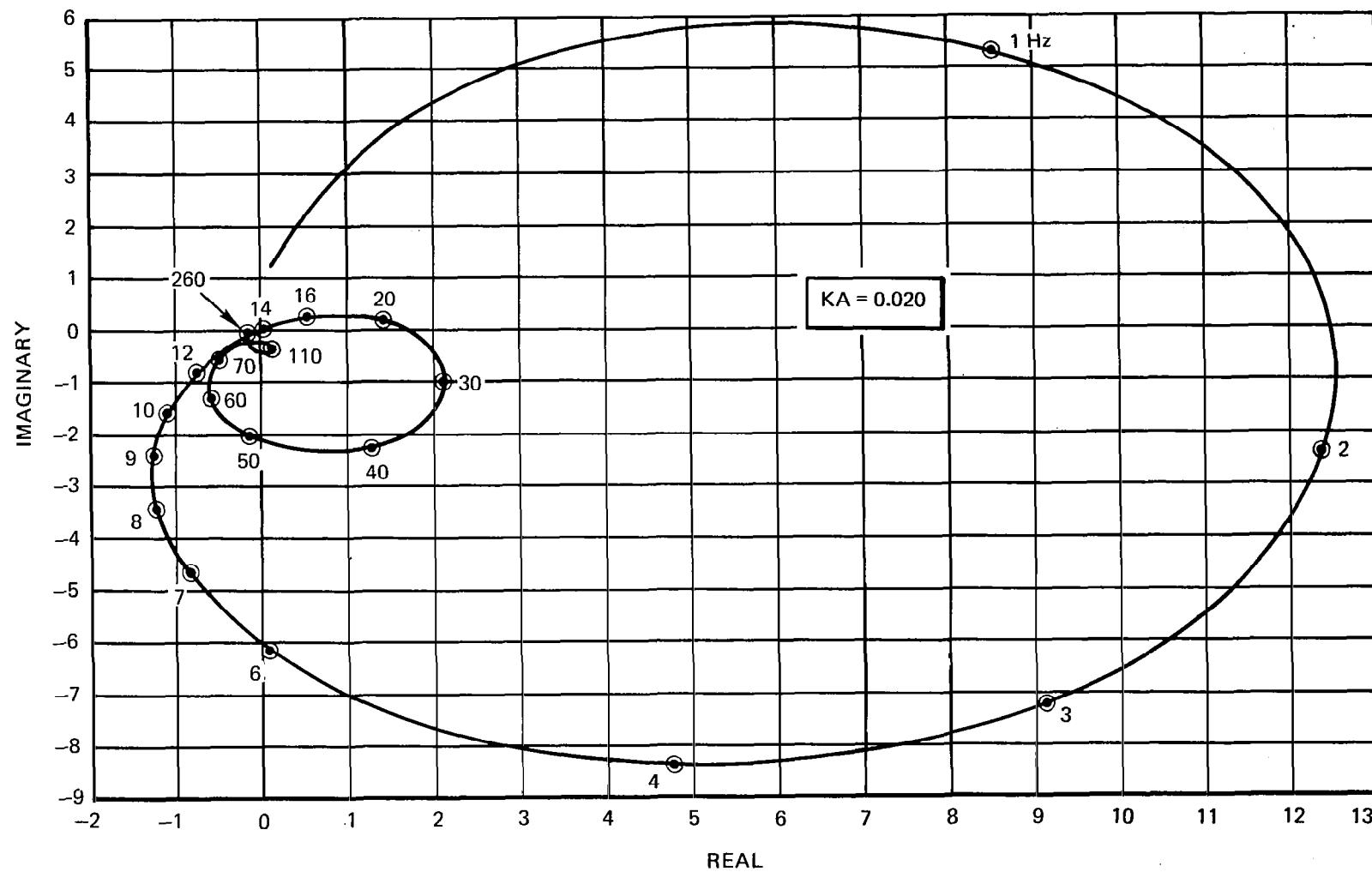


FIGURE 3-14 TOTALLY COMPENSATED LOOP

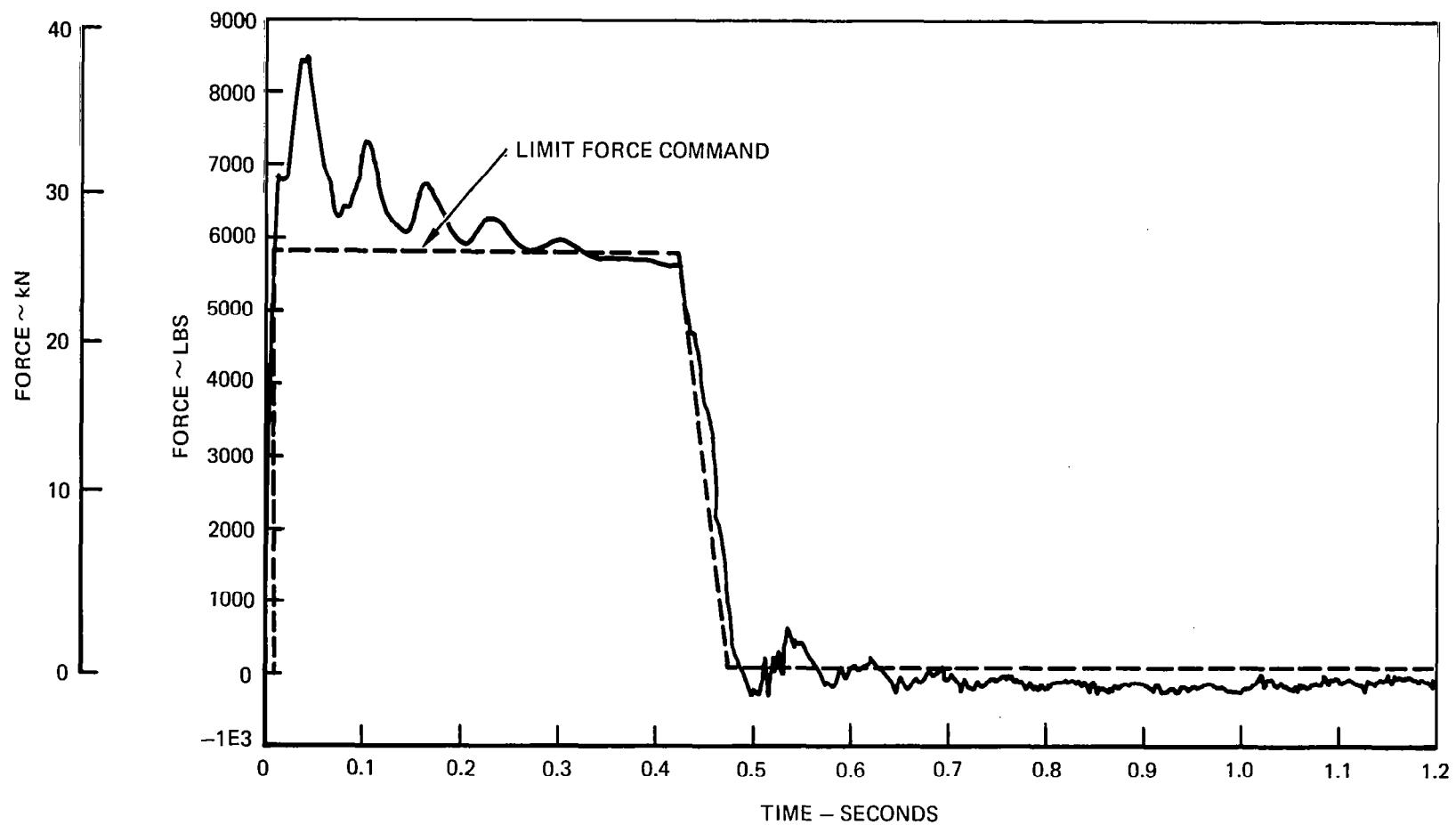


FIGURE 3-15 CASE 2 COMP: NOTCH AT 90 Hz

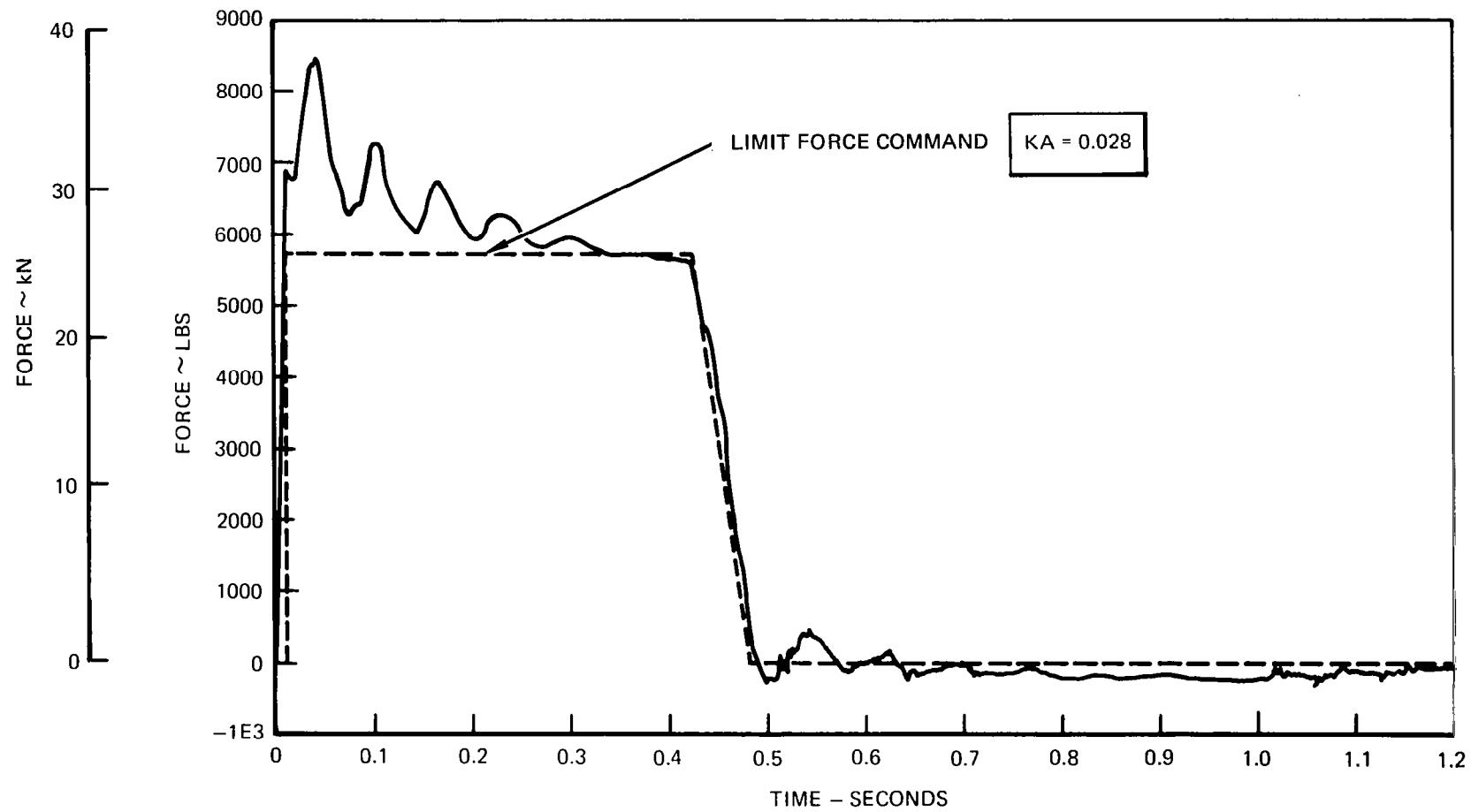


FIGURE 3-16 COMPENSATION – NOTCH @ 90 Hz  
+ LEAD/LAG @ 617, 6170 RAD/SEC

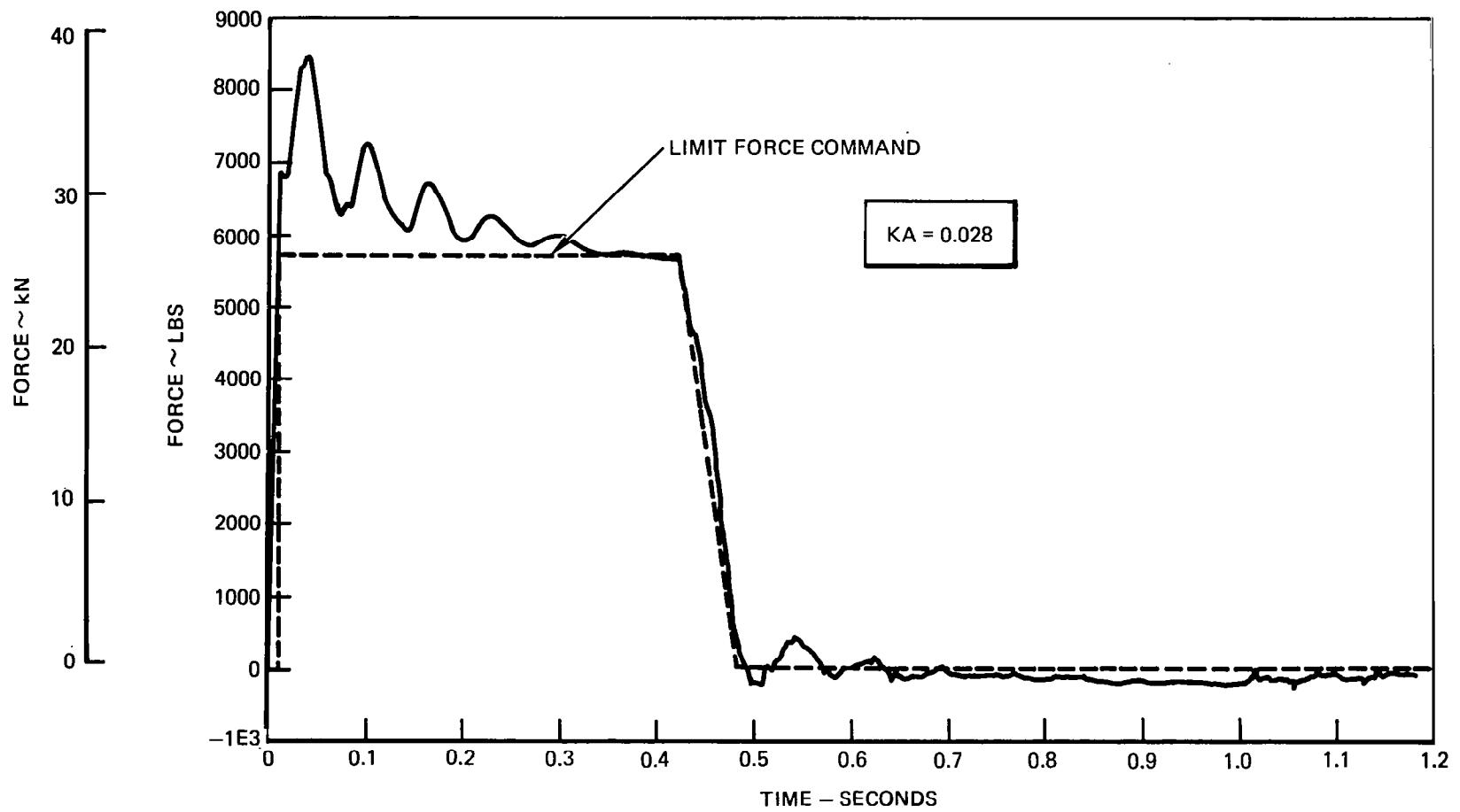


FIGURE 3-17 TOTALLY COMPENSATED LOOP

The resultant force transients are shown in Figures 3-15, 3-16, and 3-17 for the notch compensation only, the notch plus the first lead/lag compensation only, and the entire compensation network, respectively. The results show that the notch stabilizes the system, and the lead/lag networks effectively reduce the oscillatory behavior of the system at the higher frequencies. It should be noted that the amplifier gain was set at 0.028 milliamperes per volt for each of these runs, which is higher than the final design value of 0.020.

Note from Figures 3-15 through 3-17 that the system exhibits a low damped oscillatory behavior at about 15 Hertz. This behavior is also exhibited in the Nyquist diagrams already presented. The linear model Nyquist diagram results predict a somewhat lower frequency of oscillation than the nonlinear results, however, (compare Figures 3-6 and 3-7) an attempt was made to increase the damping of these oscillations by adding some phase lead in that frequency range using another 20 dB lead/lag network. Although the resultant linear model Nyquist diagram looked promising, the nonlinear vertical drop results showed marginal improvement in the low-frequency oscillatory behavior. The resultant compensation also possessed significantly greater high-frequency amplification, an undesirable result. The approach was thus taken to employ the compensation network described previously (Equation 3-1), and improve the low frequency oscillations by reducing the loop gain as much as possible without significantly degrading the performance of the active control concept. It was found that the amplifier gain of 0.028 milliamperes/volt used in Figures 3-15 through 3-17 could be reduced to 0.020 mA/V without significantly affecting the ability of the active control gear to reduce the wing/gear forces, for all the cases run herein.

The block diagram of the system is shown in Figure 3-18.

### 3.6 VERTICAL DROP ANALYTICAL RESULTS

The nonlinear model was used to simulate various vertical drop landings and rollouts over repaired bomb craters using active control on the F-4 landing gear. In all cases the passive gear was also simulated in order to evaluate the effectiveness of active control in reducing the loads transmitted through the wing/gear interface. The compensation developed in Section 3.5 (Equation 3-1) was employed in all active control cases, and the amplifier gain used was 0.020 mA/V.

#### 3.6.1 Vertical Drop, Case I

The conditions for vertical drop case number 1 are as follows:

1. The sink rate prior to impact is 1.83 m/sec (72 in/sec).
2. The lift equals airplane weight (per gear) at all times.
3. The ground level remains constant.

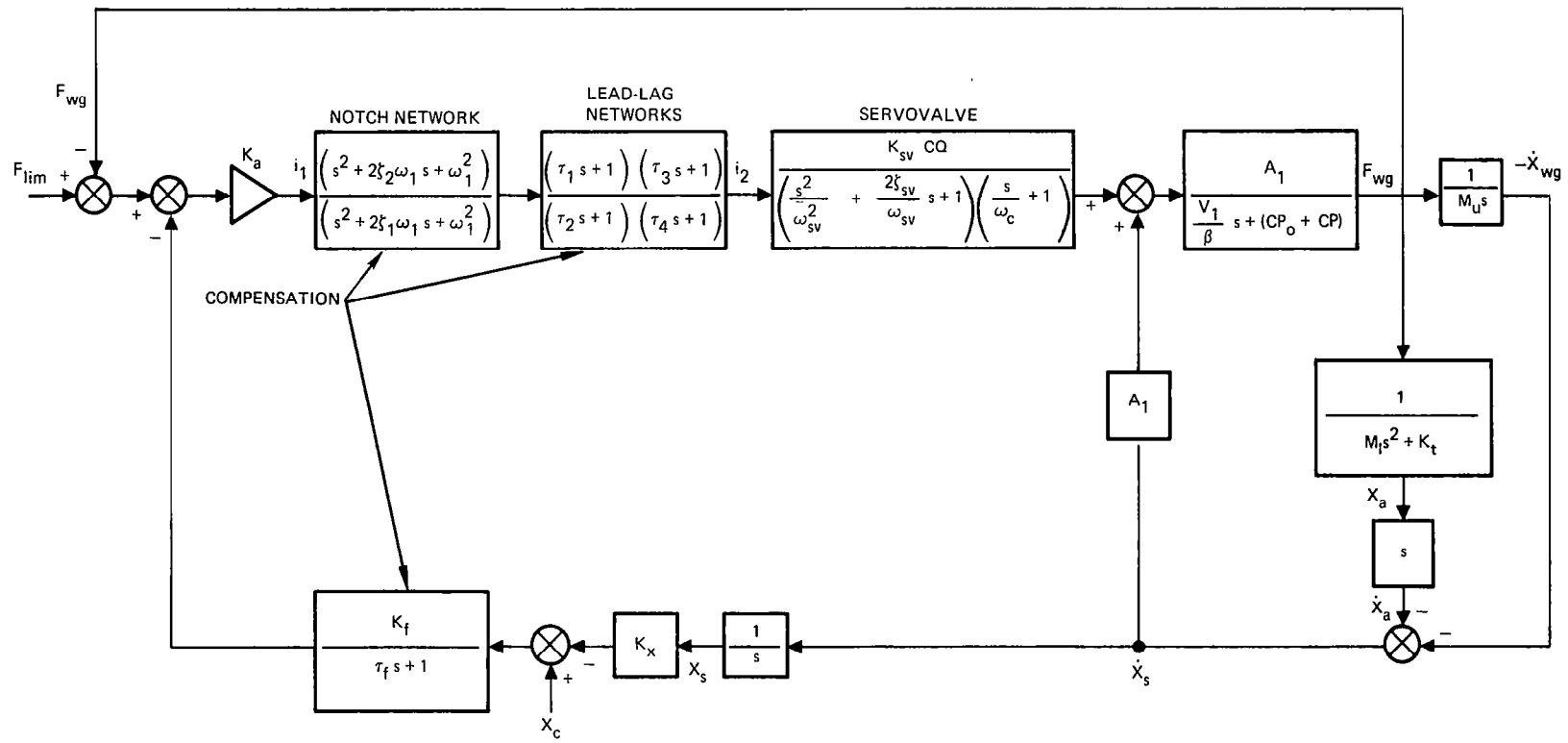


FIGURE 3-18 BLOCK DIAGRAM OF LINEAR MATH MODEL

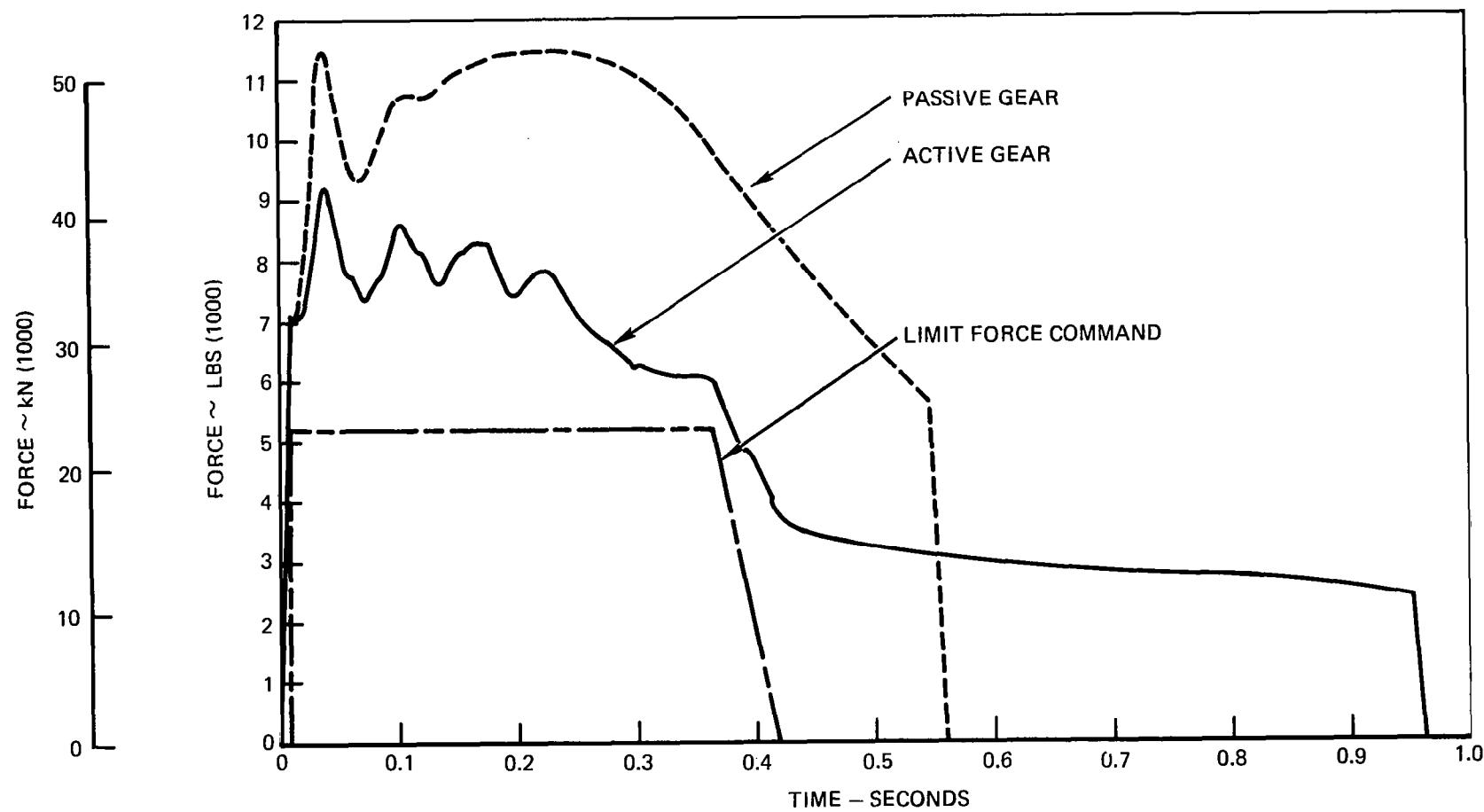


FIGURE 3-19 CASE 1

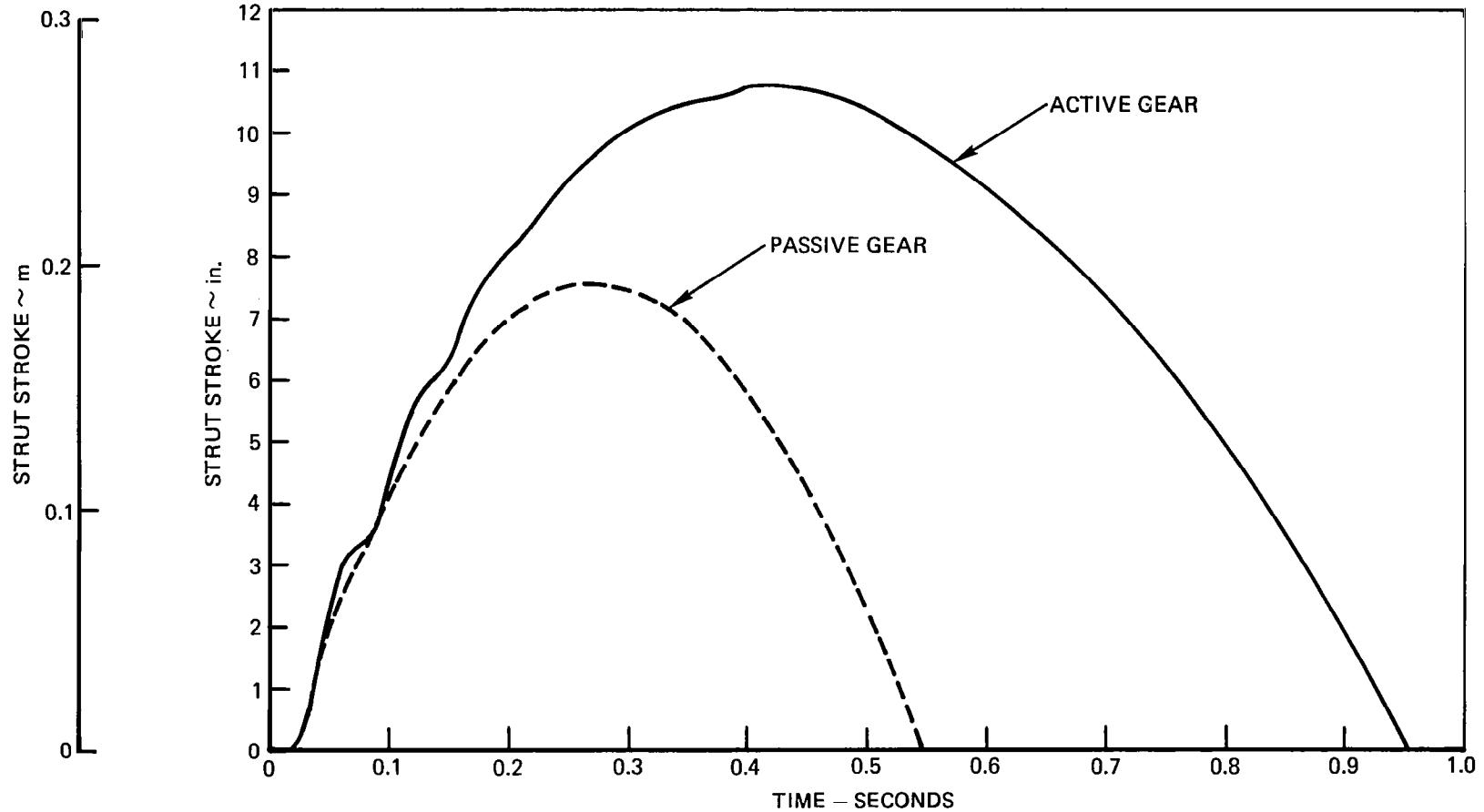


FIGURE 3-20 CASE 1.

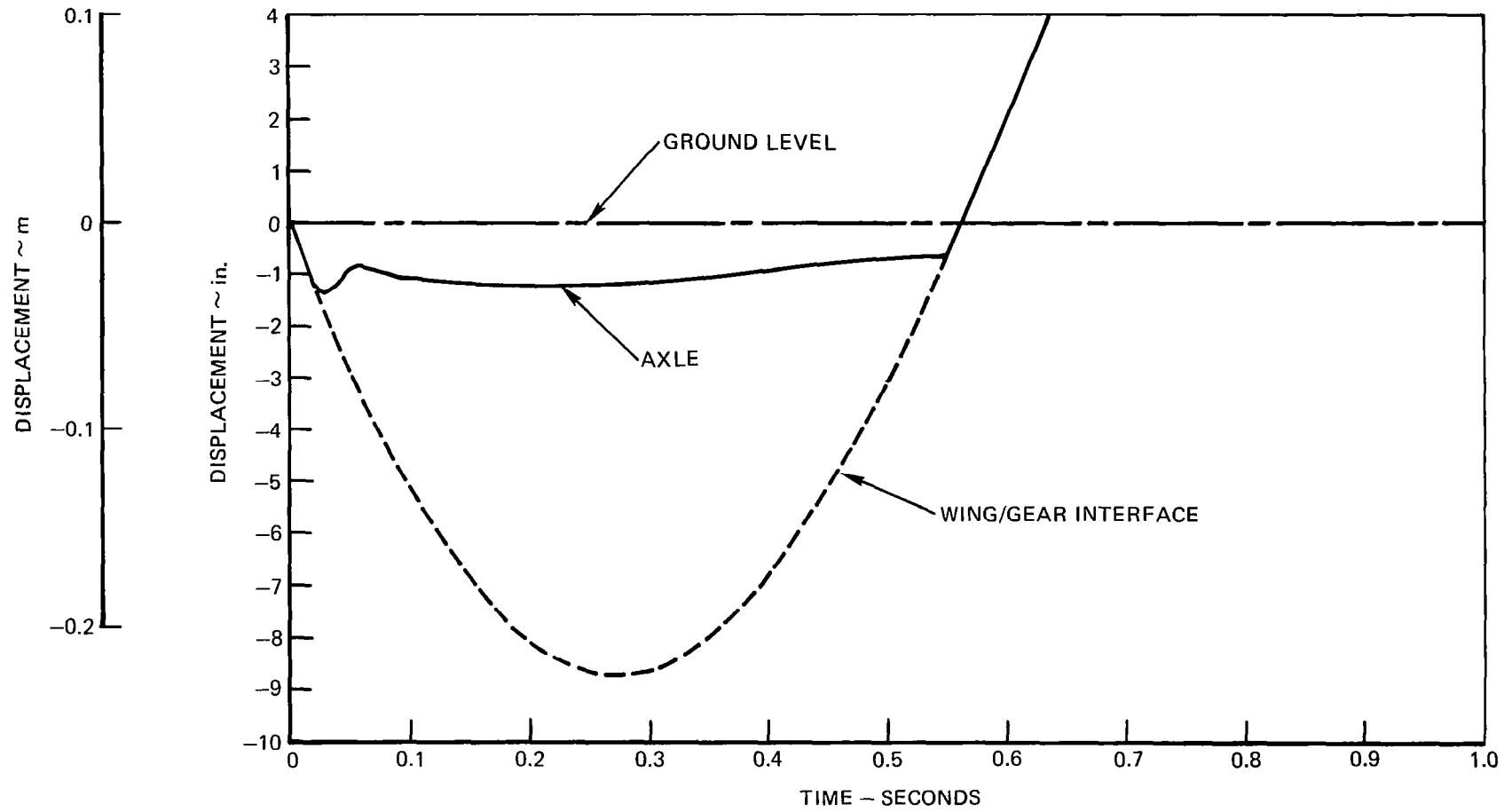


FIGURE 3-21 CASE 1 PASSIVE GEAR

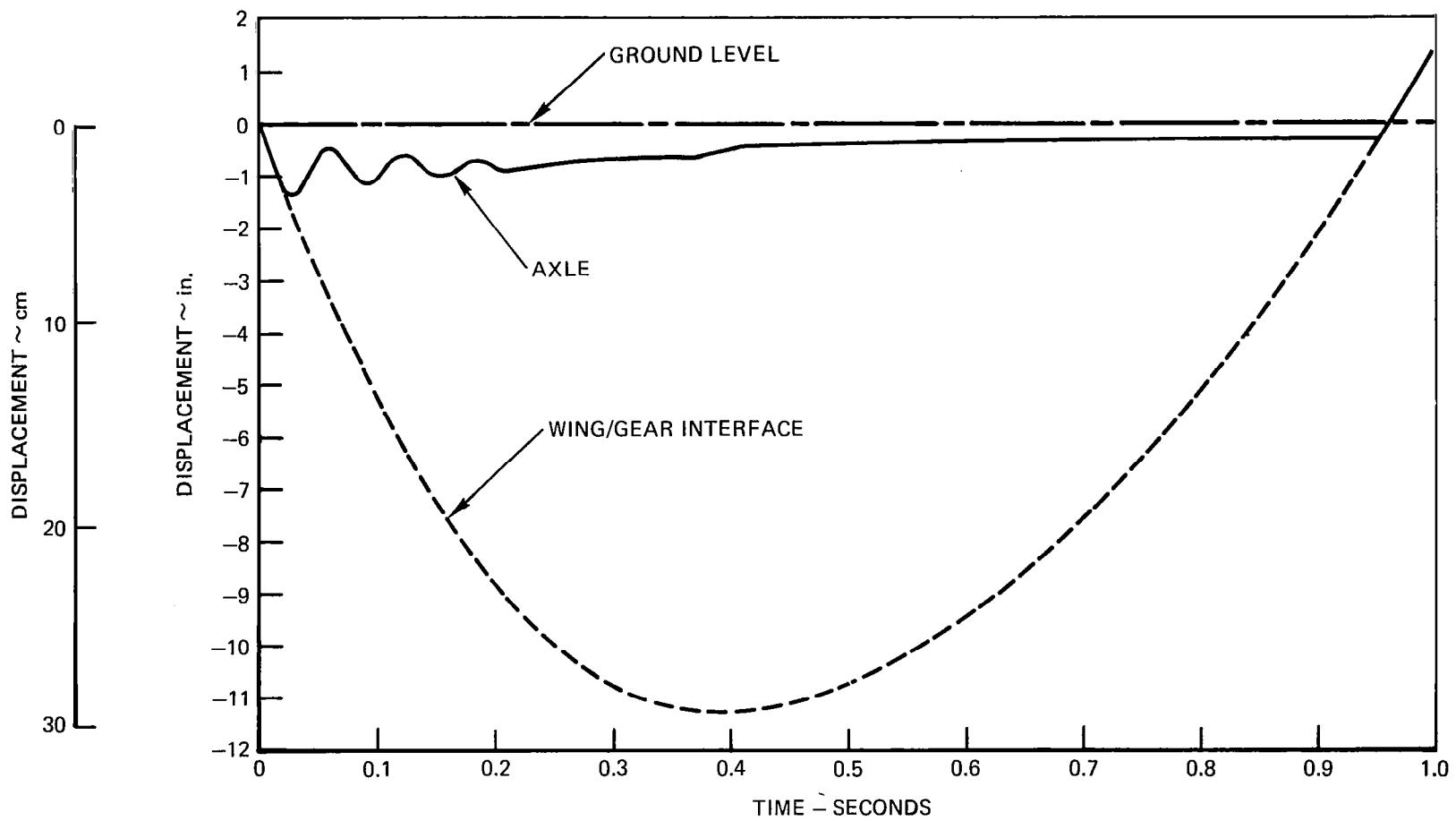


FIGURE 3-22. CASE 1 ACTIVE GEAR

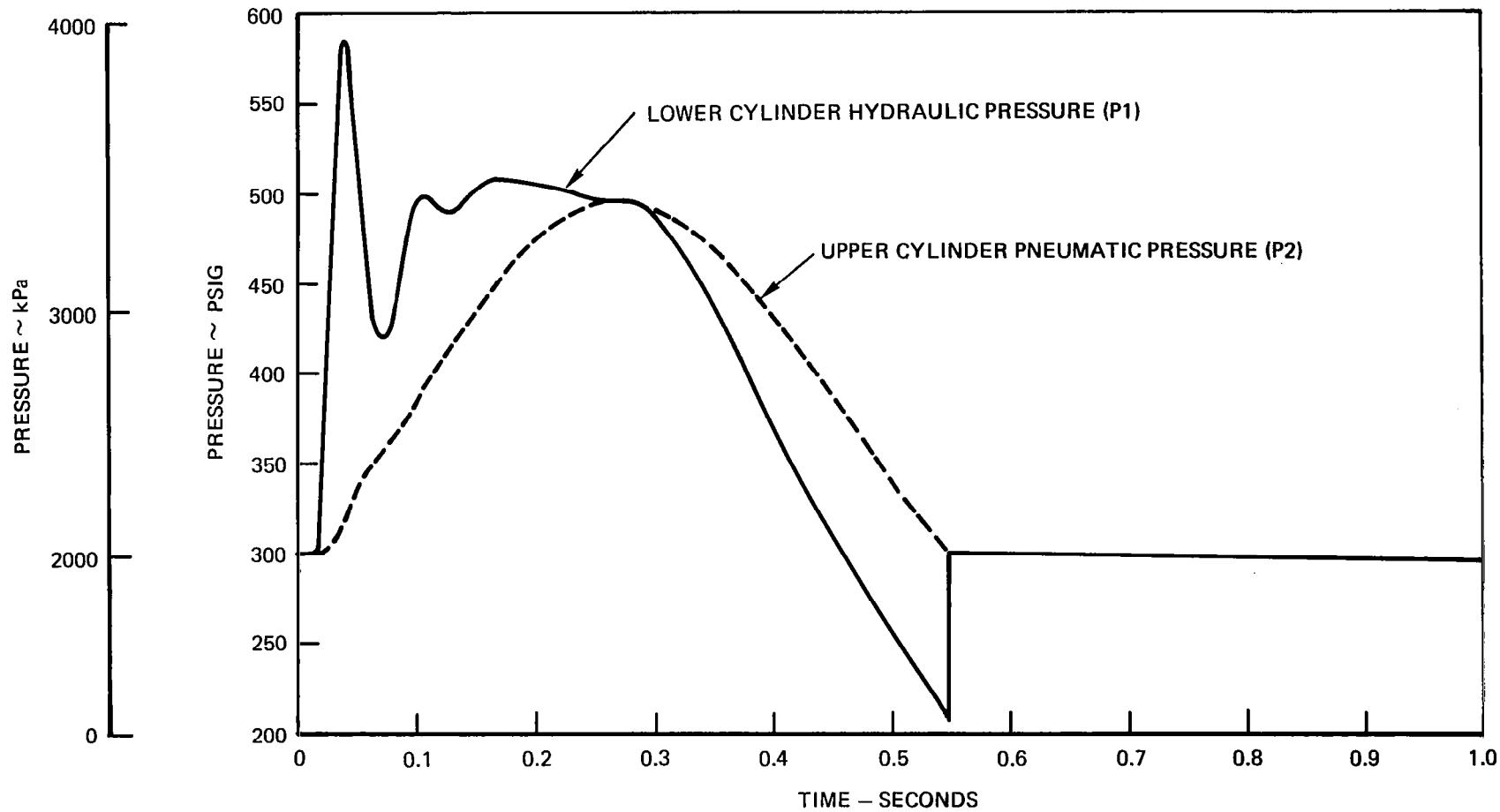


FIGURE 3-23 CASE 1 PASSIVE GEAR

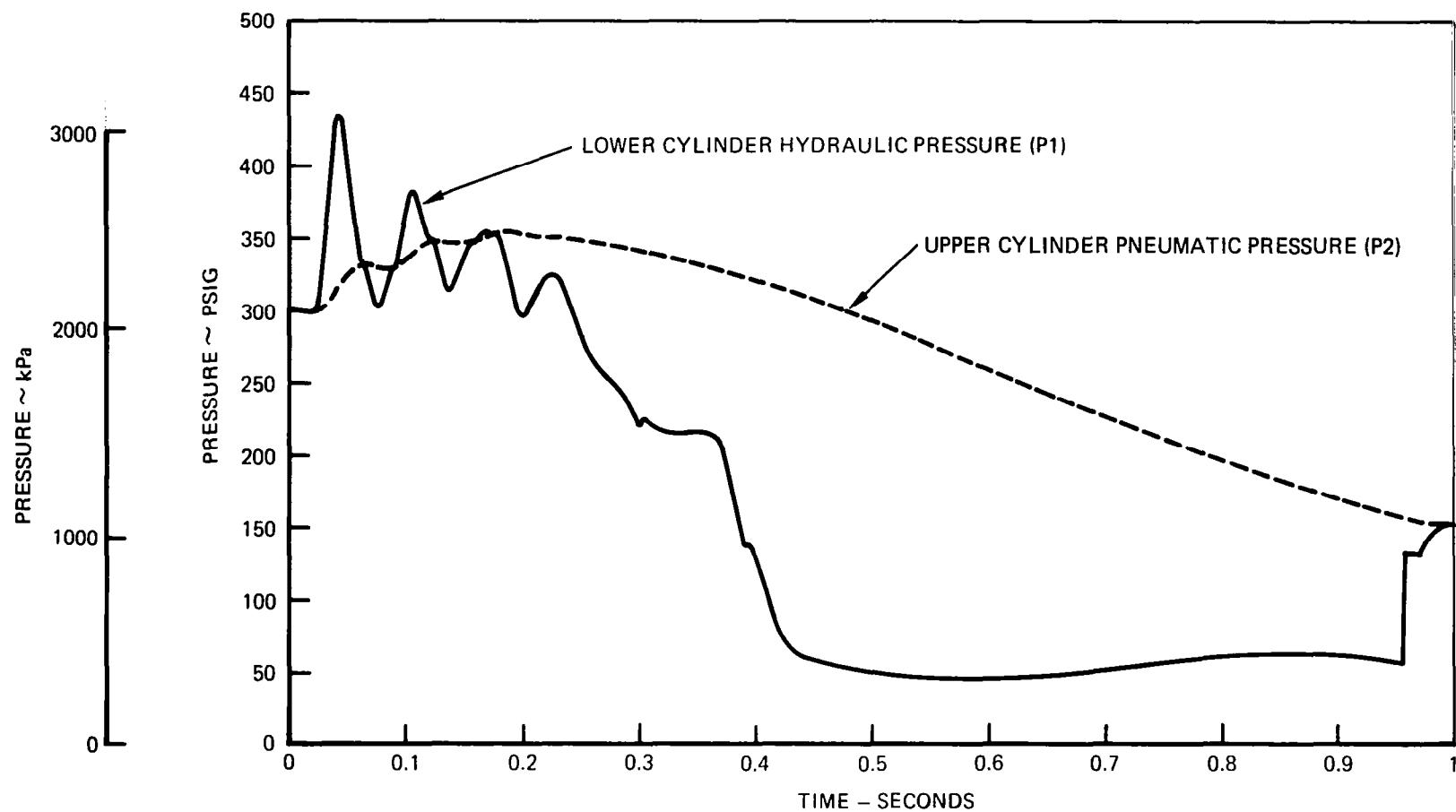


FIGURE 3-24 CASE 1 ACTIVE GEAR

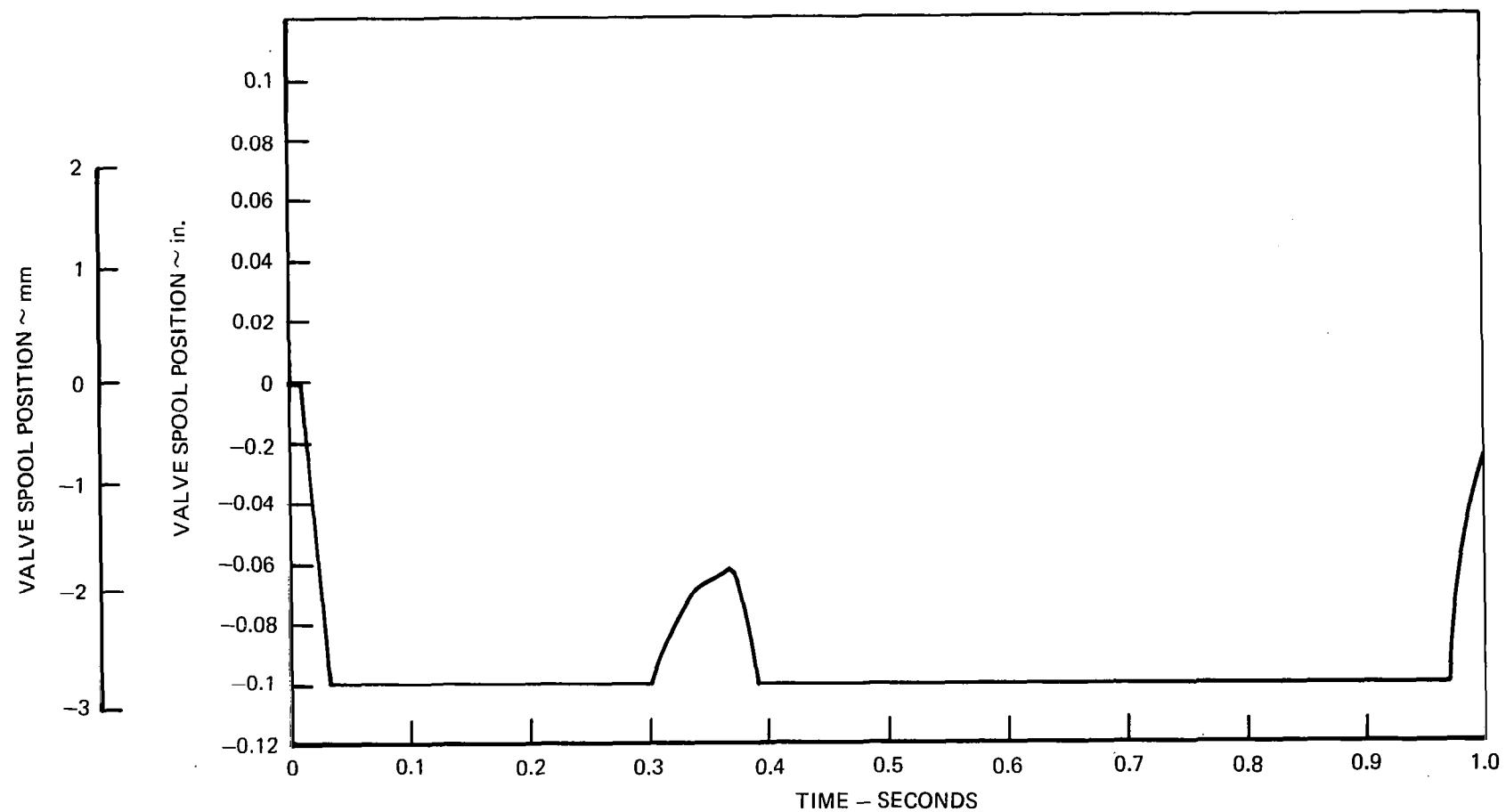


FIGURE 3-25 CASE 1 ACTIVE GEAR

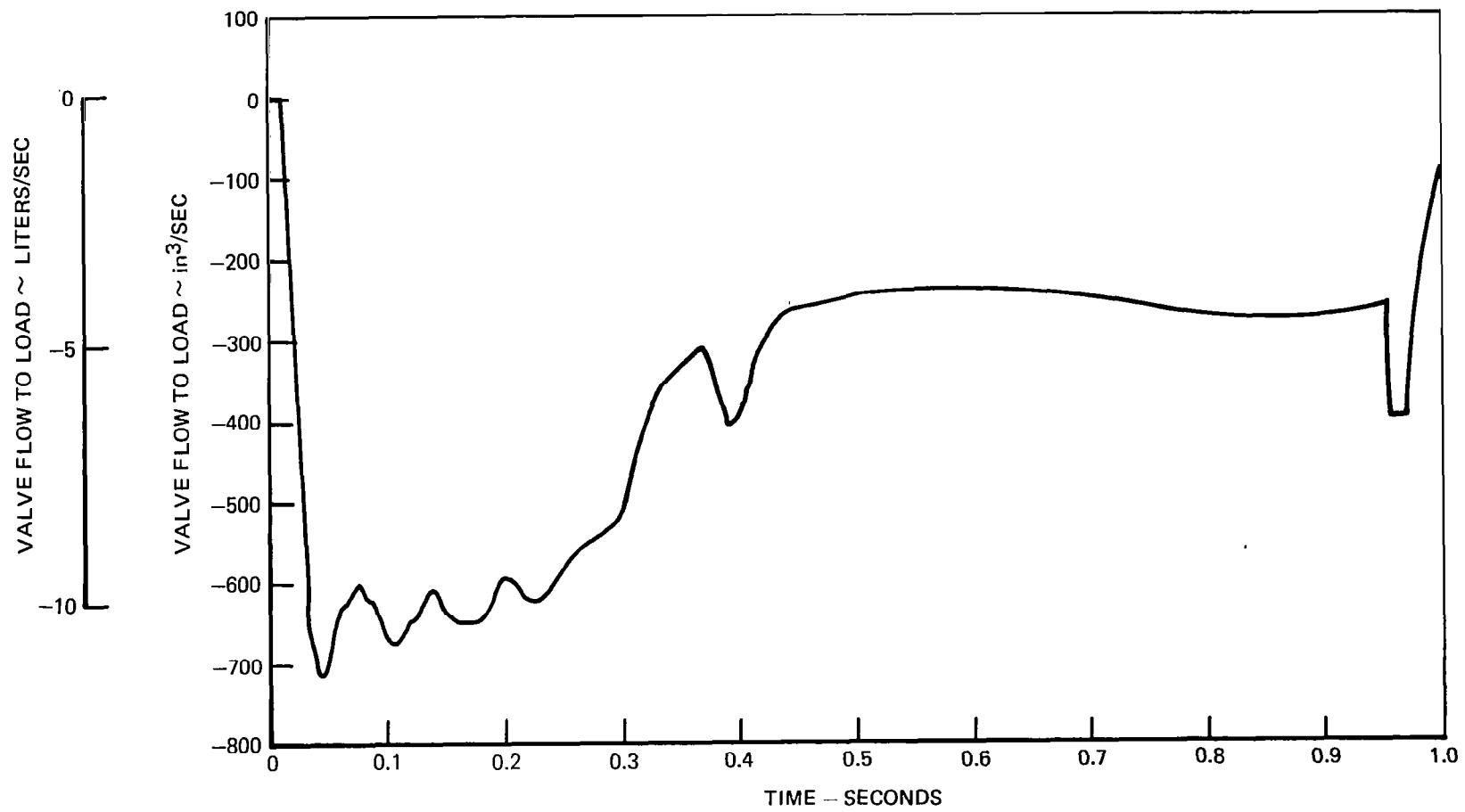


FIGURE 3-26 CASE 1 ACTIVE GEAR

Command limit force is set automatically by the controller. Figure 3-19 shows the resultant wing/gear force transients for the active and passive gears. Active control reduces the peak force 20 percent below the passive gear case. Figure 3-20 compares the strut stroke between the two cases. The active gear uses significantly more stroke than the passive gear. Figures 3-21 and 3-22 show the vertical displacements of the ground level, landing gear axle, and wing/gear interface for the passive gear and the active gear, respectively. The displacements are positive in the up direction, and are all referenced to the condition where the gear is fully extended and barely in contact with the ground with zero tire compression. Thus, at the point of impact (at time = 0) all the variables are zero. When the axle displacement is below the ground level (which is constant), the tire is in compression; when it is above, the landing gear is off the ground. Also, when the wing gear interface displacement is the same as the axle displacement, the gear is fully extended. Thus, in Figure 3-21 for the passive gear, the landing gear becomes fully extended at 0.548 second and rebounds (i.e., leaves the ground) at 0.560 second. Note from Figure 3-22 that the active control causes the gear to remain in contact with the ground longer and when it rebounds, it does so at a lower upward velocity. Figures 3-23 and 3-24 show the lower and upper cylinder pressure transients for the passive and active gears, respectively. The pressures are significantly reduced in both cylinders as a result of active control. Finally, Figures 3-25 and 3-26 show the valve third stage spool displacement and the valve hydraulic flow rate to the gear respectively, for the active control case.

### 3.6.2 Vertical Drop, Case 2

The conditions for vertical drop case number 2 are as follows:

1. The sink rate prior to impact is 1.83 m/sec (72 in/sec).
2. The lift equals airplane weight (per gear) prior to and up to the point of impact, then lift is linearly reduced to 10 percent of airplane weight during the first second after impact, and lift is held constant at ten percent thereafter.
3. The ground level remains constant. Figures 3-27 through 3-34 show the transient response of the various variables of interest for the passive and active gear simulations. Active control in this case reduces the peak wing/gear force 22 percent below the passive gear case.

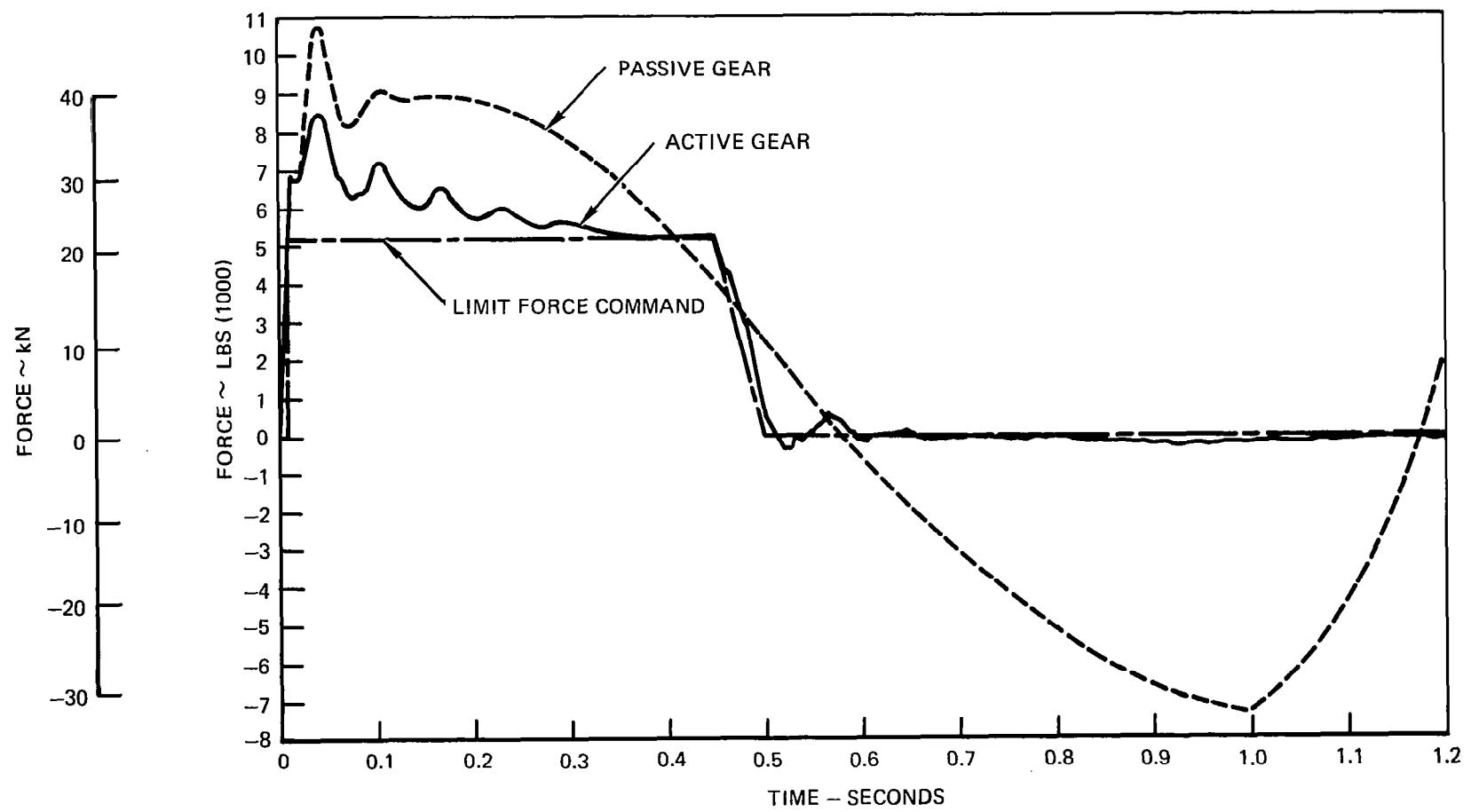


FIGURE 3-27 CASE 2

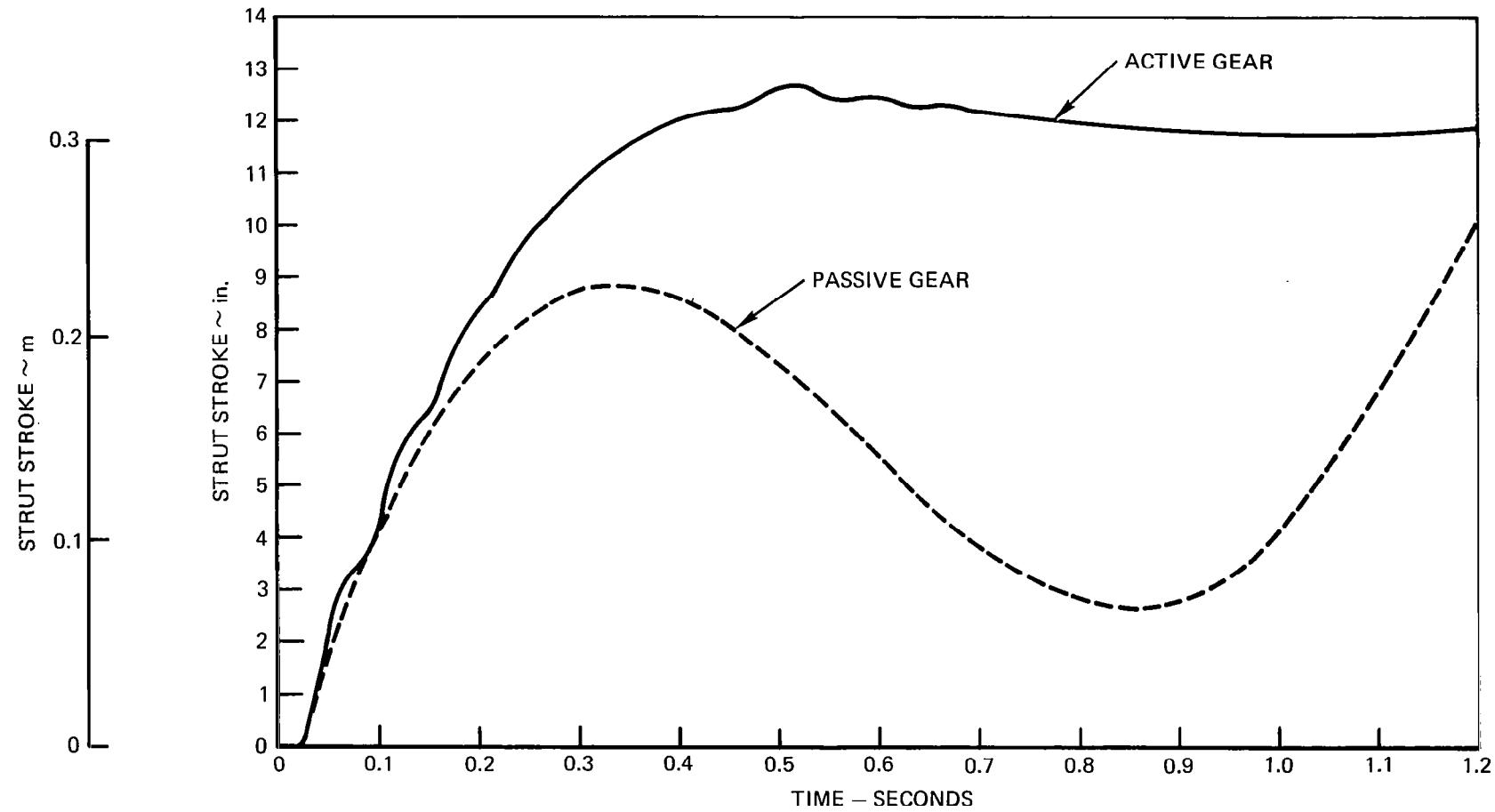


FIGURE 3-28 CASE 2

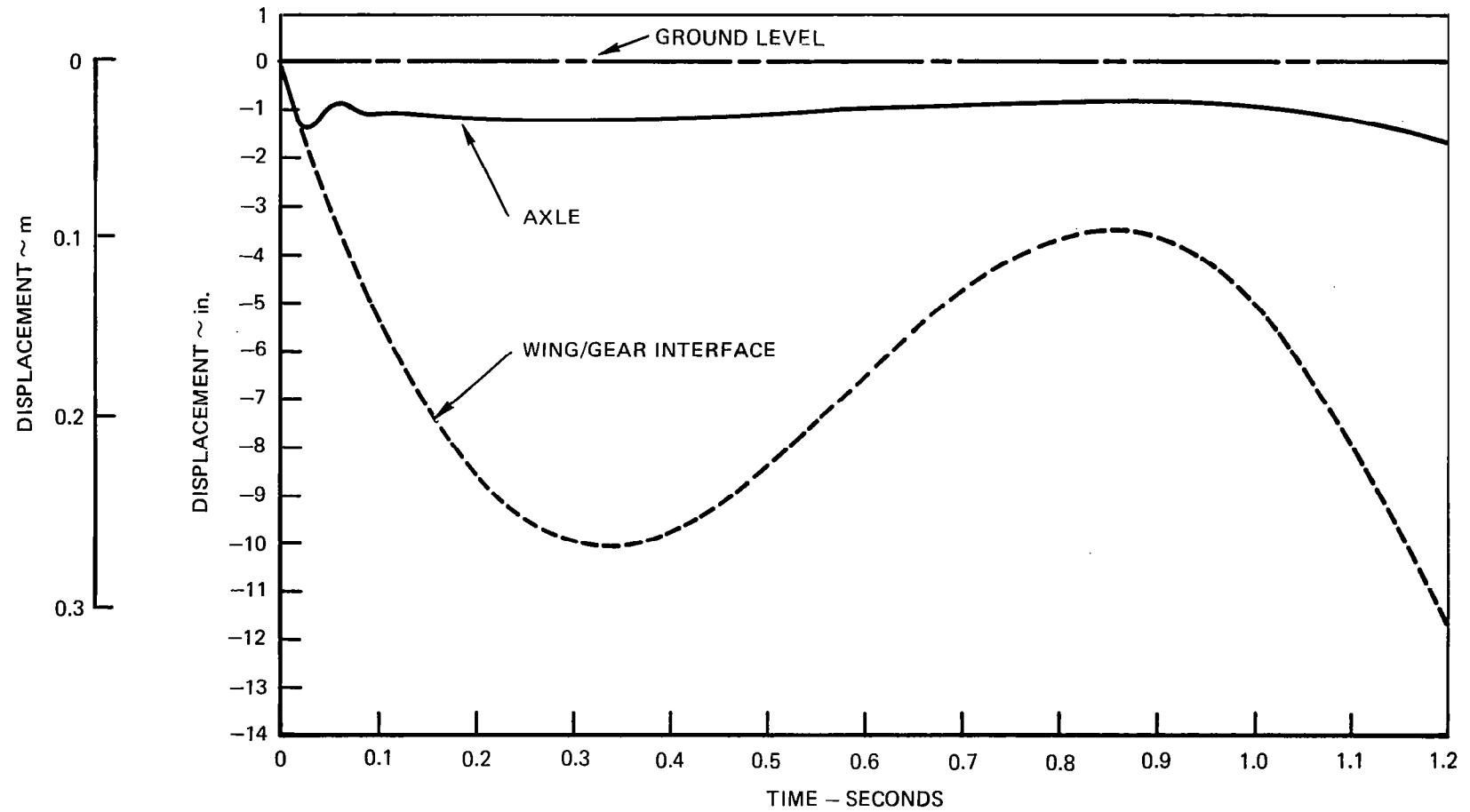


FIGURE 3-29 CASE 2 PASSIVE GEAR

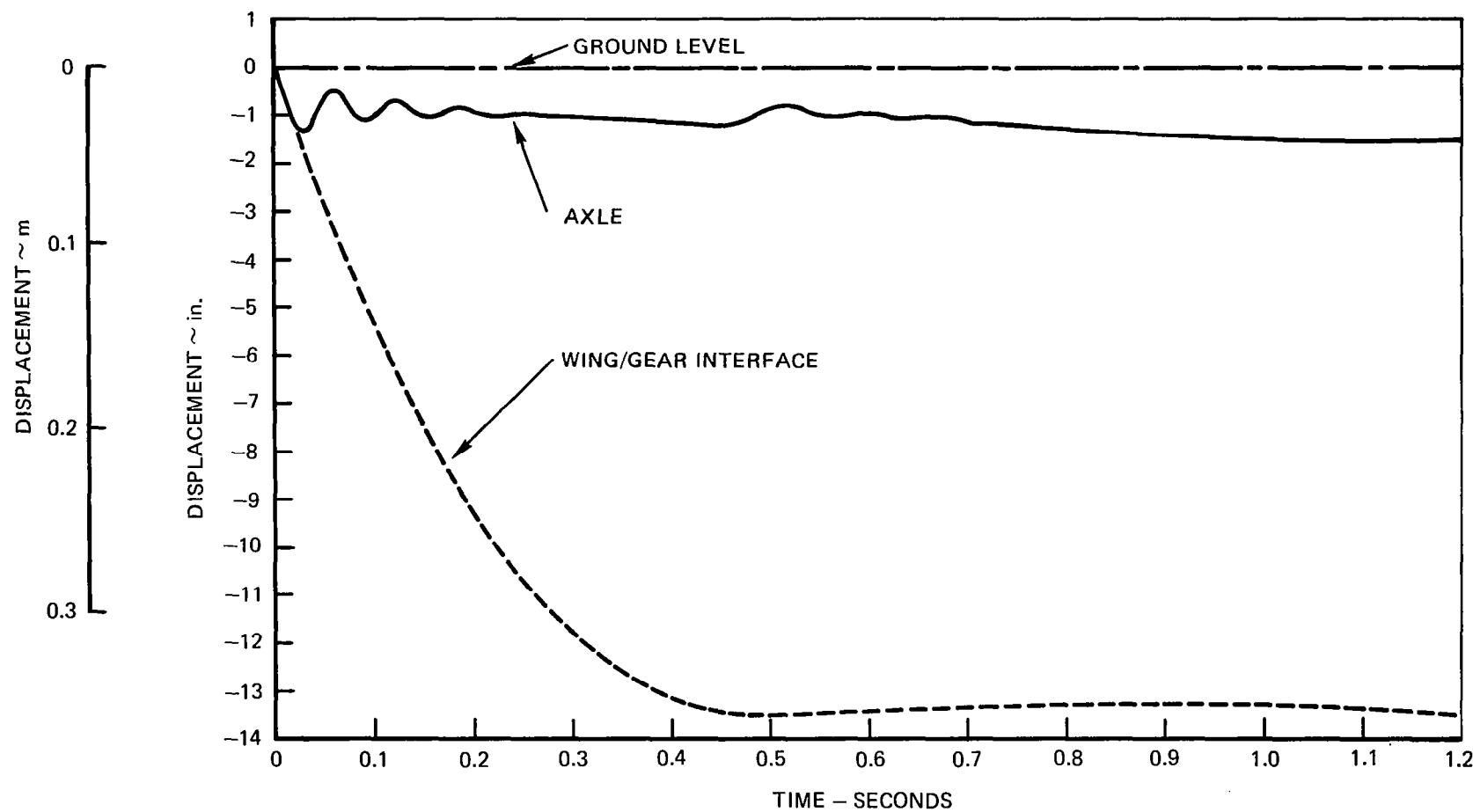


FIGURE 3-30 CASE 2 ACTIVE GEAR

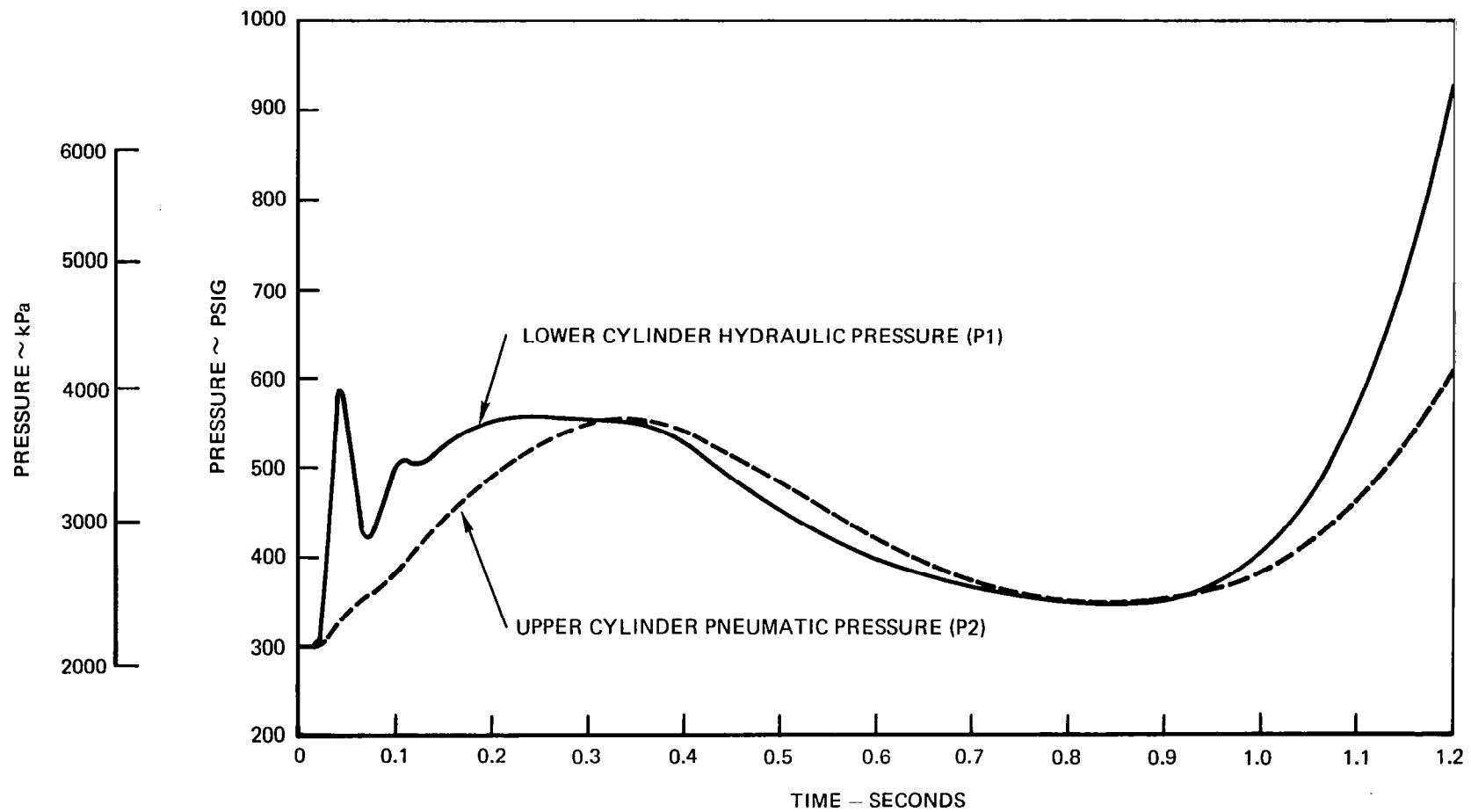


FIGURE 3-31 CASE 2 PASSIVE GEAR

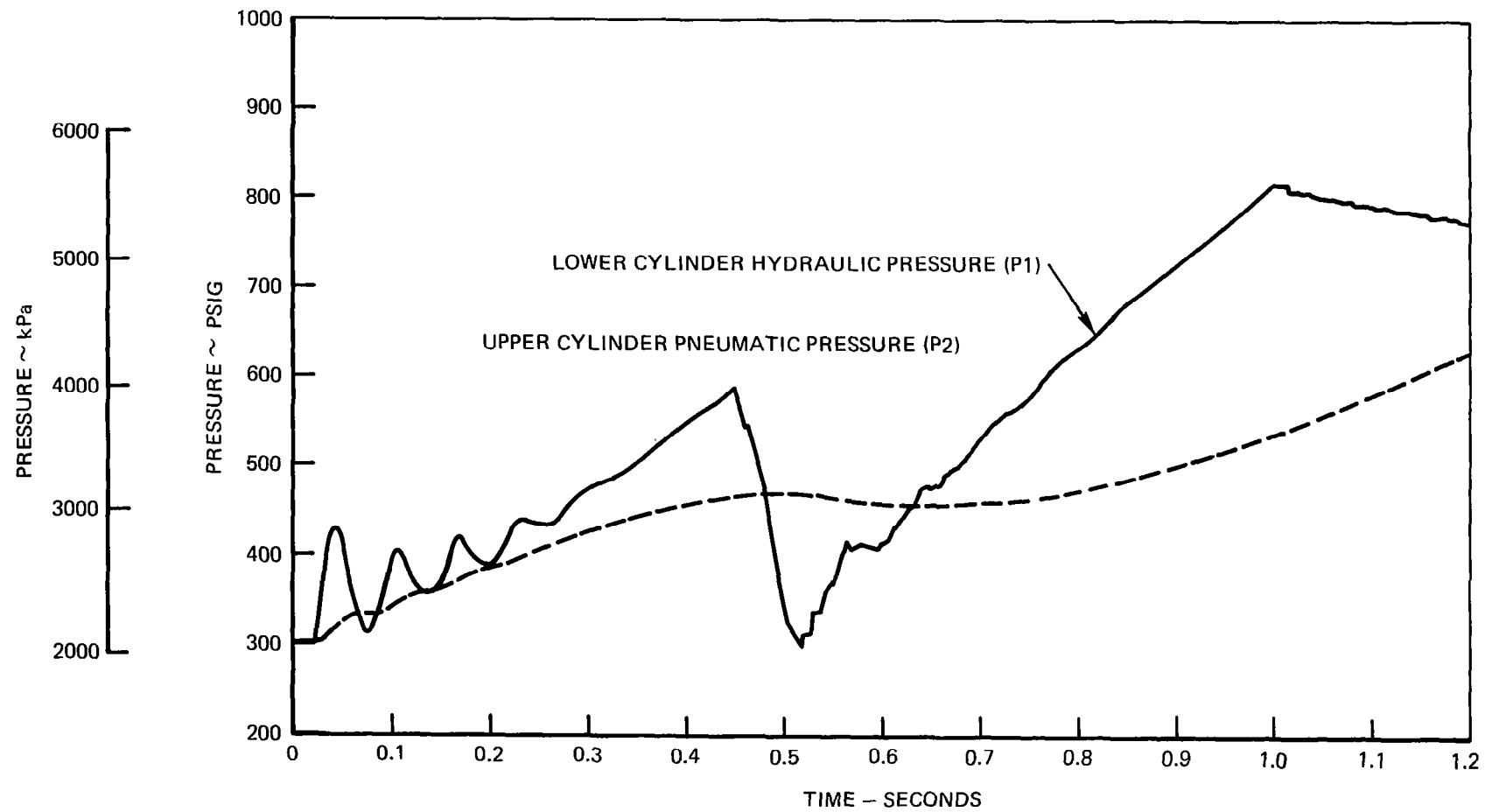


FIGURE 3-32 CASE 2 ACTIVE GEAR

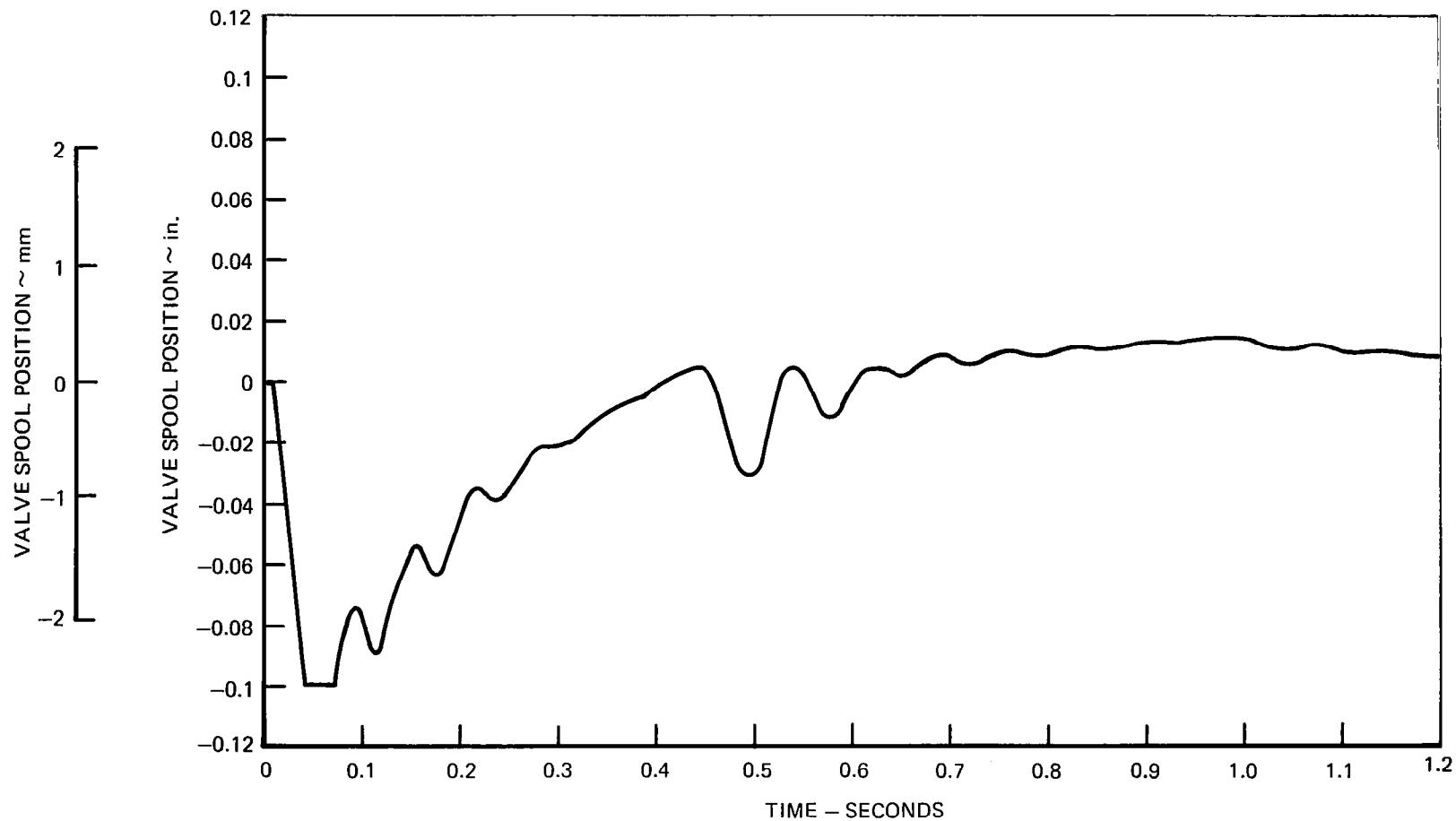


FIGURE 3-33 CASE 2 ACTIVE GEAR

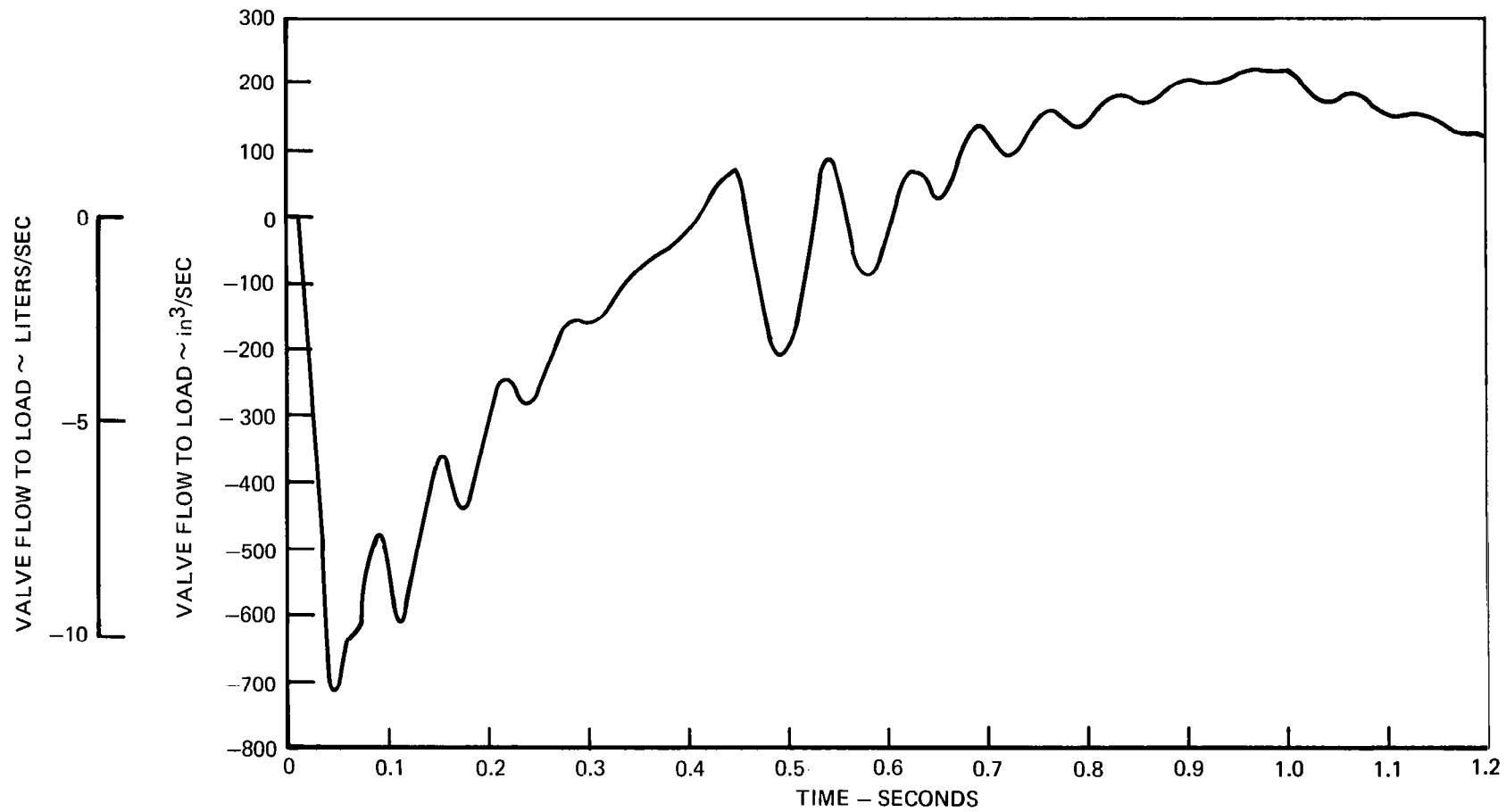


FIGURE 3-34 CASE 2 ACTIVE

### 3.6.3 Rollout Over Repaired Bomb Crater

Simulation of aircraft rollout over a repaired bomb crater (subsequent to an impact landing) was accomplished using the nonlinear vertical drop model. Initial conditions are calculated assuming the aircraft is in contact with the ground and the landing gear has reached an equilibrium condition in supporting the aircraft weight minus its lift. Assuming some horizontal speed for the aircraft, actual physical changes in ground level can be represented as transient changes which can be input into the nonlinear model. For this case a Class I repaired bomb crater was used. This was chosen because it was the worst-case profile out of all those supplied by NASA in support of this project. A diagram of the bomb crater is shown in Figure 3-35. The horizontal speed of the aircraft was assumed to be 51.8 m/sec (170 ft/sec). The command limit force is set to zero with a force deadband of  $\pm 8.9$  kN ( $\pm 2000$  lbf) throughout the transient, consistent with the assumption that the disturbance occurs during rollout, subsequent to an impact landing. The lift is set to 10 percent of the aircraft weight (per gear) throughout the transient. Figures 3-36 through 3-43 show the transient response of the various variables of interest for the passive and active gear simulations. Active control in this case reduces the peak wing/gear force 74 percent below the passive gear case. Note also from Figures 3-38 and 3-39 that the passive gear leaves the ground three separate times during the transient, while the active gear leaves the ground only once, very briefly.

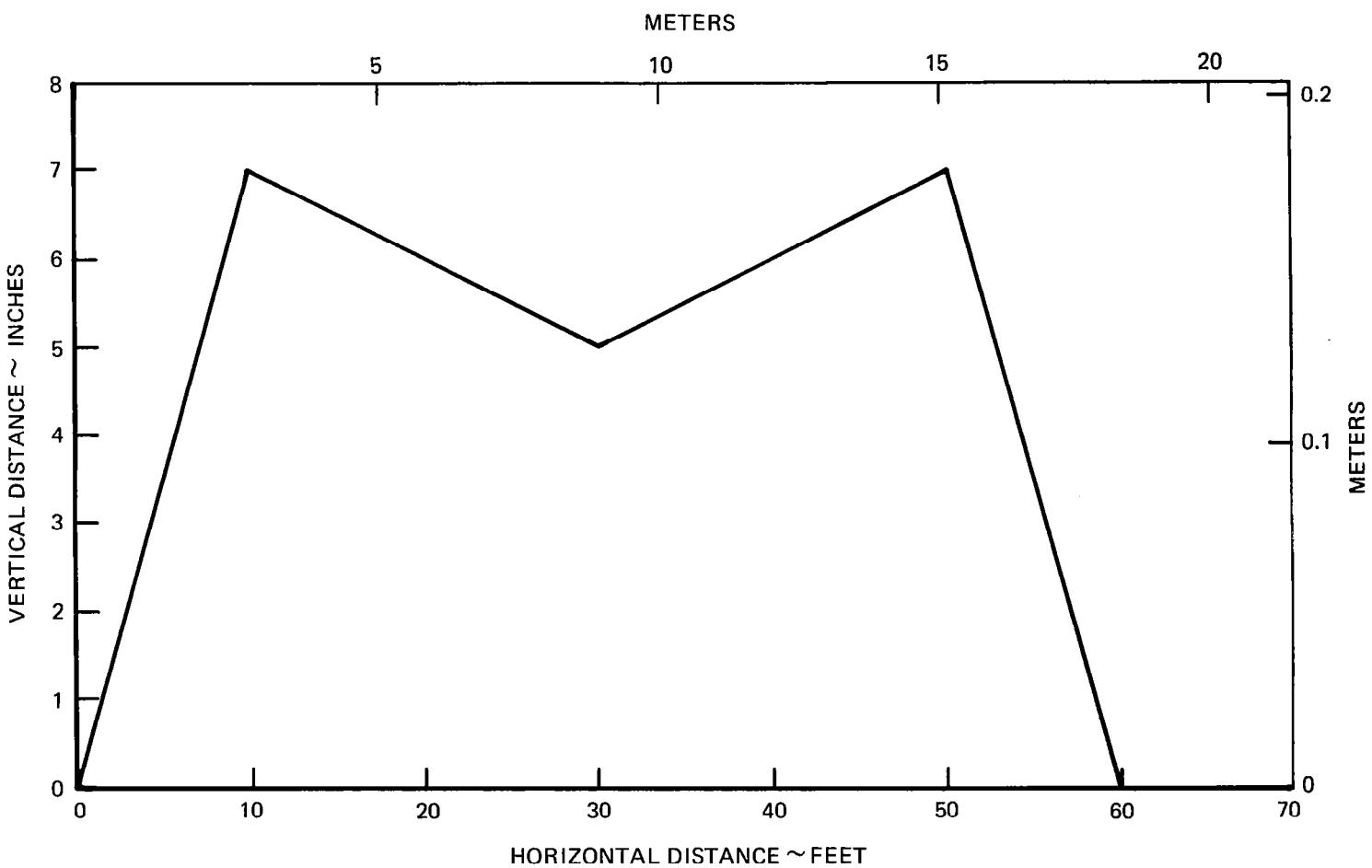


FIGURE 3-35. BOMB CRATER PROFILE

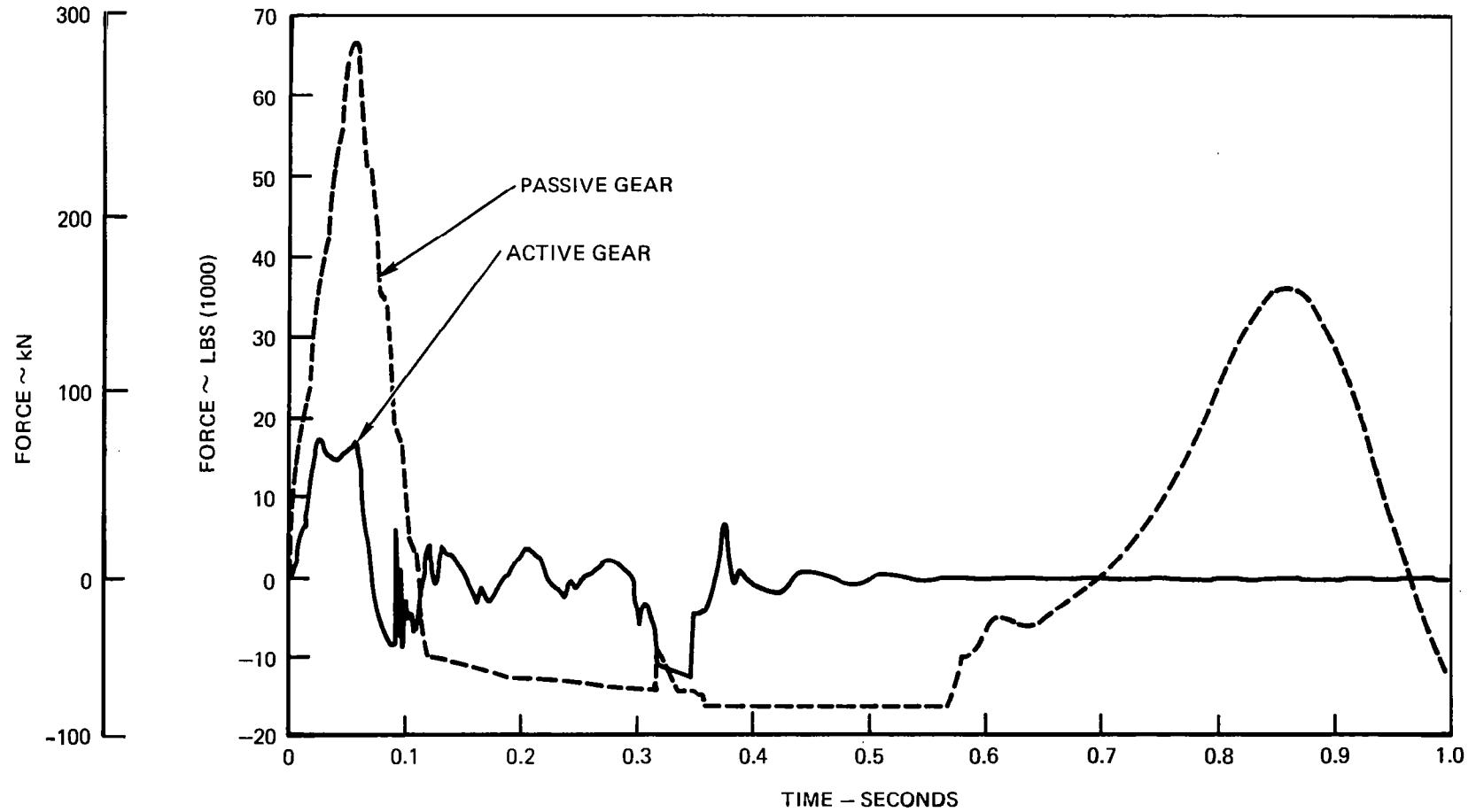


FIGURE 3-36 BOMB CRATER LANDING

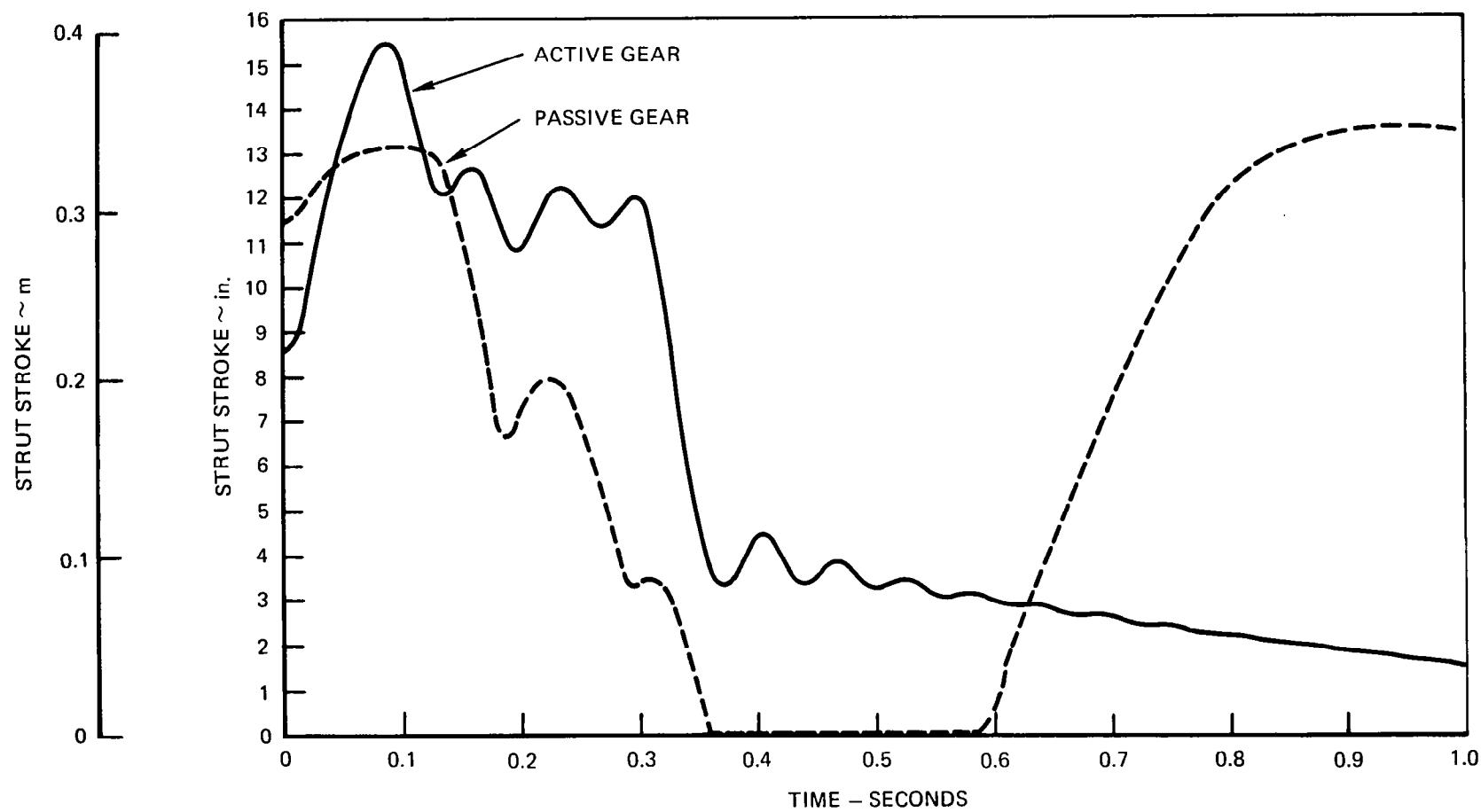


FIGURE 3-37 BOMB CRATER LANDING

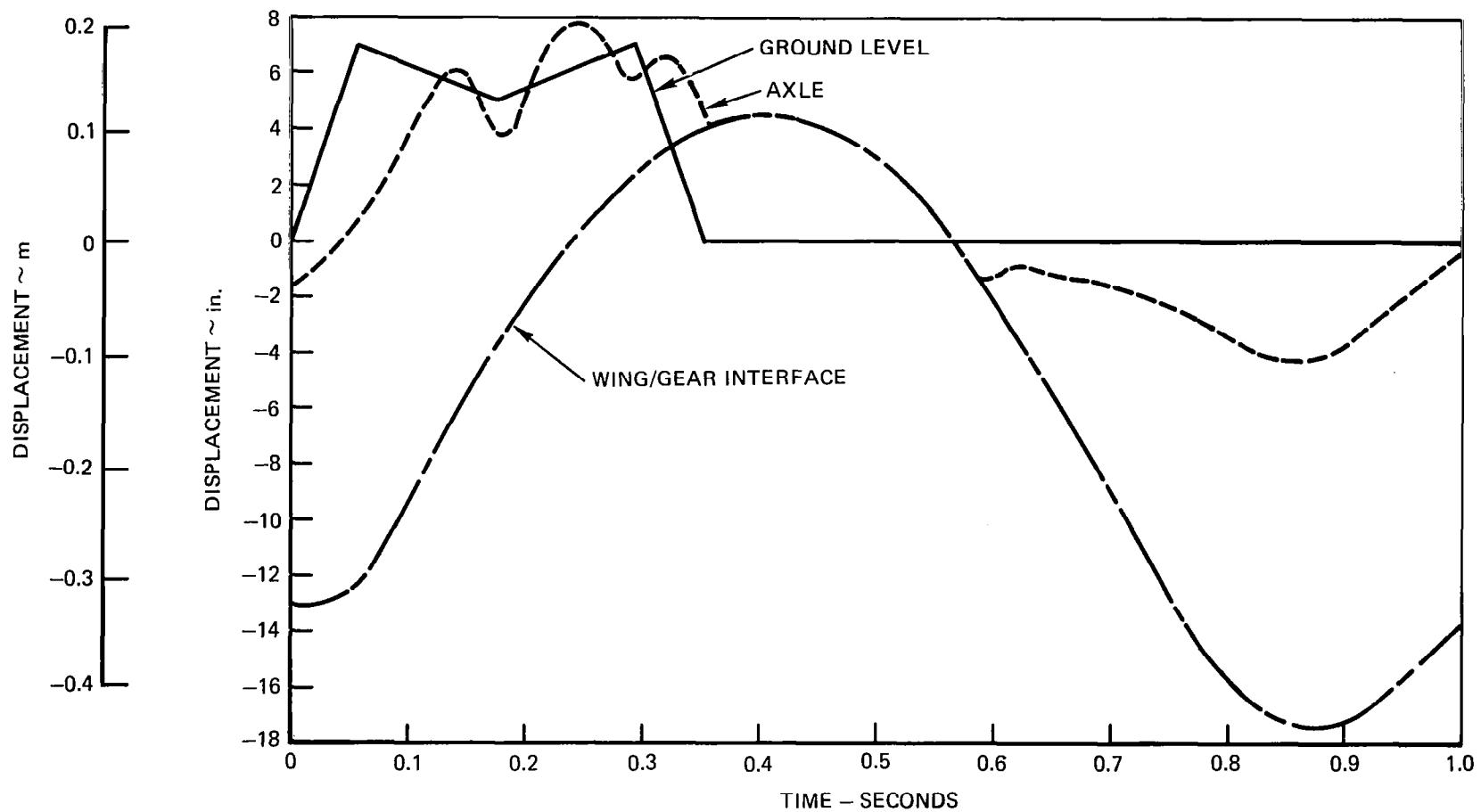


FIGURE 3-38 BOMB CRATER LANDING PASSIVE GEAR

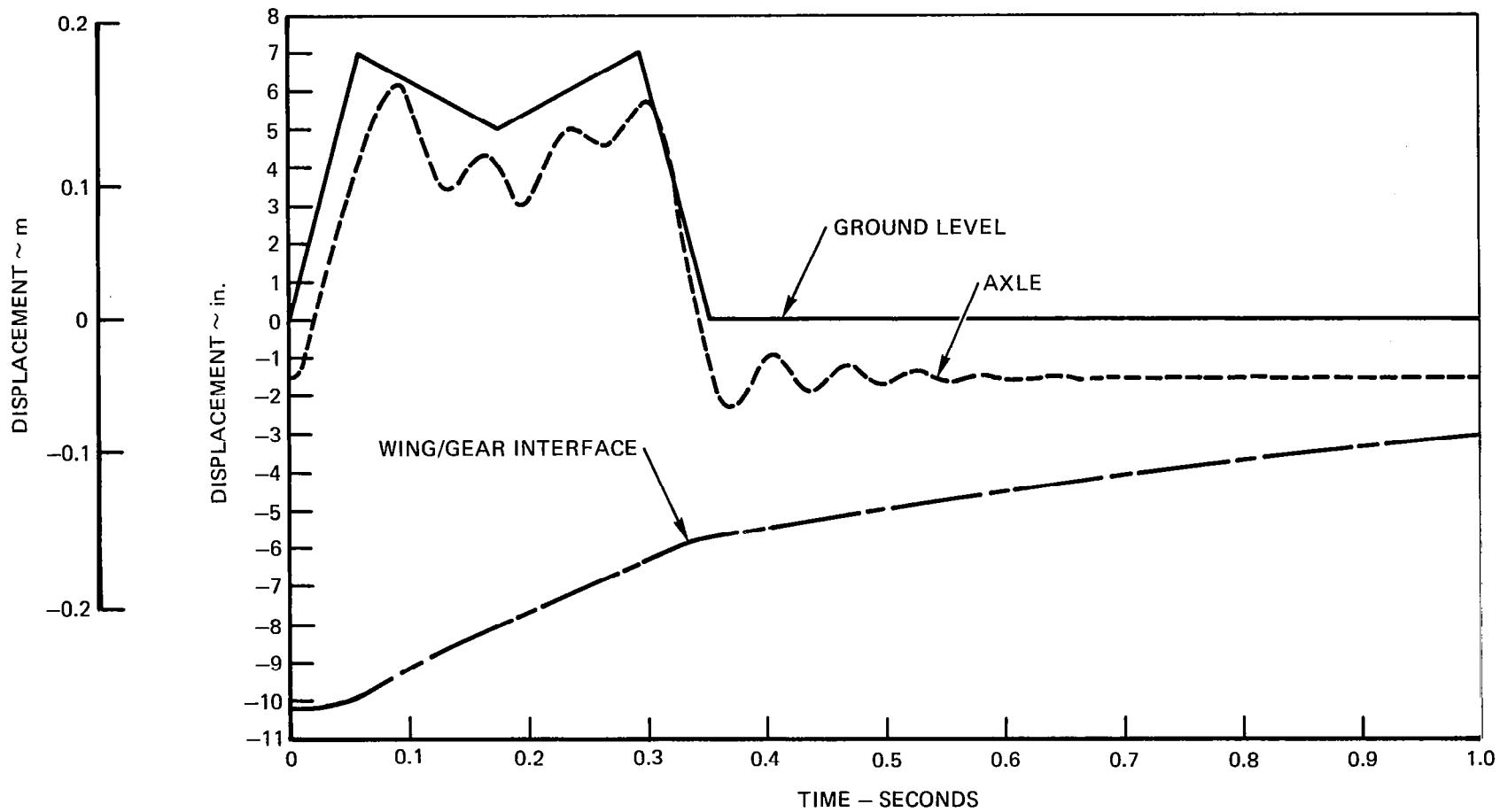


FIGURE 3-39 BOMB CRATER LANDING ACTIVE GEAR

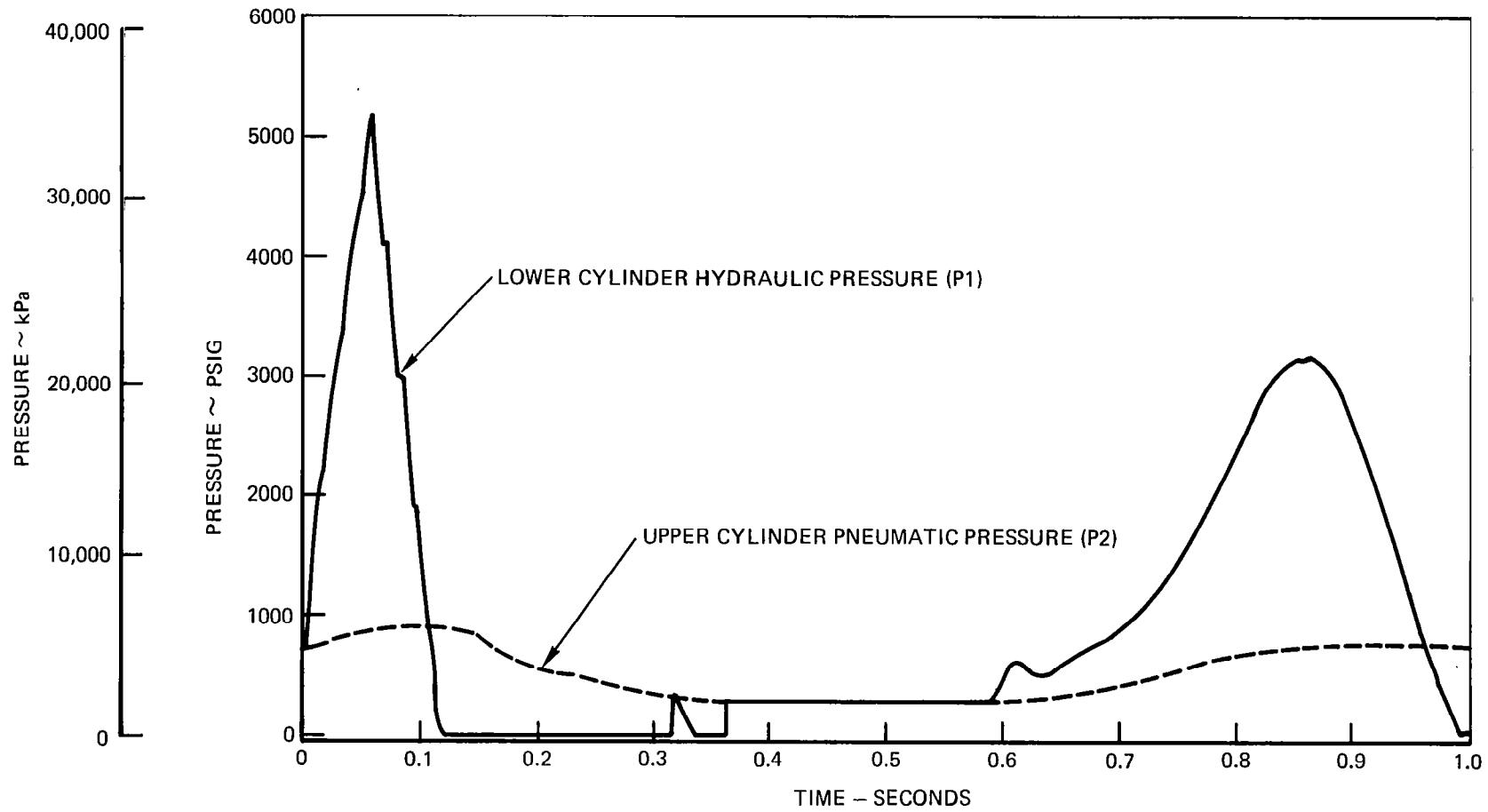


FIGURE 3-40 BOMB CRATER LANDING PASSIVE GEAR

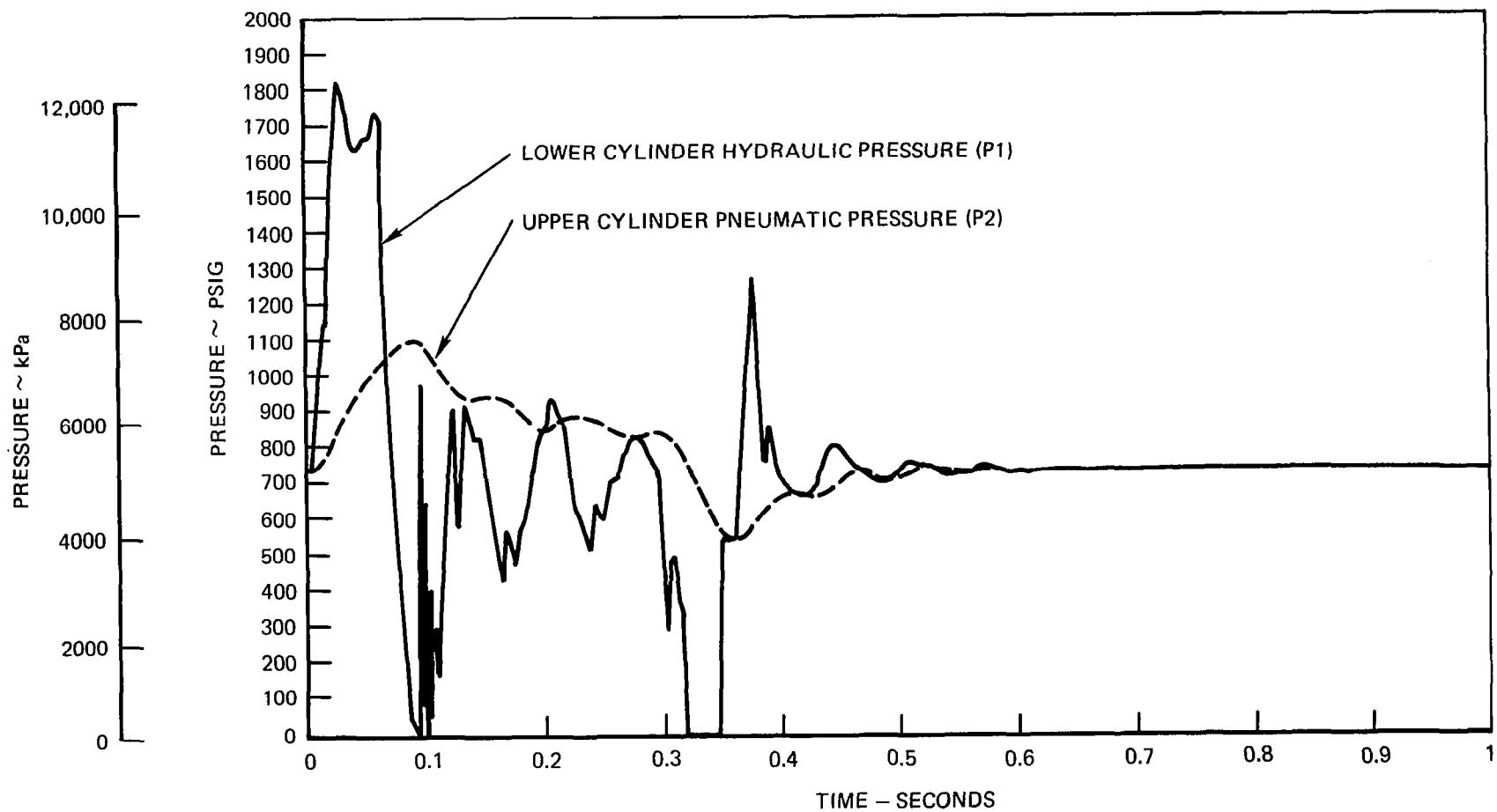
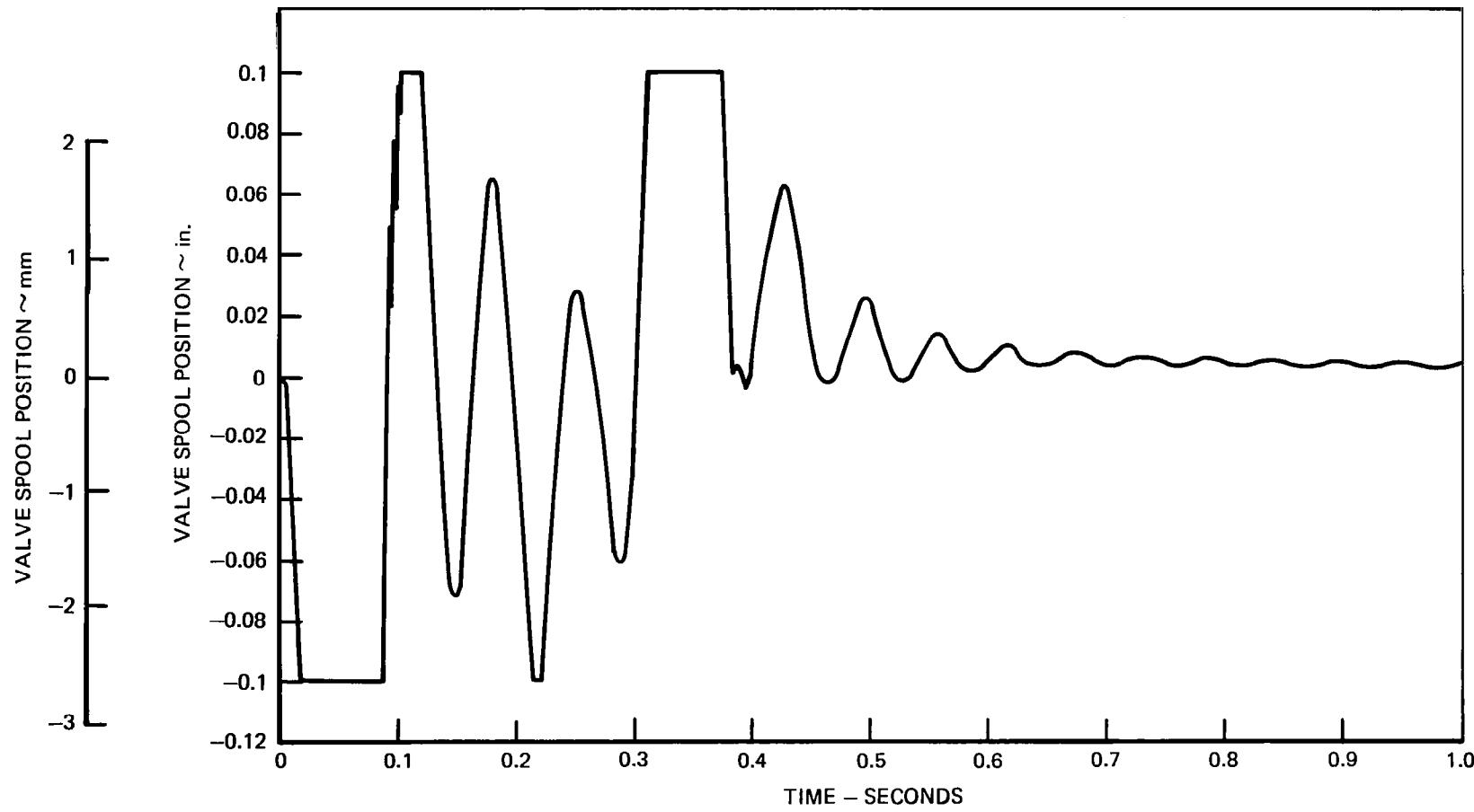


FIGURE 3-41 BOMB CRATER LANDING ACTIVE GEAR



55

FIGURE 3-42 BOMB CRATER LANDING ACTIVE GEAR

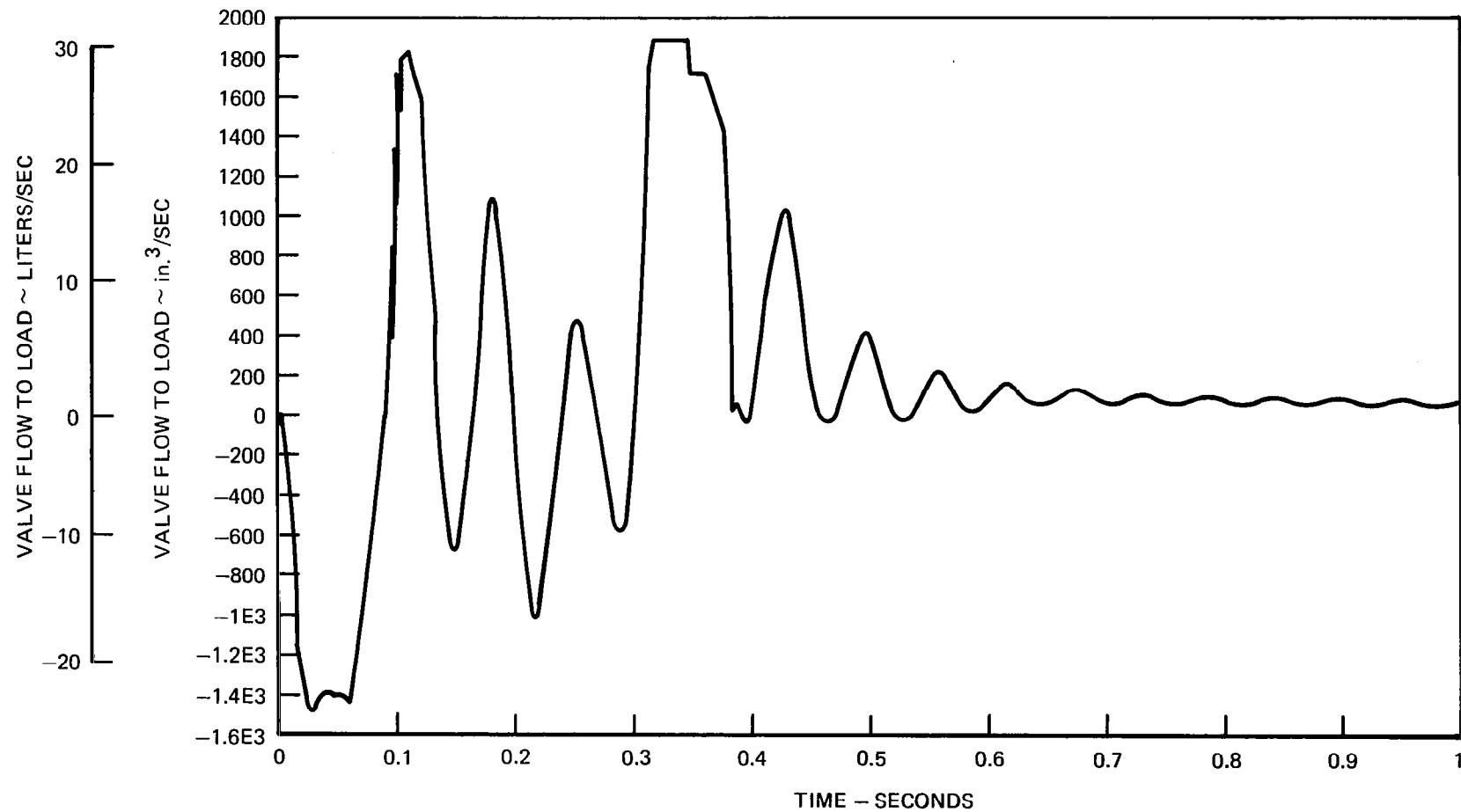
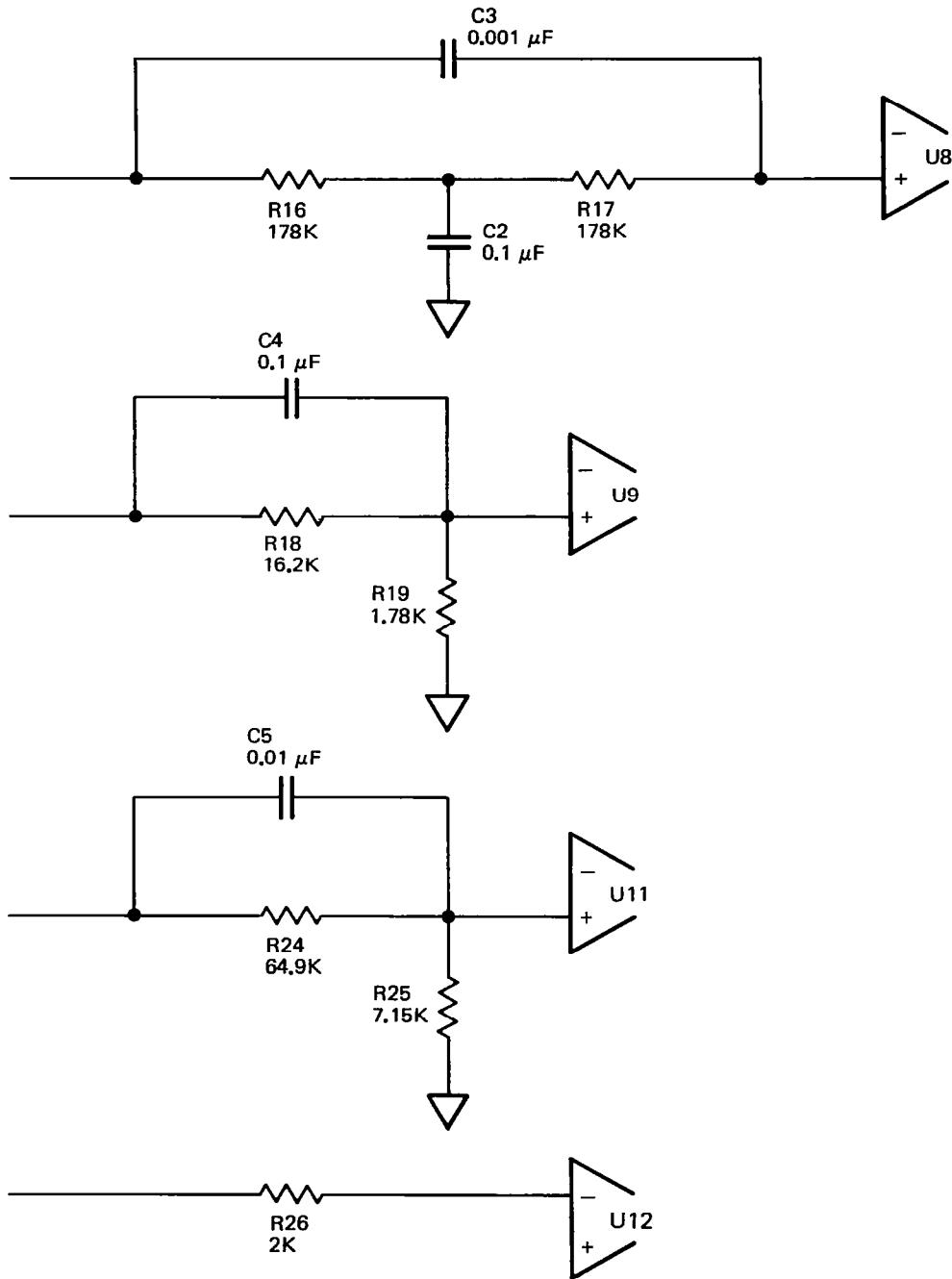


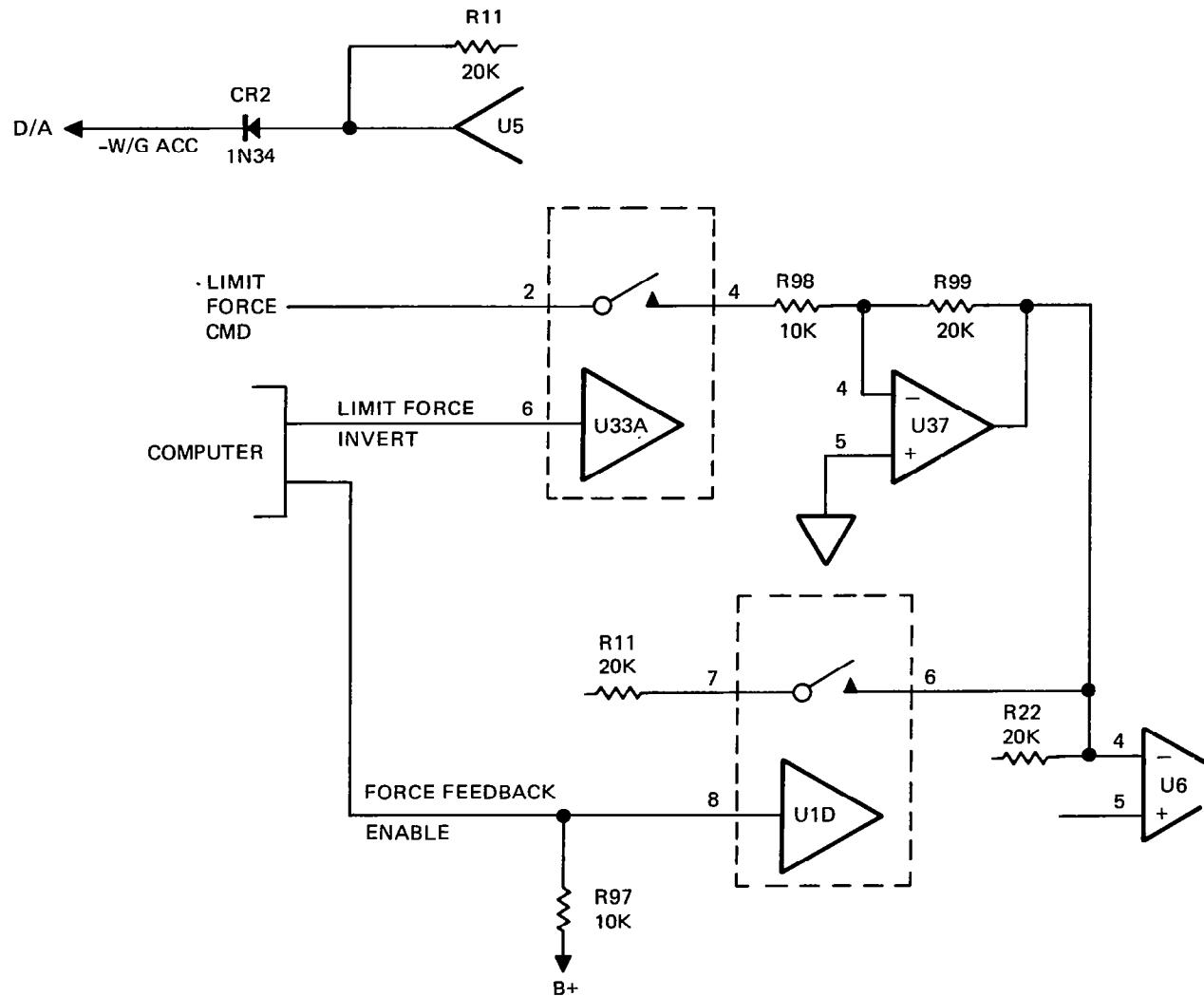
FIGURE 3-43 BOMB CRATER LANDING ACTIVE GEAR

#### 4.0 HARDWARE

The analog electronic hardware is the same as that designed during the Reference 1 investigation (HR drawing 88000080-201) except for the following circuit changes:



The original hardware mechanization of Reference 1 did not include provisions for a force deadband. These provisions were subsequently added and consist of the following:



## 5.0 SOFTWARE

The digital software of the system described in Reference 1 was modified to be compatible with the F-4 landing gear. Changes were incorporated to reflect the new scaling which was necessitated by the new weight and strut stroke. The scaling is discussed in detail in Appendix A.

In addition, program changes made by NASA were incorporated, including a force deadband which is effective in the takeoff mode as well as the landing mode.

The software flow chart is shown in Figure 5-1 and the complete program is listed in Appendix B.

## 6.0 CONCLUSION

An analysis has been made on the active control landing gear concept applied to the F-4 aircraft. Servocontrol loops and signal shaping have been defined. The results of the analysis show that the active control landing gear can significantly reduce the loads transmitted to the aircraft for both landing impact cases and rollout over ground level perturbations. For the vertical drop landing impact cases analyzed, reductions in wing/gear interface force of 20 to 22 percent were achieved from the conventional passive gear case. For the case of rollout over a repaired bomb crater, a reduction of 74 percent was achieved.

## 7.0 RECOMMENDATIONS

Based on the conclusion of this report it is recommended that the study be continued by investigating the following areas:

1. The benefits of the ACLG vs. any penalties involved such as cost, weight, and the effect on aircraft structure and hydraulic systems.
2. The possibility that under extremely uneven landing conditions the gear could be depleted of fluid and the effect of such depletion.
3. Requirements for landing at higher sink rates, i.e., 3.05 m/sec (10 ft/sec).
4. The design of a flightworthy ACLG for the F-4 aircraft.
5. The application of the results of this analysis to other aircraft systems.

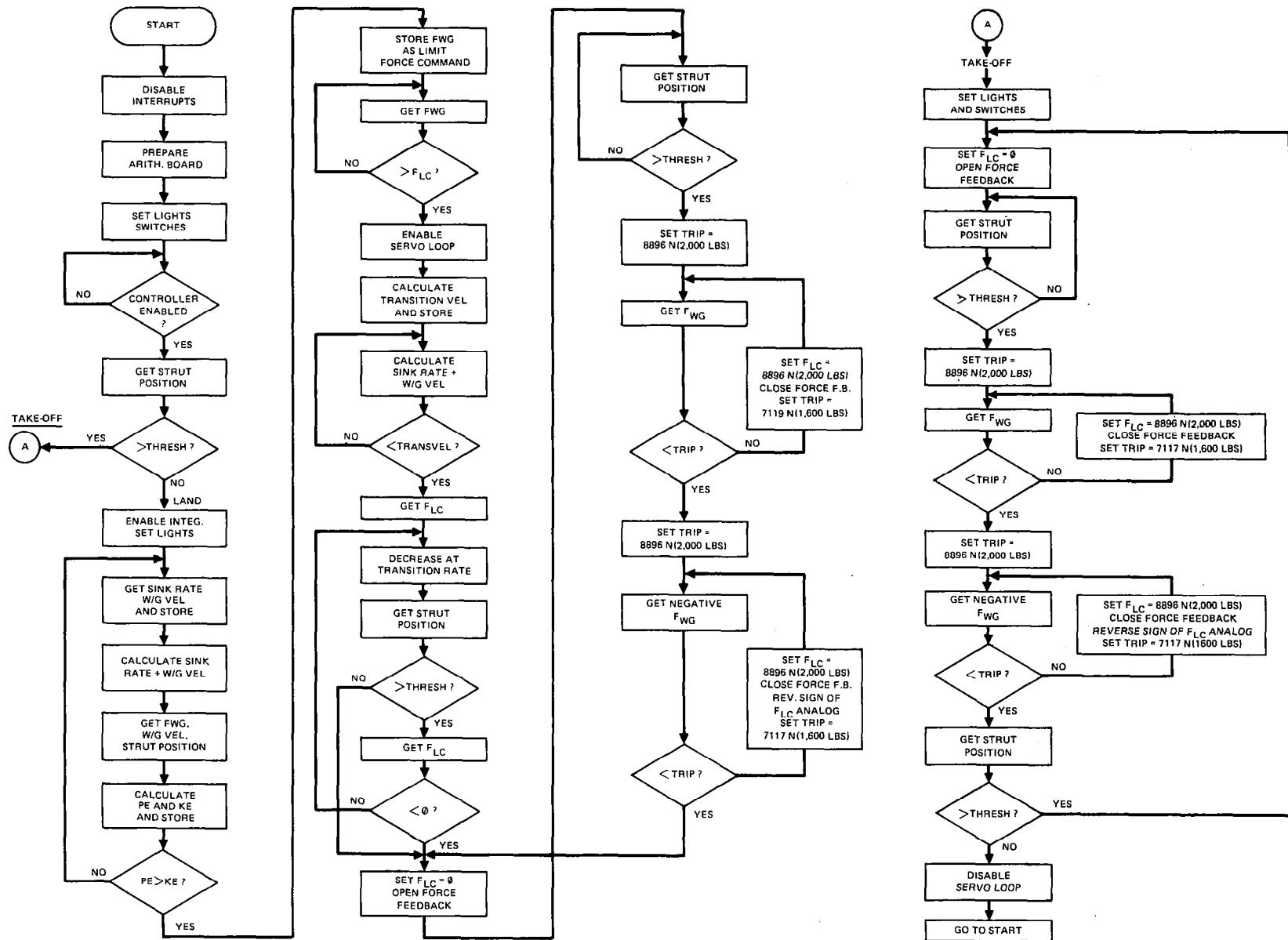


FIGURE 5-1 SOFTWARE FLOW CHART

## APPENDIX A

### DIGITAL SCALING

Since the stroke of the strut is 0.4034 m (15.88 in) it is anticipated that a (0.51m) potentiometer will be used to measure strut displacement.

It is further anticipated that the wing/gear interface accelerometer will be the same as that used in the system described in Reference 1.

Based on these assumptions the digital scaling is as follows:

#### (1) W/G acceleration:

The scale factor of the accelerometer is 2.85 v/g. The accelerometer signal is attenuated to 0.6316 of its value and then amplified by a factor of 6 in the analog circuitry to produce a scale factor of 10.8 v/g. Since  $10 \text{ v} = 4095 \text{ bits}$  the digital acceleration scale factor is:

$$10.8 \times 409.5 = 4423 \text{ bits/g.}$$

Since the aircraft weight is:

$$8.184 \times 10^4 \text{ N (18,398 lb)}$$

the scale factor in terms of force is:

$$0.05404 \text{ bits/N (0.2404 bits/lb)}$$

#### (2) W/G Velocity:

As stated above, the accelerometer scale factor is 2.85 v/g

$$\text{or } \frac{2.85v}{9.807\text{m/sec}^2} \quad \text{or: } 0.2907 \text{ v/m/sec}^2 (0.00738 \text{ v/in/sec}^2)$$

In the analog circuitry this signal is amplified by a factor of  $(0.6316)(21.47) = 13.56$

and integrated to produce w/g velocity. The velocity scale factor is then:

$$(0.2907)(13.56) = 3.94 \text{ v/m/sec / (0.1 v/in/sec).}$$

Digitally the scale factor is:

$(3.94 \text{ v/m/sec})(409.5 \text{ bits/v}) = 1614 \text{ bits/m/sec (40.95 bits/in/sec)}$ .

(3) Sink rate

The sink rate is scaled at

$3.94 \text{ v/m/sec (0.1 v/in/sec)}$

or digitally at

$1614 \text{ bits/m/sec (40.95 bits/in/sec)}$

to match the scaling of the w/g velocity signal.

(4) Strut displacement:

The strut potentiometer produces

$10 \text{ v for } 0.508 \text{ m (20 in).}$

Its scale factor is then

$19.96 \text{ v/m. (0.5 v/in).}$

The potentiometer signal is multiplied by 0.715 in the analog circuitry to produce a scale factor of:

$19.96 (0.715) = 14.08 \text{ v/m (0.3575 v/in).}$

Digitally the scale factor is

$(14.08 \text{ v/m} (409.5 \frac{\text{bits}}{\text{v}}) = 5766 \text{ bits/m (146.4 bits/in).}$

The maximum strut displacement is equivalent to:

$(0.4034 \text{ m})(5766 \text{ bits/m}) = 2325 \text{ bits}$

which is 0915 HEXADECIMAL (H).

(5) Work potential of the strut:

$$WP = Fwg (X_{max} - X_s) = MXwg (X_{max} - X_s)$$

If  $Xwg = 1 \text{ g}$  and  $X_{max} - X_s = 0.0254 \text{ m (1 in)}$  then

$$MXwg = 8.184 \times 10^4 \text{ N (18,398 lb.)}$$

WP is then  $(8.184 \times 10^4 \times 0.0254) = 2079 \text{ N-m (18,398 lbf ins)}$

As pointed out in (1) and (4) above, 1 g is equivalent to:

10.8 v and 0.0254 m (1 in) is equivalent to 0.3575 v.

Digitally then,

$$WP = (10.8 v)(409.5 \frac{\text{bits}}{v})(0.3575 v)(409.5 \frac{\text{bits}}{v}) = 6.4745 \times 10^5 \text{ bits.}$$

The scale factor of WP is therefore:

$$\frac{6.4745 \times 10^5}{2079} = 3.114 \times 10^2 \text{ bits/N-m} \quad (35.19 \text{ bits/lbf.in})$$

(6) Kinetic Energy:

$$KE = \frac{1}{2} \frac{W}{g} (V_{\text{tot}})^2 \text{ where } V_{\text{tot}} = V_{\text{touchdown}} + \int_0^\tau x_{wgdt}$$

If  $V_{\text{tot}} = 0.0254 \text{ m/sec}$  (1 in/sec) then

$$KE = \frac{1}{2} \frac{(8.184 \times 10^4 \text{ N})}{(9.807 \text{ m/sec}^2)} (0.0254 \text{ m/sec})^2 = 2.692 \text{ Nm} \quad (23.83 \text{ lb in})$$

Digitally, from (2),

$$V_{\text{tot}} = (0.0254 \text{ m/sec})(3.94 \text{ v/m/sec})(409.5 \text{ bits/v}) = 40.95 \text{ bits.}$$

$$\text{Then } KE = (40.95)^2 = 1676.9 \text{ bits.}$$

Therefore the scale factor for KE is:

$$\frac{1676.9 \text{ bits}}{2.692 \text{ N-m}} = 622.9 \text{ bits/N-m} = (70.37 \text{ bits/lb in})$$

which is twice the scale factor of WP, from (5) above.

Therefore, to compare KE to WP it must be divided by 2. This is accomplished in the software by a right shift.

(7) Decrease of limit force command during transition:

10 v. corresponds to:

$$8.184 \times 10^4 \text{ N} \quad (18,398 \text{ lbs}_f) \text{ of } F_{LI},$$

so that the scale factor of  $F_{LI}$  is:

$$8184 \text{ N/v} \quad (1840 \text{ lbs}_f/\text{v.})$$

Digitally the scale factor is:

$$\frac{(8184 \text{ N/v})}{409.5 \text{ bits/v}} = 19.98 \frac{\text{N}}{\text{bit}} (4.49 \text{ lbf/bit}) = 0.05 \text{ bit/N} (0.22 \text{ bit/lbf})$$

During transition  $F_{1I}$  is decreased at a rate of

$$4.448 \times 10^5 \text{ N/sec} (10^5 \text{ lbf/sec})$$

or digitally at a rate of

$$2.226 \times 10^4 \text{ bits/sec.}$$

(8) Transition velocity (Vt):

$$V_t = \frac{F_{1I}^2}{2(W/g)R}$$

$$W = 8.178 \times 10^4 \text{ N} (18,398 \text{ lb}) \text{ and } R = 4.445 \times 10^5 \text{ (10}^5 \text{ lb/sec)}$$

then:

$$V_t = \frac{F_{1I}^2}{2(9.178 \times 10^4 / 9.8)(4.445)10^5} = 1.348 \times 10^{-10} F_{1I}^2$$

Therefore:

4.445 N (1 lb) produces:

$$2.663 \times 10^{-9} \text{ m/sec} (1.049 \times 10^{-7} \text{ in/sec})$$

From (7) the digital scale factor of  $F_{1I}$  is  $0.05 \frac{\text{bit}}{\text{N}}$

so that the digital signal produced is:

$$(4.445 \times 0.05)^2 = 0.0494 \text{ bits.}$$

The scale factor of  $V_t$  is therefore:

$$\frac{0.0494}{2.663 \times 10^{-9}} = 1.86 \times 10^7 \text{ bits/m/sec} (4.724 \times 10^5 \text{ bits/in/sec})$$

From (2) and (3) the scale factor for Vtot is:

1614 bits/m/sec (40.95 bits/in/sec)

Therefore, in order to compare Vtot to Vt, Vt must be multiplied

$$\frac{1614}{1.885 \times 10^7} = 0.00008669$$

in the arithmetic board which is accomplished as follows:

0.0000869 DECIMAL (D)=

0.000000000000101101011100110101010110110000001 Binary (B)

which equals:

1.01101011100110101010110  $\times 2^{-14}$  (B)

The exponent is -14(D)

The bias in the arithmetic board is:

07F HEXIDECLIMAL (H) or 127 (D)

Therefore the number must be applied with a bias of

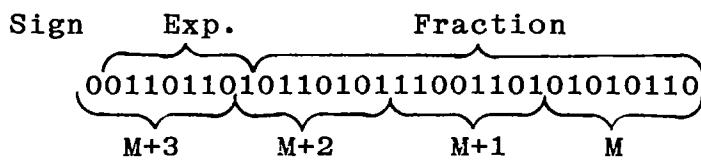
127 -14 = 113 (D).

In addition, a factor must be applied due to the fact that the numbers from the A/D converter are stored in the most significant 12 bits of the 16 so that the number for Vtot is too high by a factor of 16. Vt is a function of FLI<sup>2</sup> and is too high by a factor of (16)<sup>2</sup>. The net result is that Vt is too high by a factor of 16. It must therefore be reduced by a factor of 16 or 2<sup>4</sup>.

Therefore the exponent of the applied number is:

113 - 4 = 109(D) = 6D (H) = 01101101 (B)

A sign bit ("0" for positive) must precede the exponent. The format of the applied number is:



Therefore, if this is stored in memory starting at address M, the contents of memory are:

M	36(H)
M+1	B5(H)
M+2	CD(H)
M+3	56(H)

## APPENDIX B

### MICROPROCESSOR PROGRAM

The microprocessor program is listed in the following pages. It should be pointed out that for three-byte instructions, the listing of the last two bytes is in an order which is reversed from the order in which the bytes are stored in memory. This is a peculiarity of the assembler which was used.

ASSEMBLED AT 0000

MACRO-80 3.34 04-NOV-79 PAGE 1

```
00001 ; ****
00002 ; PROGRAM FOR AN ELECTROHYDRAULIC ACTIVE
00003 ; CONTROL AIRCRAFT LANDING GEAR
00004 ; ****
00005 ; NOTES:
00006 ; RAM LOCATION 3DOEH ADDED AS A TEMP LOC
00007 ; FOR FLIM 2/13/80
00008 ; ****
00009 ; REVISED FOR F4 GEAR 6/28/81
00010 ORG 00
0000' F3 00011 START: DI ;DISABLE INTERRUPTS
0001' 21 3FFF 00012 LXI H,3FFFH ;INIT. STACK
0004' F9 00013 SPHL
0005' 3E 82 00014 MVI A,82H ;INIT. MATH BOARD
0007' D3 EB 00015 OUT 0EBH
0009' 3E 00 00016 MVI A,00 ;SET MEM. BASE ADD.
000B' D3 A1 00017 OUT 0A1H
000D' 3E 80 00018 MVI A,80H
000F' D3 A2 00019 OUT 0A2H
0011' 21 0092 00020 LXI H,0092H ;STRUT THRESH.=1 IN.
00021 ;MULT THRESH BY 16
0014' CD 0289' 00022 CALL ANM ;FOR LATER USE
0017' 22 3F8A 00023 SHLD 3F8AH ;LABEL BXTHR
001A' 3E 4C 00024 MVI A,4CH ;SET LIGHTS,SWITCHES
001C' D3 EA 00025 OUT 0EAH
00026 ; ****
00027 ; OUTPUT MUXO-MUX4 FOR A/D BOARD CHECK
00028 ; ALSO LOOK FOR CONTROLLER ENABLED
00029 ; ****
001E' 16 03 00030 TL1: MVI D,3
0020' 06 FF 00031 TL2: MVI B,0FFH
0022' 0E 05 00032 MVI C,5
0024' 7A 00033 TL3: MOV A,D
0025' CD 0251' 00034 CALL IN1
0028' 22 F708 00035 SHLD 0F708H ;OUTPUT TO DACO
002B' DB E9 00036 IN 0E9H ;CONTROLLER ENABLED?
002D' 1F 00037 RAR
002E' DA 0042' 00038 JC L1 ;YES, JUMP TO L1
0031' 05 00039 DCR B
0032' C2 0024' 00040 JNZ TL3
0035' 06 FF 00041 MVI B,0FFH
0037' OD 00042 DCR C
0038' C2 0024' 00043 JNZ TL3
003B' 15 00044 DCR D
003C' FA 001E' 00045 JM TL1
003F' C3 0020' 00046 JMP TL2
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0042' 3E 02          00047 ;*****
0044' CD 0251'        00048 ; CONTROLLER HAS BEEN ENABLED
0047' 2A 3F8A         00049 ;*****
004A' EB              00050 L1:    MVI   A,02
004B' 2A 3F86         00051 CALL  IN1      ;GET STRUT POS FOR LAND/TO. DEC.
004E' CD 0282'         00052 LHLD  3F8AH   ;GET STRUT THRESHOLD
0051' DA 01BD'         00053 XCHG
0054' 3E 03          00054 LHLD  3F86H   ;LOAD HL WITH STRUT POSITION
0056' CD 0251'         00055 CALL  SUB2    ;CALC. THRESHOLD-STRUT
0059' 22 3F88         00056 JC    L12A    ;YES, JUMP TO 12A
005C' 21 0915         00057 ;*****
005F' CD 0289'         00058 ; LANDING - MAKE PREPARATIONS
0062' 22 3F8C         00059 ;*****
0064' 3E 9E          00060 MVI   A,03
0066' D3 EA          00061 CALL  IN1      ;GET SINK RATE
0069' CD 0220'         00062 SHLD  3F88H   ;STORE IT
0071' EB              00063 LXI   H,0915H ;MULT XMAX BY 16 TO SHIFT INTO
0072' 95              00064 CALL  ANM     ;UPPER 12 BITS
0073' 6F              00065 SHLD  3F8CH   ;STORE IT
0074' 7A              00066 ;*****
0075' 9C              00067 ; ENABLE INTEGRATOR
0076' 67              00068 ; START ENERGY CALCULATIONS
0077' 22 8004         00069 ;*****
007A' AF              00070 MVI   A,9EH
007B' CD 028E'         00071 OUT   OEAH    ;ENABLE INTEGRATOR
007E' 2A 8000         00072 L8:    CALL  IN3
0081' 22 3F8E         00073 XCHG
0084' 2A 8002         00074 LHLD  3F8CH
0087' 22 3F90         00075 XCHG
008A' 2A 3F88         00076 MOV   A,E
008D' EB              00077 SUB   L
008E' 2A 3F84         00078 MOV   L,A
0091' CD 0282'         00079 MOV   A,D
0092'                 00080 SBB   H
0093'                 00081 MOV   H,A
0094'                 00082 SHLD  8004H
0095'                 00083 XRA   A
0096'                 00084 CALL  MATH
0097'                 00085 LHLD  8000H
0098'                 00086 SHLD  3F8EH
0099'                 00087 LHLD  8002H
0100'                 00088 SHLD  3F90H
0101'                 00089 LHLD  3F88H
0102'                 00090 XCHG
0103'                 00091 LHLD  3F84H
0104'                 00092 CALL  SUB2

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0094'	22 8000	00093	SHLD	8000H
0097'	22 8004	00094	SHLD	8004H
009A'	AF	00095	XRA	A
009B'	CD 028E'	00096	CALL	MATH
		00097	*****	
		00098	DIVIDE KE BY 2 THEN DO A BYTE BY BYTE COMPARE	
		00099	TO TEST IF PE)KE. DON'T BOTHER TO TEST LSB-IT	
		00100	CONTAINS NO USEFUL DATA	
		00101	*****	
009E'	B7	00102	ORA	A ;CLEAR CARRY
009F'	06 03	00103	MVI	B,3 ;SET A BYTE COUNTER
00A1'	21 8003	00104	LXI	H,8003H ;GET KINETIC ENERGY
00A4'	7E	00105	L8A:	MOV A,M ;SHIFT RIGHT 3 BYTES AND
00A5'	1F	00106	RAR	;RE-SAVE
00A6'	77	00107	MOV	M,A
00A7'	2B	00108	DCX	H
00A8'	05	00109	DCR	B
00A9'	C2 00A4'	00110	JNZ	L8A
00AC'	06 03	00111	MVI	B,03
00AE'	21 3F91	00112	LXI	H,3F91H
00B1'	11 8003	00113	LXI	D,8003H
00B4'	1A	00114	L9:	LDAX D
00B5'	BE	00115	CMP	M
00B6'	C2 00C2'	00116	JNZ	L10
00B9'	1B	00117	DCX	D
00BA'	2B	00118	DCX	H
00BB'	05	00119	DCR	B
00BC'	C2 00B4'	00120	JNZ	L9
00BF'	C3 00C5'	00121	JMP	L11
00C2'	D2 0069'	00122	L10:	JNC L8
		00123	*****	
		00124	TIME TO INITIATE ACTIVE CONTROL	
		00125	*****	
00C5'	2A 3F80	00126	L11:	LHLD 3F80H
00C8'	22 F70A	00127	SHLD	OF70AH
00CB'	22 3D0E	00128	SHLD	3DOEH ;SAVE ORIGINAL "FLIM"
00CE'	EB	00129	XCHG	3DOEH ;FLIM TO DE FOR COMPARE
00CF'	CD 0220'	00130	CHACEL:	CALL IN3 ;GET W/G ACCEL.
00D2'	2A 3F80	00131	LHLD	3F80H ;PUT NEW ACCEL INTO HL
00D5'	CD 0282'	00132	CALL	SUB2 ;DE-HL IS NEW W/G ACCEL.
		00133		;GREATER THAN FLIM?
00D8'	F2 00CF'	00134	JP	CHACEL ;NO-LOOP TILL IT IS
(00DB-00E0) = 00 (NOP)		00135	:CALL	SPTH ;HAS GEAR STARTED STROKE?
		00136	;JP	CHACEL ;LOOP TILL GEAR ) THRESHOLD
00E1'	3E 9F	00137	MVI	A,9FH
00E3'	D3 EA	00138	OUT	OEAH ;ENABLE SERVOLOOP

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00139 ; ****
00140 ;      GEAR IS NOW UNDER ACTIVE CONTROL
00141 ; ****
00E5' 2A 3DOE   00142 LHLD  3DOEH ; GET ORIGINAL FLIM TO CALCULATE
00E8' 22 8000   00143 SHLD  8000H ; TRANSITION VELOCITY
00EB' 21 0000   00144 LXI   H,0000
00EE' 22 8002   00145 SHLD  8002H
00F1' 3E 08     00146 MVI   A,08
00F3' CD 028E'  00147 CALL   MATH
00F6' 3E 06     00148 MVI   A,06
00F8' CD 028E'  00149 CALL   MATH
00FB' 3E 36     00150 MVI   A,36H
00FD' 32 8004   00151 STA   8004H
0100' 3E E5     00152 MVI   A,OB5H
0102' 32 8005   00153 STA   8005H
0105' 3E CD     00154 MVI   A, OCDH
0107' 32 8006   00155 STA   8006H
010A' 3E 56     00156 MVI   A,56H
010C' 32 8007   00157 STA   8007H
010F' 3E 02     00158 MVI   A,02
0111' CD 028E'  00159 CALL   MATH
0114' 2A 8000   00160 LHLD  8000H
0117' 22 3F92   00161 SHLD  3F92H
011A' 2A 8002   00162 LHLD  8002H
011D' 22 3F94   00163 SHLD  3F94H
00164 ; ****
00165 ;      TRANSITION VELOCITY STORED AS FLOATING PT, 32 BIT
00166 ;      NUMBER.  START COMPARING THIS AGAINST (SINK RATE
00167 ;      -W/G VEL.) TO DETERMINE START OF TRANSITION
00168 ; ****
0120' 2A 3F88   00169 L4:   LHLD  3F88H
0123' EB        00170 XCHG
0124' 3E 00     00171 MVI   A,00
0126' CD 0251'  00172 CALL   IN1
0129' CD 0282'  00173 CALL   SUB2
012C' 22 8000   00174 SHLD  8000H
012F' 21 0000   00175 LXI   H,0000
0132' 22 8002   00176 SHLD  8002H
0135' 3E 08     00177 MVI   A,08
0137' CD 028E'  00178 CALL   MATH
013A' 2A 3F92   00179 LHLD  3F92H
013D' 22 8004   00180 SHLD  8004H
0140' 2A 3F94   00181 LHLD  3F94H
0143' 22 8006   00182 SHLD  8006H
0146' 3E 0A     00183 MVI   A,0AH
0148' CD 028E'  00184 CALL   MATH

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01B3' 3E 9D          00231 L14: MVI   A,9DH
01B5' D3 EA          00232 OUT   OEAH    ;CLOSE FORCE FEEDBACK
01B7' 01 1800         00233 LXI   B,1800H ;SET BC TO 1600 LBS
01BA' C3 017F'        00234 JMP   L15
00235 ;*****
00236 ; TAKEOFF MODE
00237 ;*****
01BD' 21 0000         00238 L12A: LXI   H,0000
01CO' 22 F70A         00239 SHLD  OF70AH ;SET LIMIT FORCE CMD='0 LBS.
01C3' 3E A9          00240 MVI   A,0A9H
01C5' D3 EA          00241 OUT   OEAH    ;ENABLE SERVOLOOP & OPEN F FDBK
01C7' CD 020E'        00242 FLOA: CALL  SPTH   ;CHECK STRUT POSITION
01CA' F2 01C7'        00243 JP    FLOA
01CD' 01 1EE0         00244 LXI   B,1EE0H ;SET BC TO 2000 LBS
01D0' CD 0220'        00245 L15A: CALL  IN3
01D3' 2A 8000         00246 LHLD  8000H
01D6' 22 F708         00247 SHLD  OF708H ;OUTPUT W/G ACCEL TO DACO
01D9' CD 027D'        00248 CALL   FTEST
01DC' D2 01FE'        00249 JNC   L13A
01DF' 01 1EE0         00250 LXI   B,1EE0H ;SET BC TO 2000 LBS
01E2' CD 0266'        00251 L16A: CALL  IN4
01E5' 22 F708         00252 SHLD  OF708H ;OUTPUT W/G ACCEL TO DACO
01E8' CD 027D'        00253 CALL   FTEST
01EB' DA 01BD'        00254 JC    L12A
01EE' 21 1EE0         00255 LXI   H,1EE0H ;SET HL TO 2000 LBS
01F1' 3E A5          00256 MVI   A,0A5H ;CLOSE F FDBK & REVERSE SIGN
01F3' D3 EA          00257 OUT   OEAH    ;OF LIMIT FORCE CMD (ANALOG)
01F5' 22 F70A         00258 SHLD  OF70AH ;SET LIMIT FORCE CMD=2000 LBS
01F8' 01 1800         00259 LXI   B,1800H ;SET BC TO 1600 LBS
01FB' C3 01E2'        00260 JMP   L16A
01FE' 21 1EE0         00261 L13A: LXI   H,1EE0H ;SET HL TO 2000 LBS
0201' 22 F70A         00262 SHLD  OF70AH ;SET LIMIT FORCE CMD=2000 LBS
0204' 3E AD          00263 L14A: MVI   A,0ADH
0206' D3 EA          00264 OUT   OEAH    ;CLOSE FORCE FEEDBACK
0208' 01 1800         00265 LXI   B,1800H ;SET BC TO 1600 LBS
020B' C3 01D0'        00266 JMP   L15A
00267 ;*****
00268 ; ROUTINE TO SUBTRACT STRUT POS'N FROM THRESHOLD
00269 ;*****
020E' E5              00270 SPTH: PUSH  H      ;SPTH SETS SIGN FLAG POSITIVE
020F' D5              00271 PUSH  D      ;UNTIL STRUT POS'N ) THRESHOLD
0210' CD 024F'        00272 CALL  STP     ;GET STRUT POSITION
0213' 21 0B20          00273 LXI   H,0B20H ;THRESHOLD 0160H=.05"
00274 ;                02DOH=.1"
00275 ;                0590H=.2"
00276 ;                0B20H=.4"

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014B'	DB A1	00185	IN	0A1H
014D'	E6 20	00186	ANI	20H
014F'	CA 0120'	00187	JZ	L4
		00188	*****	
		00189	; TRANSITION PHASE	
		00190	*****	
0152'	2A 3DOE	00191	LHLD	3DOEH ;FLIM
0155'	11 FFFE	00192	LXI	D, OFFFEH
0158'	22 F70A	00193	L5:	SHLD OF70AH
015B'	19	00194	DAD	D
015C'	CD 020E'	00195	EXST:	CALL SPTH ;CHECK (THRESH.-STRUT POS'N)
015F'	F2 016C'	00196	JP	L12
0162'	01 1BDO	00197	LXI	B, 1BDOH
0165'	7D	00198	MOV	A,L
0166'	91	00199	SUB	C
0167'	7C	00200	MOV	A,H
0168'	98	00201	SBB	B
0169'	D2 0158'	00202	JNC	L5
		00203	*****	
		00204	; ROLLOUT PHASE	
		00205	*****	
016C'	21 0000	00206	L12:	LXI H,0000
016F'	22 F70A	00207	SHLD	OF70AH ;SET FLC=0 LBS.
0172'	3E 99	00208	MVI	A,99H
0174'	D3 EA	00209	OUT	OEAH ;OPEN FORCE FEEDBACK
0176'	CD 020E'	00210	FLO:	CALL SPTH ;CHECK STRUT POS'N.
0179'	F2 0176'	00211	JP	FLO
017C'	01 1EE0	00212	LXI	B,1EEOH ;SET BC TO 2000 LBS.
017F'	CD 0220'	00213	L15:	CALL IN3
0182'	2A 8000	00214	LHLD	8000H
0185'	22 F708	00215	SHLD	OF708H ;OUTPUT W/G ACCEL. TO DACO
0188'	CD 027D'	00216	CALL	FTEST
018B'	D2 01AD'	00217	JNC	L13
018E'	01 1EE0	00218	LXI	B,1EEOH ;SET BC TO 2000 LBS.
0191'	CD 0266'	00219	L16:	CALL IN4
0194'	22 F708	00220	SHLD	OF708H ;OUTPUT W/G ACCEL TO DACO
0197'	CD 027D'	00221	CALL	FTEST
019A'	DA 016C'	00222	JC	L12
019D'	21 1EE0	00223	LXI	H,1EEOH ;SET HL TO 2000 LBS.
01A0'	3E 95	00224	MVI	A,95H ;CLOSE F FDBK & REV. SIGN
01A2'	D3 EA	00225	OUT	OEAH ;OF LIMIT FORCE CMD (ANALOG)
01A4'	22 F70A	00226	SHLD	OF70AH ;SET LIMIT FORCE CMD=2000 LBS
01A7'	01 1800	00227	LXI	B,1800H ;SET BC TO 1600 LBS
01AA'	C3 0191'	00228	JMP	L16
01AD'	21 1EE0	00229	L13:	LXI H,1EEOH ;SET HL TO 2000 LBS
01BO'	22 F70A	00230	SHLD	OF70AH ;SET LIMIT FORCE CMD=2000 LBS

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0216'	EB	00277	XCHG	
0217'	2A 3F86	00278	LHLD	3F86H ;GET STRUT POSITION
021A'	CD 0282'	00279	CALL	SUB2 ;SUBTRACT STRUT FROM THRESH.
021D'	D1	00280	POP	D
021E'	E1	00281	POP	H
021F'	C9	00282	RET	
		00283	*****	
		00284	ROUTINE TO INPUT AND STORE DATA FROM MUX0,1 & 2	
		00285	*****	
0220'	3E 01	00286	IN3:	MVI A,01
0222'	21 F701	00287	LXI	H,OF701H
0225'	77	00288	MOV	M,A
0226'	2B	00289	DCX	H
0227'	36 01	00290	MVI	M,01
0229'	7E	00291	M1:	MOV A,M
022A'	07	00292	RLC	
022B'	D2 0229'	00293	JNC	M1
022E'	36 00	00294	MVI	M,00
0230'	2A F704	00295	LHLD	OF704H
0233'	22 8000	00296	SHLD	8000H
0236'	22 3F80	00297	SHLD	3F80H
0239'	3E 00	00298	MVI	A,00
023B'	21 F701	00299	LXI	H,OF701H
023E'	77	00300	MOV	M,A
023F'	2B	00301	DCX	H
0240'	36 01	00302	MVI	M,01
0242'	7E	00303	M2:	MOV A,M
0243'	07	00304	RLC	
0244'	D2 0242'	00305	JNC	M2
0247'	36 00	00306	MVI	M,00
0249'	2A F704	00307	LHLD	OF704H
024C'	22 3F84	00308	SHLD	3F84H
024F'	3E 02	00309	STP:	MVI A,02
0251'	21 F701	00310	IN1:	LXI H,OF701H
0254'	77	00311	MOV	M,A
0255'	2B	00312	DCX	H
0256'	36 01	00313	MVI	M,01
0258'	7E	00314	M3:	MOV A,M
0259'	07	00315	RLC	
025A'	D2 0258'	00316	JNC	M3
025D'	36 00	00317	MVI	M,00
025F'	2A F704	00318	LHLD	OF704H
0262'	22 3F86	00319	SHLD	3F86H
0265'	C9	00320	RET	
		00321	*****	
		00322	ROUTINE TO INPUT AND STORE DATA FROM MUX4	

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0266' 3E 04          00323 ;*****
0268' 21 F701        00324 IN4:   MVI    A,04
026B' 77             00325 LXI    H,OF701H
026C' 2B             00326 MOV    M,A
026D' 36 01          00327 DCX    H
026F' 7E             00328 MVI    M,01
0270' 07             00329 M4:   MOV    A,M
0271' D2 026F'       00330 RLC
0274' 36 00          00331 JNC    M4
0276' 2A F704        00332 MVI    M,00
0279' 22 3F96        00333 LHLD   OF704H
027C' C9             00334 SHLD   3F96H
00335 RET
00336 ;*****
00337 ; ROUTINE TO SUBTRACT BC FROM HL
00338 ;*****
027D' 7D             00339 FTEST: MOV    A,L
027E' 91             00340 SUB    C
027F' 7C             00341 MOV    A,H
0280' 98             00342 SBB    B
0281' C9             00343 RET
00344 ;*****
00345 ; ROUTINE FOR DOUBLE PRECISION SUBTRACT
00346 ; HL=DE-HL
00347 ;*****
0282' 7B             00348 SUB2:  MOV    A,E
0283' 95             00349 SUB    L
0284' 6F             00350 MOV    L,A
0285' 7A             00351 MOV    A,D
0286' 9C             00352 SBB    H
0287' 67             00353 MOV    H,A
0288' C9             00354 RET
00355 ;*****
00356 ; ROUTINE TO SHIFT VALUE IN HL LEFT 4 PLACES
00357 ;*****
0289' 29             00358 ANM:   DAD    H
028A' 29             00359 DAD    H
028B' 29             00360 DAD    H
028C' 29             00361 DAD    H
028D' C9             00362 RET
00363 ;*****
00364 ; ROUTINE TO ACTIVATE MATH BOARD & WAIT FOR RESULT
00365 ; ACCUMULATOR HAS OPCODE
00366 ;*****
028E' D3 A0           00367 MATH:  OUT    OAOH
0290' DB A7           00368 WAIT:  IN     OA7H

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0292' E6 01          00369      ANI      01
0294' C2 0290'       00370      JNZ      WAIT
0297' C9             00371      RET
00372      ;*****
00373      ;*****
00374      ;      THE FOLLOWING ARE SPECIAL CHECK-OUT ROUTINES
00375      ;      AND NOT PART OF THE MAIN PROGRAM
00376      ;*****
00377      ;*****
00378      ;
00379      ;
00380      ;*****
00381      ;      ROUTINE TO INPUT A VALUE FROM A/D & STORE IN RAM
00382      ;*****
0298' F3             00383      DI
0299' 3E 00           00384      MVI      A,00
029B' CD 0251'        00385      CALL     IN1
029E' CF              00386      RST      01H
029F' 00              00387      NOP
02A0' 00              00388      NOP
00389      ;*****
00390      ;      ROUTINE TO DO PGA TEST ON A/D
00391      ;*****
02A1' F3             00392      DI
02A2' 21 F701         00393      LXI      H,0F701H
02A5' 36 00           00394      PGA:    MVI      M,00
02A7' 36 C0           00395      MVI      M,OCOH
02A9' C3 02A5'        00396      JMP      PGA
02AC' 00              00397      NOP
00398      ;*****
00399      ;      ROUTINE TO OUTPUT A VALUE TO DAC0 & DAC1
00400      ;*****
02AD' F3             00401      DI
02AE' 00              00402      NOP
02AF' 21 0000         00403      R2:    LXI      H,0000
02B2' 22 F708         00404      SHLD    0F708H
02B5' 22 F70A         00405      SHLD    0F70AH
02B8' 00              00406      NOP
02B9' 00              00407      NOP
02BA' 00              00408      NOP
02BB' C3 02AF'        00409      JMP      R2
00410      END

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## MACROS:

## SYMBOLS:

ANM	0289'	CHACEL	00CF'	EXST	015C'	FLO	0176'
FLOA	01C7'	FTEST	027D'	IN1	0251'	IN3	0220'
IN4	0266'	L1	0042'	L10	00C2'	L11	00C5'
L12	016C'	L12A	01BD'	L13	01AD'	L13A	01FE'
L14	01B3'	L14A	0204'	L15	017F'	L15A	01DO'
L16	0191'	L16A	01E2'	L4	0120'	L5	0158'
L8	0069'	L8A	00A4'	L9	00B4'	M1	0229'
M2	0242'	M3	0258'	M4	026F'	MATH	028E'
PGA	02A5'	R2	02AF'	SPTH	020E'	START	0000'
STP	024F'	SUB2	0282'	TL1	001E'	TL2	0020'
TL3	0024'	WAIT	0290'				

NO FATAL ERROR(S)

ASSEMBLED AT 3D10

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00001 ;*****  
00002 ; PROGRAM FOR AN ELECTROHYDRAULIC ACTIVE  
00003 ; CONTROL AIRCRAFT LANDING GEAR  
00004 ;*****  
00005 ; NOTES:  
00006 ; RAM LOCATION 3DOEH ADDED AS A TEMP LOC  
00007 ; FOR FLIM 2/13/80  
00008 ;*****  
00009 ; REVISED FOR F4 GEAR 6/28/81  
00010 ORG 3D10H  
3D10' F3 00011 START: DI ;DISABLE INTERRUPTS  
3D11' 21 3FFF 00012 LXI H,3FFFH ;INIT. STACK  
3D14' F9 00013 SPHL  
3D15' 3E 82 00014 MVI A,82H ;INIT. MATH BOARD  
3D17' D3 EB 00015 OUT OEBH  
3D19' 3E 00 00016 MVI A,00 ;SET MEM. BASE ADD.  
3D1B' D3 A1 00017 OUT OA1H  
3D1D' 3E 80 00018 MVI A,80H  
3D1F' D3 A2 00019 OUT OA2H  
3D21' 21 0092 00020 LXI H,0092H ;STRUT THRESH.=1 IN.  
00021 ;MULT THRESH BY 16  
3D24' CD 3F99' 00022 CALL ANM ;FOR LATER USE  
3D27' 22 3F8A 00023 SHLD 3F8AH ;LABEL BXTHR  
3D2A' 3E 4C 00024 MVI A,4CH ;SET LIGHTS,SWITCHES  
3D2C' D3 EA 00025 OUT OEAH  
00026 ;*****  
00027 ; OUTPUT MUX0-MUX4 FOR A/D BOARD CHECK  
00028 ; ALSO LOOK FOR CONTROLLER ENABLED  
00029 ;*****  
3D2E' 16 03 00030 TL1: MVI D,3  
3D30' 06 FF 00031 TL2: MVI B,0FFH  
3D32' 0E 05 00032 MVI C,5  
3D34' 7A 00033 TL3: MOV A,D  
3D35' CD 3F61' 00034 CALL IN1  
3D38' 22 F708 00035 SHLD 0F708H ;OUTPUT TO DAC0  
3D3B' DB E9 00036 IN OE9H ;CONTROLLER ENABLED?  
3D3D' 1F 00037 RAR  
3D3E' DA 3D52' 00038 JC L1 ;YES, JUMP TO L1  
3D41' 05 00039 DCR B  
3D42' C2 3D34' 00040 JNZ TL3  
3D45' 06 FF 00041 MVI B,0FFH  
3D47' 0D 00042 DCR C  
3D48' C2 3D34' 00043 JNZ TL3  
3D4B' 15 00044 DCR D  
3D4C' FA 3D2E' 00045 JM TL1  
3D4F' C3 3D30' 00046 JMP TL2
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00047 ;*****
00048 ;CONTROLLER HAS BEEN ENABLED
00049 ;*****
3D52' 3E 02      00050 L1:    MVI   A,02
3D54'  CD 3F61'  00051     CALL  IN1    ;GET STRUT POS FOR LAND/TO. DEC.
3D57'  2A 3F8A   00052     LHLD  3F8AH   ;GET STRUT THRESHOLD
3D5A'  EB        00053     XCHG
3D5B'  2A 3F86   00054     LHLD  3F86H   ;LOAD HL WITH STRUT POSITION
3D5E'  CD 3F92'  00055     CALL  SUB2    ;CALC. THRESHOLD-STRUT
3D61'  DA 3ECD'  00056     JC    L12A    ;YES, JUMP TO 12A
00057 ;*****
00058 ;LANDING - MAKE PREPARATIONS
00059 ;*****
3D64'  3E 03      00060     MVI   A,03
3D66'  CD 3F61'  00061     CALL  IN1    ;GET SINK RATE
3D69'  22 3F88   00062     SHLD  3F88H   ;STORE IT
3D6C'  21 0915   00063     LXI   H,0915H ;MULT XMAX BY 16 TO SHIFT INTO
3D6F'  CD 3F99'  00064     CALL  ANM    ;UPPER 12 BITS
3D72'  22 3F8C   00065     SHLD  3F8CH   ;STORE IT
00066 ;*****
00067 ;ENABLE INTEGRATOR
00068 ;START ENERGY CALCULATIONS
00069 ;*****
3D75'  3E 9E      00070     MVI   A,9EH
3D77'  D3 EA      00071     OUT   0EAH    ;ENABLE INTEGRATOR
3D79'  CD 3F30'  00072 L8:    CALL  IN3
3D7C'  EB        00073     XCHG
3D7D'  2A 3F8C   00074     LHLD  3F8CH
3D80'  EB        00075     XCHG
3D81'  7B        00076     MOV   A,E
3D82'  95        00077     SUB   L
3D83'  6F        00078     MOV   L,A
3D84'  7A        00079     MOV   A,D
3D85'  9C        00080     SBB   H
3D86'  67        00081     MOV   H,A
3D87'  22 8004   00082     SHLD  8004H
3D8A'  AF        00083     XRA
3D8B'  CD 3F9E'  00084     CALL  MATH
3D8E'  2A 8000   00085     LHLD  8000H
3D91'  22 3F8E   00086     SHLD  3F8EH
3D94'  2A 8002   00087     LHLD  8002H
3D97'  22 3F90   00088     SHLD  3F90H
3D9A'  2A 3F88   00089     LHLD  3F88H
3D9D'  EB        00090     XCHG
3D9E'  2A 3F84   00091     LHLD  3F84H
3DA1'  CD 3F92'  00092     CALL  SUB2

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3DA4' 22 8000      00093      SHLD    8000H
3DA7' 22 8004      00094      SHLD    8004H
3DAA' AF           00095      XRA     A
3DAB' CD 3F9E'     00096      CALL    MATH
00097      ;*****
00098      ; DIVIDE KE BY 2 THEN DO A BYTE BY BYTE COMPARE
00099      ; TO TEST IF PE)KE. DON'T BOTHER TO TEST LSB-IT
00100      ; CONTAINS NO USEFUL DATA
00101      ;*****
3DAE' B7           00102      ORA     A      ;CLEAR CARRY
3DAF' 06 03         00103      MVI    B,3   ;SET A BYTE COUNTER
3DB1' 21 8003         00104      LXI    H,8003H ;GET KINETIC ENERGY
3DB4' 7E           00105      L8A:   MOV    A,M   ;SHIFT RIGHT 3 BYTES AND
3DB5' 1F           00106      RAR     ;RE-SAVE
3DB6' 77           00107      MOV    M,A
3DB7' 2B           00108      DCX     H
3DB8' 05           00109      DCR     B
3DB9' C2 3DB4'     00110      JNZ    L8A
3DBC' 06 03         00111      MVI    B,03
3DBE' 21 3F91         00112      LXI    H,3F91H
3DC1' 11 8003         00113      LXI    D,8003H
3DC4' 1A           00114      L9:   LDAX   D
3DC5' BE           00115      CMP     M
3DC6' C2 3DD2'     00116      JNZ    L10
3DC9' 1B           00117      DCX     D
3DCA' 2B           00118      DCX     H
3DCB' 05           00119      DCR     B
3DCC' C2 3DC4'     00120      JNZ    L9
3DCF' C3 3DD5'     00121      JMP    L11
3DD2' D2 3D79'     00122      L10:   JNC    L8
00123      ;*****
00124      ; TIME TO INITIATE ACTIVE CONTROL
00125      ;*****
3DD5' 2A 3F80         00126      L11:   LHLD   3F80H
3DD8' 22 F70A         00127      SHLD   OF70AH
3DDB' 22 3DOE         00128      SHLD   3DOEH   ;SAVE ORIGINAL "FLIM"
3DDE' EB           00129      XCHG
3DDF' CD 3F30'         00130      CHACEL: CALL   IN3   ;GET W/G ACCEL.
3DE2' 2A 3F80         00131      LHLD   3F80H   ;PUT NEW ACCEL INTO HL
3DE5' CD 3F92'         00132      CALL   SUB2   ;DE-HL IS NEW W/G ACCEL.
00133      ;GREATER THAN FLIM?
3DE8' F2 3DDF'         00134      JP    CHACEL ;NO-LOOP TILL IT IS
(3DEB-3DFO) = 00 (NOP) 00135      ;CALL  SPTH   ;HAS GEAR STARTED STROKE?
00136      ;JP    CHACEL ;LOOP TILL GEAR ) THRESHOLD
3DF1' 3E 9F           00137      MVI    A,9FH
3DF3' D3 EA           00138      OUT    OEAH   ;ENABLE SERVOLOOP

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00139 ;*****
00140 ; GEAR IS NOW UNDER ACTIVE CONTROL
00141 ;*****
3DF5' 2A 3DOE 00142 LHLD 3DOEH ;GET ORIGINAL FLIM TO CALCULATE
3DF8' 22 8000 00143 SHLD 8000H ;TRANSITION VELOCITY
3DFB' 21 0000 00144 LXI H,0000
3DFE' 22 8002 00145 SHLD 8002H
3E01' 3E 08 00146 MVI A,08
3E03' CD 3F9E' 00147 CALL MATH
3E06' 3E 06 00148 MVI A,06
3E08' CD 3F9E' 00149 CALL MATH
3E0B' 3E 36 00150 MVI A,36H
3E0D' 32 8004 00151 STA 8004H
3E10' 3E B5 00152 MVI A,OB5H
3E12' 32 8005 00153 STA 8005H
3E15' 3E CD 00154 MVI A,OCDH
3E17' 32 8006 00155 STA 8006H
3E1A' 3E 56 00156 MVI A,56H
3E1C' 32 8007 00157 STA 8007H
3E1F' 3E 02 00158 MVI A,02
3E21' CD 3F9E' 00159 CALL MATH
3E24' 2A 8000 00160 LHLD 8000H
3E27' 22 3F92 00161 SHLD 3F92H
3E2A' 2A 8002 00162 LHLD 8002H
3E2D' 22 3F94 00163 SHLD 3F94H
00164 ;*****
00165 ; TRANSITION VELOCITY STORED AS FLOATING PT,32 BIT
00166 ; NUMBER. START COMPARING THIS AGAINST (SINK RATE
00167 ; -W/G VEL.) TO DETERMINE START OF TRANSITION
00168 ;*****
3E30' 2A 3F88 00169 L4: LHLD 3F88H
3E33' EB 00170 XCHG
3E34' 3E 00 00171 MVI A,00
3E36' CD 3F61' 00172 CALL IN1
3E39' CD 3F92' 00173 CALL SUB2
3E3C' 22 8000 00174 SHLD 8000H
3E3F' 21 0000 00175 LXI H,0000
3E42' 22 8002 00176 SHLD 8002H
3E45' 3E 08 00177 MVI A,08
3E47' CD 3F9E' 00178 CALL MATH
3E4A' 2A 3F92 00179 LHLD 3F92H
3E4D' 22 8004 00180 SHLD 8004H
3E50' 2A 3F94 00181 LHLD 3F94H
3E53' 22 8006 00182 SHLD 8006H
3E56' 3E 0A 00183 MVI A,0AH
3E58' CD 3F9E' 00184 CALL MATH

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3E5B'	DB A1	00185	IN	0A1H
3E5D'	E6 20	00186	ANI	20H
3E5F'	CA 3E30'	00187	JZ	L4
00188 ;*****				
00189 ;TRANSITION PHASE				
00190 ;*****				
3E62'	2A 3DOE	00191	LHLD	3DOEH ;FLIM
3E65'	11 FFFE	00192	LXI	D,OFFFEH
3E68'	22 F70A	00193	L5: SHLD	OF70AH
3E6B'	19	00194	DAD	D
3E6C'	CD 3F1E'	00195	EXST:	CALL SPTH ;CHECK (THRESH.-STRUT POS'N)
3E6F'	F2 3E7C'	00196	JP	L12
3E72'	01 1BDO	00197	LXI	B,1BDOH
3E75'	7D	00198	MOV	A,L
3E76'	91	00199	SUB	C
3E77'	7C	00200	MOV	A,H
3E78'	98	00201	SBB	B
3E79'	D2 3E68'	00202	JNC	L5
00203 ;*****				
00204 ;ROLLOUT PHASE				
00205 ;*****				
3E7C'	21 0000	00206	L12: LXI	H,0000
3E7F'	22 F70A	00207	SHLD	OF70AH ;SET FLC=0 LBS.
3E82'	3E 99	00208	MVI	A,99H
3E84'	D3 EA	00209	OUT	OEAH ;OPEN FORCE FEEDBACK
3E86'	CD 3F1E'	00210	FLO:	CALL SPTH ;CHECK STRUT POS'N.
3E89'	F2 3E86'	00211	JP	FLO
3E8C'	01 1EE0	00212	LXI	B,1EEOH ;SET BC TO 2000 LBS.
3E8F'	CD 3F30'	00213	L15: CALL	IN3
3E92'	2A 8000	00214	LHLD	8000H
3E95'	22 F708	00215	SHLD	OF708H ;OUTPUT W/G ACCEL. TO DACO
3E98'	CD 3F8D'	00216	CALL	FTEST
3E9B'	D2 3EBD'	00217	JNC	L13
3E9E'	01 1EE0	00218	LXI	B,1EEOH ;SET BC TO 2000 LBS.
3EA1'	CD 3F76'	00219	L16: CALL	IN4
3EA4'	22 F708	00220	SHLD	OF708H ;OUTPUT W/G ACCEL TO DACO
3EA7'	CD 3F8D'	00221	CALL	FTEST
3EAA'	DA 3E7C'	00222	JC	L12
3EAD'	21 1EE0	00223	LXI	H,1EEOH ;SET HL TO 2000 LBS.
3EB0'	3E 95	00224	MVI	A,95H ;CLOSE F FDBK & REV. SIGN
3EB2'	D3 EA	00225	OUT	OEAH ;OF LIMIT FORCE CMD (ANALOG)
3EB4'	22 F70A	00226	SHLD	OF70AH ;SET LIMIT FORCE CMD=2000 LBS
3EB7'	01 1800	00227	LXI	B,1800H ;SET BC TO 1600 LBS
3EBA'	C3 3EA1'	00228	JMP	L16
3EBD'	21 1EE0	00229	L13: LXI	H,1EEOH ;SET HL TO 2000 LBS
3EC0'	22 F70A	00230	SHLD	OF70AH ;SET LIMIT FORCE CMD=2000 LBS

3EC3'	3E 9D	00231	L14:	MVI	A,9DH	
3EC5'	D3 EA	00232	OUT	OEAH	; CLOSE FORCE FEEDBACK	
3EC7'	01 1800	00233	LXI	B,1800H	; SET BC TO 1600 LBS	
3ECA'	C3 3E8F'	00234	JMP	L15		
		00235	*****			
		00236	; TAKEOFF MODE			
		00237	*****			
3ECD'	21 0000	00238	L12A:	LXI	H,0000	
3ED0'	22 F70A	00239	SHLD	OF70AH	; SET LIMIT FORCE CMD= 0 LBS.	
3ED3'	3E A9	00240	MVI	A,0A9H		
3ED5'	D3 EA	00241	OUT	OEAH	; ENABLE SERVOLOOP & OPEN F FDBK	
3ED7'	CD 3F1E'	00242	FLOA:	CALL	SPTH ;CHECK STRUT POSITION	
3EDA'	F2 3ED7'	00243		JP	FLOA	
3EDD'	01 1EE0	00244		LXI	B,1EE0H ;SET BC TO 2000 LBS	
3EE0'	CD 3F30'	00245	L15A:	CALL	IN3	
3EE3'	2A 8000	00246		LHLD	8000H	
3EE6'	22 F708	00247		SHLD	OF708H ;OUTPUT W/G ACCEL TO DACO	
3EE9'	CD 3F8D'	00248		CALL	FTEST	
3ECC'	D2 3FOE'	00249		JNC	L13A	
3EEF'	01 1EE0	00250		LXI	B,1EE0H ;SET BC TO 2000 LBS	
3EF2'	CD 3F76'	00251	L16A:	CALL	IN4	
3EF5'	22 F708	00252		SHLD	OF708H ;OUTPUT W/G ACCEL TO DACO	
3EF8'	CD 3F8D'	00253		CALL	FTEST	
3EFB'	DA 3ECD'	00254		JC	L12A	
3EFE'	21 1EE0	00255		LXI	H,1EE0H ;SET HL TO 2000 LBS	
3FO1'	3E A5	00256		MVI	A,0A5H ;CLOSE F FDBK & REVERSE SIGN	
3FO3'	D3 EA	00257		OUT	OEAH ;OF LIMIT FORCE CMD (ANALOG)	
3FO5'	22 F70A	00258		SHLD	OF70AH ;SET LIMIT FORCE CMD=2000 LBS	
3FO8'	01 1800	00259		LXI	B,1800H ;SET BC TO 1600 LBS	
3FOB'	C3 3EF2'	00260		JMP	L16A	
3FOE'	21 1EE0	00261	L13A:	LXI	H,1EE0H ;SET HL TO 2000 LBS	
3F11'	22 F70A	00262		SHLD	OF70AH ;SET LIMIT FORCE CMD=2000 LBS	
3F14'	3E AD	00263	L14A:	MVI	A,0ADH	
3F16'	D3 EA	00264		OUT	OEAH ;CLOSE FORCE FEEDBACK	
3F18'	01 1800	00265		LXI	B,1800H ;SET BC TO 1600 LBS	
3F1B'	C3 3EE0'	00266		JMP	L15A	
		00267	*****			
		00268	; ROUTINE TO SUBTRACT STRUT POS'N FROM THRESHOLD			
		00269	*****			
3F1E'	E5	00270	SPTH:	PUSH	H	;SPTH SETS SIGN FLAG POSITIVE
3F1F'	D5	00271		PUSH	D	;UNTIL STRUT POS'N ) THRESHOLD
3F20'	CD 3F5F'	00272		CALL	STP	;GET STRUT POSITION
3F23'	21 OB20	00273		LXI	H,OB20H ;THRESHOLD 0160H=.05"	
		00274				02DOH=.1"
		00275				059OH=.2"
		00276				OB20H=.4"

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3F26' EB 00277 XCHG
3F27' 2A 3F86 00278 LHLD 3F86H ; GET STRUT POSITION
3F2A' CD 3F92' 00279 CALL SUB2 ;SUBTRACT STRUT FROM THRESH.
3F2D' D1 00280 POP D
3F2E' E1 00281 POP H
3F2F' C9 00282 RET
00283 ****
00284 ; ROUTINE TO INPUT AND STORE DATA FROM MUXO, 1 & 2
00285 ****
3F30' 3E 01 00286 IN3: MVI A,01
3F32' 21 F701 00287 LXI H,OF701H
3F35' 77 00288 MOV M,A
3F36' 2B 00289 DCX H
3F37' 36 01 00290 MVI M,01
3F39' 7E 00291 M1: MOV A,M
3F3A' 07 00292 RLC
3F3B' D2 3F39' 00293 JNC M1
3F3E' 36 00 00294 MVI M,00
3F40' 2A F704 00295 LHLD OF704H
3F43' 22 8000 00296 SHLD 8000H
3F46' 22 3F80 00297 SHLD 3F80H
3F49' 3E 00 00298 MVI A,00
3F4B' 21 F701 00299 LXI H,OF701H
3F4E' 77 00300 MOV M,A
3F4F' 2B 00301 DCX H
3F50' 36 01 00302 MVI M,01
3F52' 7E 00303 M2: MOV A,M
3F53' 07 00304 RLC
3F54' D2 3F52' 00305 JNC M2
3F57' 36 00 00306 MVI M,00
3F59' 2A F704 00307 LHLD OF704H
3F5C' 22 3F84 00308 SHLD 3F84H
3F5F' 3E 02 00309 STP: MVI A,02
3F61' 21 F701 00310 IN1: LXI H,OF701H
3F64' 77 00311 MOV M,A
3F65' 2B 00312 DCX H
3F66' 36 01 00313 MVI M,01
3F68' 7E 00314 M3: MOV A,M
3F69' 07 00315 RLC
3F6A' D2 3F68' 00316 JNC M3
3F6D' 36 00 00317 MVI M,00
3F6F' 2A F704 00318 LHLD OF704H
3F72' 22 3F86 00319 SHLD 3F86H
3F75' C9 00320 RET
00321 ****
00322 ; ROUTINE TO INPUT AND STORE DATA FROM MUX4

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00323 ;*****
00324 IN4: MVI A,04
00325 LXI H,0F701H
00326 MOV M,A
00327 DCX H
00328 MVI M,01
00329 M4: MOV A,M
00330 RLC
00331 JNC M4
00332 MVI M,00
00333 LHLD 0F704H
00334 SHLD 3F96H
00335 RET
00336 ;*****
00337 ; ROUTINE TO SUBTRACT BC FROM HL
00338 ;*****
00339 FTEST: MOV A,L
00340 SUB C
00341 MOV A,H
00342 SBB B
00343 RET
00344 ;*****
00345 ; ROUTINE FOR DOUBLE PRECISION SUBTRACT
00346 ; HL=DE-HL
00347 ;*****
00348 SUB2: MOV A,E
00349 SUB L
00350 MOV L,A
00351 MOV A,D
00352 SBB H
00353 MOV H,A
00354 RET
00355 ;*****
00356 ; ROUTINE TO SHIFT VALUE IN HL LEFT 4 PLACES
00357 ;*****
00358 ANM: DAD H
00359 DAD H
00360 DAD H
00361 DAD H
00362 RET
00363 ;*****
00364 ; ROUTINE TO ACTIVATE MATH BOARD & WAIT FOR RESULT
00365 ; ACCUMULATOR HAS OPCODE
00366 ;*****
00367 MATH: OUT 0AOH
00368 WAIT: IN 0A7H

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3FA2' E6 01          00369      AN1      01
3FA4' C2 3FA0'        00370      JNZ      WAIT
3FA7' C9             00371      RET
00372      ;*****
00373      ;*****
00374      ;***** THE FOLLOWING ARE SPECIAL CHECK-OUT ROUTINES
00375      ;***** AND NOT PART OF THE MAIN PROGRAM
00376      ;*****
00377      ;*****
00378      ;
00379      ;
00380      ;*****
00381      ;***** ROUTINE TO INPUT A VALUE FROM A/D & STORE IN RAM
00382      ;*****
3FA8' F3             00383      DI
3FA9' 3E 00           00384      MVI     A,00
3FAB' CD 3F61'        00385      CALL    IN1
3FAE' CF              00386      RST     01H
3FAF' 00              00387      NOP
3FB0' 00              00388      NOP
00389      ;*****
00390      ;***** ROUTINE TO DO PGA TEST ON A/D
00391      ;*****
00392      DI
00393      LXI     H,0F701H
00394      PGA:   MVI     M,00
00395      MVI     M,OCOH
00396      JMP     PGA
00397      NOP
00398      ;*****
00399      ;***** ROUTINE TO OUTPUT A VALUE TO DAC0 & DAC1
00400      ;*****
3FBD' F3             00401      DI
3FBE' 00              00402      NOP
3FBF' 21 0000          00403      R2:    LXI     H,0000
3FC2' 22 F708          00404      SHLD   0F708H
3FC5' 22 F70A          00405      SHLD   0F70AH
3FC8' 00              00406      NOP
3FC9' 00              00407      NOP
3FC A' 00              00408      NOP
3FCB' C3 3FBF'         00409      JMP     R2
00410      END

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## MACROS:

## SYMBOLS:

ANM	3F99'	CHACEL	3DDF'	EXST	3E6C'	FLO	3E86'
FLOA	3ED7'	FTEST	3F8D'	IN1	3F61'	PN3	3F30'
IN4	3F76'	L1	3D52'	L10	3DD2'	L11	3DD5'
L12	3E7C'	L12A	3ECD'	L13	3EBD'	L13A	3FOE'
L14	3EC3'	L14A	3F14'	L15	3E8F'	L15A	3EE0'
L16	3EA1'	L16A	3EF2'	L4	3E30'	L5	3E68'
L8	3D79'	L8A	3DB4'	L9	3DC4'	M1	3F39'
M2	3F52'	M3	3F68'	M4	3F7F'	MATH	3F9E'
PGA	3FB5'	R2	3FBF'	SPTH	3F1E'	START	3D10'
STP	3F5F'	SUB2	3F92'	TL1	3D2E'	TL2	3D30'
TL3	3D34'	WAIT	3FA0'				

NO FATAL ERROR(S)

8.0 REFERENCE

1. Ross, Irving and Edson, Ralph: An Electronic Control for an Electrohydraulic Active Control Aircraft Landing Gear. NASA Contractor Report 3113, April, 1979.



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7. Author(s)  Irving Ross and Ralph Edson		6. Performing Organization Code	
9. Performing Organization Name and Address  Hydraulic Research Textron, Inc. Valencia, CA 91355		8. Performing Organization Report No. 12203 HR 74600000	
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16. Abstract  <p>HR Textron Inc., under NASA Contract NAS1-14459, has developed and designed a controller for an electro-hydraulic active control landing gear for the F-4 aircraft. A controller, developed under NASA Contract NAS1-14459, was modified for this application. Simulation results indicate that during landing and rollout over repaired bomb craters the active gear effects a force reduction, relative to the passive gear, of approximately 70%.</p>		13. Type of Report and Period Covered Contractor Report	
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